# MODELING OF WIND INDUCED FATIGUE OF THINWALLED FOLDED PLATES 

Osvaldo Rosario Galanes

A thesis submitted in partial fulfillment of the requirements for the degree of

MASTER OF SCIENCE
in
CIVIL ENGINEERING

UNIVERSITY OF PUERTO RICO
MAYAGÜEZ CAMPUS
2011
© Osvaldo Rosario 2011
Approved by:

Luis A. Godoy, PhD
Date
President, Graduate Committee

Ricardo R. López, PhD
Date
Member, Graduate Committee

Ali Saffar, PhD
Date
Member, Graduate Committee

Néstor Pérez, PhD
Date
Representative of Graduate Studies

Date


#### Abstract

During high wind events thin-walled folded plates such as roof claddings suffer extensive damage due to fatigue failure at their screwed connections. Currently, damage prediction is based on fragility curves and semi-empirical models where there is no direct relation between loads and failure parameters. In this research a finite element model and fatigue criteria were validated in correlating a wind loading protocol to a specific fatigue failure mechanism. The 3D fatigue process was reduced to a one-dimensional problem by using one or two strain and stress parameters from the finite element model. These parameters were used on a fatigue stress-based and strain-based criteria to establish a fatigue failure mechanism. The fatigue life, and crack directions were predicted as a function of loading parameters and specific cladding configuration. As a result a roofing assembly can be evaluated and rated to withstand a specific wind resistance according to a specific loading protocol.


## Resumen

Los techos y cubiertas de placas de pared delgada de lámina metálica sufren daños considerables debido a fallas por fatiga en las conexiones atornilladas durante eventos de viento. Actualmente la predicción de daño está basada en curvas de fragilidad y modelos semi empíricos en donde no existe una relación directa entre carga y parámetros de falla. En esta investigación se validó un modelo de elementos finitos y un criterio de fatiga que correlacionan un protocolo de cargas de viento a un mecanismo de falla por fatiga. El proceso tridimensional se redujo a un problema unidimensional utilizando uno o dos parámetros de esfuerzo y deformación del modelo de elementos finitos. Estos parámetros se usaron en un criterio de fatiga basado en esfuerzos y deformaciones para establecer un mecanismo de daño por fatiga. La vida de fatiga y las direcciones de propagación de las grietas se predijeron en función de los parámetros de carga y la configuración especifica estudiada. Como resultado una cubierta de techo puede ser evaluada y calificada para resistir una carga de viento en específico de acuerdo a un protocolo de carga.

## Acknowledgments

In first place I want to thank my advisor, Dr. Luis A. Godoy, for inspiring me to see beyond the purely academic limits of the classroom into the art of visualizing reality through the mathematical equations that give a perfect physical meaning to our world. I also want to thank Dr. Ricardo López and Dr. Ali Saffar for guiding me throughout the process and getting me closer to achieving my goal. Dr. Marcelo Suárez also recommended important references for this research. Finally I am very grateful to my family and special people that supported me throughout this investigation. Ultimately I am grateful to God for all of my achievements and all of the great minds that motivated me to keep thinking.

## Table of Contents

Chapter 1: Introduction
1.1 Motivation ..... 1
1.2 Significance of the Present Study ..... 2
1.3 Scope of the Research ..... 3
1.4 Objectives ..... 4
Chapter 2: Literature Review
2.1 Previous Publications ..... 5
Chapter 3: Methodology
3.1 Introduction ..... 12
3.2 Overall Methodology ..... 13
3.3 Finite Element Mesh and Type of Elements ..... 14
3.4 Static Model ..... 18
3.5 Modeling Screwed Connections ..... 26
Chapter 4: Results of Static Analysis
4.1 Introduction ..... 36
4.2 Linear and Elastic Results ..... 36
4.3 Geometricaly Non-linear and Elastic analysis ..... 43
4.4 Geometricaly Non-linear and Elastic-plastic analysis ..... 46
4.5 Geometrically Nonlinear Results Including Ideal Plasticity ..... 47
4.6 Typical Center Connection ..... 51
4.7 Discussion of Results ..... 53
Chapter 5: Results of Fatigue Analysis
5.1 Fatigue Analysis Methodology ..... 55
5.2 Fatigue Analysis Results ..... 65
5.3 Discussion of Results and Parametric Studies ..... 67
5.4 Development of a Suitable Parameter Estimation Method ..... 71
5.5 Fatigue Failure ..... 74
5.6 Proposed Method for the Fatigue Life Prediction of Steel Thin- Walled Folded Plates Using an Analytical and Computational
Approach ..... 75
Chapter 6: Conclusions
6.1 Summary of Contributions ..... 80
6.2. Main Conclusions Derived from this Research ..... 80
6.3. Original Contributions of this Thesis ..... 81
6.4. Recommendations for Further Research ..... 82
References ..... 83

## List of Tables

Table 1 Configuration definition ..... 22
Table 2 Error estimate ..... 32
Table 3 Displacement order of error ..... 33
Table 4 Strain order of error ..... 34
Table 5 Percent error ..... 34
Table 6 Fatigue damage average and standard deviation for the 12 configurations ..... 70

## List of Figures

Figure 1 Wind pressure simulation by means of air bags in the experiments ..... 12
Figure 2 Eight node shell element (S8R) and local coordinate system adopted ..... 15
Figure 3a Geometry model ..... 18
Figure 3b Geometry definition ..... 19
Figure 3c Type B (WR) cladding with screwed connections at valleys ..... 20
Figure 3d Type B (WR) cladding with screwed connections at crests ..... 21
Figure 4a Detail of a center connection from shaded region in Figure 3c ..... 23
Figure 4b Direction of moments and forces acting on the section ..... 23
Figure 5 Strain hardening material definition ..... 24
Figure 6a Complete mesh for $1 / 4$ of the cladding shown in Figure 3a ..... 27
Figure 6b Mesh transition from a typical center screwed connection ..... 28
Figure 7 Displacement convergence ..... 30
Figure 8 Strain convergence ..... 31
Figure 9 Linear results at boundary 1 in Figure 4 ..... 37
Figure 10 Linear results, boundary 3 in Figure 4 ..... 38
Figure 11 Linear results, membrane stress for load $P$ ..... 40
Figure 12 Linear results, moment for load $P$ ..... 41
Figure 13 Linear results, shear stress for load $P$ ..... 42
Figure 14 Geometrically non-linear and elastic results at boundary 1 in Figure 4 ..... 44
Figure 15 Geometrically non-linear and elastic results at boundary 3 in Figure 4 ..... 45
Figure 16 Non-linear results at boundary 1 ..... 48
Figure 17 Non-linear results along boundary 3 ..... 49
Figure 18 Plastic zone for load $P=5.81 \mathrm{kPa}$ evaluated at the bottom surface ..... 50
Figure 19 Plastic zone for load $0.25 P$ at the bottom surface ..... 50
Figure 20a Strain concentration (major principal) at center connection, boundary 1 ..... 52
Figure 20b Strain concentration (components) at center connection, boundary 1 ..... 52
Figure 21 Octahedral shear strain at center connection, boundary 1 ..... 53
Figure 22 Completely reversed loading cycle $(\mathrm{R}=-1)$ ..... 55
Figure 23 Plastic deformation and hysteretic loop for tension loading $(\mathrm{R}=0)$ ..... 57
Figure 24 Load sequence used in Sidgers protocol (Baskaran et al. 2006) ..... 62
Figure 25 Cyclic stress-strain diagram ..... 63
Figure 26 Comparison of Fatigue Models for the universal slopes method ..... 67
Figure 27 Comparison of fatigue models for the elastic-plastic material definition ..... 68
Figure 28 Comparison of fatigue models for the strain hardening material definition ..... 69
Figure 29 Comparison of fatigue models for the elastic-plastic material definition ..... 70
Figure 30 Comparison of connection types and fatigue damage ..... 71
Figure 31 Comparison of connection types and fatigue damage for the proposed method ..... 73
Figure 32 Failure of Screwed connections by García, 2008, pp. 65 ..... 74

## List of Appendices

A. Specifications for Steel Cladding Geometry and Configuration ..... 86
B. Finite Element Analysis Output Data for the Strain Hardening and
Elastic-Plastic Material Definitions (Abaqus 2008) ..... 87
C. Fatigue Damage Results, Elastic-Plastic ..... 99
D. Fatigue Damage Results, Strain Hardening ..... 118
E. Fatigue Damage Results for the Proposed Parameter Estimation Method .. ..... 137

## List of Symbols

[H] strain-curvature compatibility matrix
[J] Jacobian matrix
[ $k$ ] element stiffness matrix
[ $\mu$ ] matrix of directional cosines
$b$ fatigue strength exponent
$B$ spatial derivative of field variable
$c$ fatigue ductility exponent
$d$ distance from edge of screw hole
$E$ modulus of elasticity
$e$ percentage of error
$F$ fatigue damage
$h$ element size
$N$ shape function
$N_{f}$ number of cycles to failure
$N_{M}$ number of cycles to failure as defined by Morrow
$N_{G}$ number of cycles to failure as defined by Goodman
$N_{S W T}$ number of cycles to failure as defined by Smith, Watson and Topper
$n$ number of cycles
$O$ order of error
$P$ wind load
$P_{\text {max }}$ previously applied maximum load
$P_{\text {min }}$ minimum sublevel load
$P_{\text {submax }}$ maximum sublevel load
$R$ ratio of minimum stress to maximum stress
$s$ sample standard deviation
$u, v, w$ displacements
$\alpha, \beta$ nodal rotational degrees of freedom
$\gamma$ shear strain
$\varepsilon$ normal strain
$\varepsilon_{a}$ strain amplitude
$\varepsilon_{\text {max }}$ maximum principal strain
$\varepsilon_{y}$ yield strain
$\varepsilon_{f}^{\prime}$ fatigue ductility coefficient
$\Delta \varepsilon_{e q}$ equivalent strain range
$\Delta \varepsilon_{\text {max }}$ maximum principal strain range
$\mu$ sample mean
$\sigma_{a}$ stress amplitude
$\sigma_{m}$ mean stress
$\sigma_{\text {max }}$ maximum stress
$\sigma_{\max p}$ maximum principal stress
$\sigma_{\text {max prev }}$ maximum previously applied maximum stress
$\sigma_{u}$ ultimate tensile strength
$\sigma_{f}^{\prime}$ fatigue strength coefficient
$v$ Poisson's ratio
$\phi$ field quantity
$\xi, \eta, \zeta$ element local coordinates

## Chapter 1: Introduction

### 1.1 Motivation

The need to understand wind damage induced failure on roof cladding systems has come a long way since the advent of new materials and building practices. The recent development of more durable high resistance thin walled structural components imposes a challenge in the design of structures for high winds. It is necessary that thin walled structures are designed to withstand the impact that such events can cause as huge economical losses have been reported throughout the industry. Knowledge of how these structural components fail during high wind events will provide new insights on how to design more efficiently and as a result minimize damage and economical losses as well as provide a sense of security.

There are several reasons that support further investigation on the behavior of these thin walled structural components during damaging wind storms. The first aspect is the difficulty in determining load fluctuations in time associated with these events. Although this research is not centered in determining loading variations in history, the nature of the load can significantly contribute to the complexity of the problem. Other challenges can arise if non-linear effects are considered on the static model. Once stress and strain components are determined from the static model, a new challenge arises in establishing successful fatigue criteria. There are further complications if one considers the phenomenon of three-dimensional fatigue and the wide margin of error attributable in making accurate predictions on the fatigue life. All of these factors introduce significant difficulties that establish a need to better understand the process associated with fatigue failure of thin walled structural components.

Although recent investigations have attempted to correlate static loading on steel claddings to wind induced failure, there is still a need to find a successful fatigue criterion that takes into account load fluctuations and cladding resistance to fatigue damage, based on analysis and understanding of the principles of stress and strain prediction as well as fatigue life. Knowledge of fatigue fracture parameters of different kinds of steels is now available in order to predict fatigue failure based on previously established criteria. This knowledge can be used in conjunction with finite element modeling and analysis of the stresses caused by the applied pressures and forces in order to predict capacity and hence failure.

### 1.2 Significance of the Present Study

Thin-walled roof claddings and structural components suffer the most when exposed to high winds due to their low weight to area ratio. The difference in pressure caused by wind can impose significant reaction loads on their screwed connections, causing high stress concentrations around them. Since industrial structures made out of these components are generally of low cost as compared to other building types, they often present deficiencies in design, which then imposes an imminent threat during wind events that occur mainly on tropical regions. Flying pieces of roof cladding can pose a threat to nearby property, as has been the case in numerous storms and hurricanes in the past.

As new insight is gained from the fatigue behavior of these structural components due to wind, previously established fatigue criteria can be used to predict and prevent failure of these systems. The present research attempts to correlate fatigue criteria to wind
induced fatigue damage of thin walled steel components by linking current knowledge about fatigue failure and its application to thin walled steel claddings.

### 1.3 Scope of the Research

This investigation is limited to the behavior of thin-walled folded plates under fluctuating lateral loads. A specific geometric configuration of trapezoidal steel claddings is considered in this thesis, in which stress concentrations are considered in a localized zone. Other geometric configurations are not considered as the fatigue process is assumed to take place within a small, localized region. The research is only based on computational modeling and not on full-scale experimental studies, but results obtained have been compared to experimental results obtained by previous investigators in order to validate the analytical model. The present modeling employs a shell finite element in which elastic as well as plastic deformations are considered. From the finite element results, the analysis of multiaxial constitutive parameters aid in reducing the multiaxial state of stress and strain from a three dimensional fatigue problem into a one or two dimensional problem. Evaluation of fatigue criteria focuses on low-cycle fatigue and on criteria were fatigue parameters are already established for different kinds of steels. Out of all the possible loading configurations attributable to wind, a specific loading protocol is used to establish damage parameters caused by fatigue failure.

### 1.4 Objectives

The primary goal of this research is to provide a plausible explanation for describing the failure mechanism of thin-walled folded plates caused by fatigue damage associated to wind fluctuations. In order to achieve this goal, several specific objectives are proposed in this research:

1. To develop a two-scale model to evaluate stress fields in the local area of the connections in a folded plate under static lateral pressures.
2. To identify appropriate existing fatigue models to represent 3-D fatigue processes in thin-walled components.
3. To implement existing loading protocols used in fatigue testing into fatigue analysis.
4. To provide explanations of fatigue mechanisms in folded plates based on computer modeling and parametric studies.

## Chapter 2: Literature Review

### 2.1 Previous Publications

It is common to observe wind induced damage of roof and wall steel cladding systems in the tropics and other high wind prone regions. The Canadian Standards Association (2004) developed a standard test method for the dynamic wind uplift resistance of mechanically attached membrane-roofing systems. The test method employs a wind loading protocol based on design wind pressures in accordance with local building codes. The dynamic protocol has five rating levels (identified as A to E) to evaluate a roof assembly for a specific wind resistance. Each level consists of eight load sequences with different pressure ranges representing different wind conditions (Baskaran et al. 2006). The structure is then rated based on how many levels it is able to withstand before failing. Several investigators have implemented this sequence in the full-scale testing of cladding systems (Avilés 2006; García 2007; García 2008). Avilés (2006) proposed to modify the original Canadian loading protocol by multiplying the total number of cycles by five. The new SIDGERS-5 loading protocol would allow any given roofing assembly to be rated in terms of the original rating system but being able to withstand five times the damage.

Research has been made in order to predict fatigue failure damage in steel cladding systems using empirical methods and fragility curves (Lee and Rosowsky 2004; García 2007; García 2008). García (2007) performed a series of tests on wood-zinc components in order to obtain the performance of a wood-zinc structure with a specific configuration. In his research, he sampled a variety of common configurations throughout

Puerto Rico and evaluated each configuration in terms of wind speed performance. He then calculated the roof performance index defined as the ratio of resistance to capacity based on the damage of individual components. Fragility curves are then obtained by plotting the probability of exceeding a given damage state versus wind speed. Based on the studies performed by García (2007), a specific structure can be rated to withstand a specific level of performance based on how well it is constructed. A contribution of this research is evaluating the performance of wood-zinc structures based on wind speed and construction practices, and to prove that these kinds of structures are able to withstand hurricane force winds if properly constructed. García (2008) based his research on what is called Component Based Fragility. In this approach, the individual capacities of each connection on a structure are found and compared to individual demands. According to a specific wind speed, the percent of capacities that are not able to withstand the demands are then found and a structure is then rated based on a specific level of damage. The probability of exceeding a given level of damage is then plotted versus wind speed and fragility curves are obtained. The main difference from García (2007) is that García (2008) studied construction practices used mainly on industrial buildings. In his research he performed full-scale tests of structural components, which are most vulnerable on these kinds of buildings, namely roof and wall steel claddings. Among empirical methods, Avilés (2006) performed similar tests on steel claddings and plotted the load versus number of cycles on a logarithmic scale in order to obtain a relationship for the fatigue life equation.

Recent investigations in Australia have adopted analytical approaches using largescale experimental models and finite element analysis models of cladding systems. Such
investigations were successful in determining a strain criterion and design formula for static pull-through failures in crest-fixed steel claddings (Mahaarachchi and Mahendran 2008, 2009).

Although there is a wealth of information about wind induced failure of steel cladding systems, research is still needed to account for building practices in the US and Puerto Rico. Almost all analytical models dealing with steel cladding systems are aimed at Australian construction practices. The differences between Australian, European, and American construction practices has been highlighted as follows: "European and American recommendations for steel claddings cannot be used as compared with Australian steel claddings, they are made of thicker, deeper and softer steel cladding fastened at valleys while Australian steel claddings are commonly made of thin, high strength steel G550 and are crest-fixed" (Mahaarachchi and Mahendran 2009). A new approach is needed that considers valley-fixed steel claddings and fatigue loading in developing a working analytical model.

The process of fatigue failure of thin-wall folded plates is governed by several factors discussed earlier. Before treating the process of fatigue failure of thin-wall folded plates, which is mostly considered based on observations and for which an analytical basis is not fairly grounded in the literature, it is first necessary to understand the behavior under static load. The static load case is well documented. Observations and analytical models have described static failure. The static analysis of an industrial building should consider the structural component that is most vulnerable to fatigue damage. These components are the thin metal roof claddings (Morgan and Beck 1977). Other studies have shown that among these components, stress concentrations around the
connections are responsible for low cycle fatigue failure (Beck and Stevens 1979), (Cook 1990), and Mahendran (1990a, b, 1995). An investigation of the static load case was made when "An inspection of steel roofs made of trapezoidal steel sheeting (Reardon and Mahendran 1988) has shown that roofing has been split in the transverse direction under the screw heads due to the over tightening of screw fasteners either accidentally or by poor workmanship." It is also postulated: "the splitting phenomenon does not depend on the fastener location and is essentially a localized effect" (Mahendran and Mahaarachchi 2004). In previous studies, Mahaarachchi and Mahendran (2000) showed that splitting starts when "the longitudinal membrane tensile strain is greater that $60 \%$ of the total tensile strain at the edge of the fastener holes, and the total tensile strain is equal to the measured failure strain from tensile coupon tests of steel." (Mahendran and Mahaarachchi 2004). Later studies (Mahaarachchi and Mahendran 2008) extensively evaluated the splitting resistance of steel claddings of different geometrical shapes and material properties and postulated a strain criterion based on their previous findings. A need for a strain criterion for the pull-through failure of steel claddings was made evident when "the finite element analyses could not predict the failure loads as elastic-perfect plastic material behavior with infinite ductility is assumed without any allowance for splitting since the local pull-through failures in the less ductile steel claddings are initiated by transverse splitting at the fastener hole." (Mahaarachchi and Mahendran 2008). This study also shows that although "tensile testing of steel coupons showed that it has very little strain hardening and failure strain is only about $2 \%$," type B roofing sustained considerable "local plastic deformations without any load increase" and could sustain even greater loads after plastic deformation in contrast with type A roofing which
sustained no load increase after plastic deformation. (Mahaarachchi and Mahendran 2008). This shows that element geometry plays a critical role in the static behavior of plates. They also evaluated design equations based on their findings.

With knowledge of the static failure of plates in mind, it is now necessary to establish a relationship between static failure and fatigue failure of plates. After Hurricane Tracey struck parts of Australia, research was made in order to discover the nature of the extensive damage caused by fatigue failure. "Morgan and Beck (1977) showed that the thin crest-fixed metal roof claddings suffered a fatigue failure of the sheeting in the vicinity of the fasteners under the action of sustained fluctuating wind loading." (Mahendran 1995). A physical description of the low cycle fatigue failure of roof claddings and experimental results are given by Mahendran (1990a, b). "Field investigations and laboratory tests identified fatigue failure near the roof fasteners as the sole reason for the severe roofing damage (Beck and Stevens 1979)." (Xu 1995). Once it was determined that fatigue was essentially a localized effect subsequent research was aimed on small-scale tests. Mahendran and Mahaarachchi (2002) conducted small-scale constant amplitude load tests of steel cladding connections and compared the results to the static failure loads. They developed simple equations relating the static failure load to constant amplitude loads. They also performed multilevel cyclic test and proposed a modification factor to be used on Miner's law in order to predict fatigue damage for variable amplitude loading. Although equations exists that relate static failure loads to fatigue damage loads, these equations are only applicable to a limited number of steel cladding configurations and materials. In order to be able to predict fatigue damage in all possible cases it is necessary to establish a solid theoretical basis that accounts for the
fatigue properties of different materials and is based on analytical models of the stresses and strains that are responsible for fatigue failure. Research is needed in relating existing fatigue criteria to fatigue damage observed on steel claddings.

Emphasis is now given to different criteria in multiaxial fatigue. Several models have been established for describing multiaxial fatigue. There are stress based models, strain based models, and energy based models. "Brown and Miller (1982) reviewed much of the available multiaxial low-cycle fatigue literature with particular emphasis on the formation and early growth of cracks. Unlike octahedral shear stress, which in some cases has been shown to be effective when correlating high-cycle fatigue failure, octahedral shear strain and maximum shear strain are not effective in describing lowcycle fatigue. Brown and Miller concluded that two strain parameters are needed to describe the fatigue process. They proposed that both the cyclic shear and normal strain on the plane of maximum shear must be considered. Brown and Miller also provided a comprehensive review of the literature in terms of strain. They considered the nucleation and growth of fatigue cracks and suggested the terms Case A and Case B cracks." (Socie and Marquis 2000). They proposed separate criteria for each type of cracking which depend on loading configuration. In contrast with static failure of claddings in which failure was a function of only one parameter (membrane tensile strain), fatigue failure of claddings is a function of two parameters (cyclic shear and normal strain).

Sufficient information has been gathered regarding the static failure of crest-fixed steel claddings and experiments have been made regarding fatigue failure of crest-fixed steel claddings but a connection is still missing between static failure parameters and fatigue failure observations. A successful fatigue model should be stated in terms of the
stress and strain tensor at a given point in the structure as compared to previous models which are stated in terms of static failure load at the connection for a given geometric configuration and material. Also valley-fixed claddings, which are common in the US and Puerto Rico, need to be treated. The scope of this research is to incorporate different geometric configurations into a simple model that could be used in the prediction of the fatigue life.

## Chapter 3: Methodology

### 3.1 Introduction

There are several methods for the assessment of wind induced fatigue damage on structural components. The method to be used in each case depends on the purpose for which the data will be used. Methods can be classified as empirical, semi-empirical or analytical. Empirical methods, such as full scale testing, can be used to give a direct correlation between wind speed and total damage on a structure.


Figure 1 Wind pressure simulation by means of air bags in the experiments by
García, 2008, pp. 62.

One such method is based on full-scale wind pressure simulation, as shown in Figure 1.
In order to perform the analysis and design of structural components based on existing knowledge on the processes that cause the failure of these components, such as fatigue damage, analytical and semi-empirical methods can provide a much faster and cheaper solution without the need for expensive and time consuming experimentation.

The method used on this research is based on computational structural analysis solved by finite element methods and its application to the prediction of fatigue damage based on total life approaches. The use of this method only requires available fatigue data used to fit the fatigue models. The results obtained are compared to experimental data on the same structural component obtained from García (2008) in order to validate the analytical model.

### 3.2 Overall Methodology

Some detail is given concerning modeling screwed connections. Among thin metal roof cladding configurations, a two-span simply supported configuration is considered to be representative because it simulates a uniform wind uplift pressure and screw reactions can be estimated (Mahendran and Tang 1998). A type B wide rib cladding has been selected in this thesis as laboratory failure is well documented for this type of configuration (García 2008). The finite element software Abaqus Standard (2008) enables the user to create shell sections using CAD drawing utilities, which permits to construct the model from geometric properties. Other aspects, such as creating elasticperfectly plastic material definitions and linear and non-linear geometry analysis, are
necessary features in order to represent the complex behavior of the structural component subject to lateral wind pressures.

The method for evaluating the fatigue life of the structure proposed can be resumed by two main activities. The first set of activities is to model the static phenomenon in the structure caused by the proposed wind load. The wind load is first modeled by a loading protocol established by the Canadian Standards Association and this load is reduced to a static load $P$, which acts upon the structure. The finite element analysis program Abaqus is used to calculate the stress and strain tensors, which are needed for the fatigue model. The second set of activities is to model the repetitive fatigue effect of this load on the structure. Although the structure is initially assumed to behave in a static manner, a dynamic load fluctuation is the cause investigated for fatigue damage. Existing multiaxial fatigue criteria are evaluated considering the strain tensor obtained from the finite element model and from available fatigue parameters for the material considered. Finally the fatigue life obtained from the fatigue criteria is compared with existing experimental data.

### 3.3 Finite Element Mesh and Type of Elements

Element description
In general, elements can be of any shape on which $\xi, \eta$, and $\zeta$ define local straight coordinates, which are used in the formulation of isoparametric elements. The need to transform from rectangular elements to an element of a more general shape is necessary in the formulation of curved shell elements. Refinement of the mesh is needed in order to mesh the zones near connections where stress concentration as well as geometry requires


Figure 2 Eight node shell element (S8R) and local coordinate system adopted.
refined and special element shapes such as the triangular STRI65 element. Mindlin shell formulation (Cook, Malkus, Plesha, Witt 2002) is used to describe field quantities on elements. Shell normal displacements are interpolated as variables that are independent of cross-sectional rotation, whereas the curvatures depend on rotation and change in rotation. Consequently transverse shear strain is calculated as the mid-surface slope minus cross-sectional rotation. Nodal degrees of freedom include three translations and three rotations, totaling six degrees of freedom per node in accordance with thick shell formulation. For nonlinear geometry analysis and large displacements, the bending and membrane stiffness are coupled on the element stiffness matrix. Quadratic interpolation of geometry and field quantity is used on eight node elements. The Abaqus element library uses Mindlin formulation as standard for eight node elements with six degrees of freedom per node. A simpler formulation ignoring transverse shear strain could have been used given that element thickness can be regarded as thin. Element shape functions and displacements can be defined as in equations 1 and 2 as suggested by Cook, Malkus, Plesha, and Witt (2002).

$$
\begin{array}{ll}
N_{1}=\frac{1}{4}(1-\xi)(1-\eta)-\frac{1}{2}\left(N_{8}+N_{5}\right) & N_{5}=\frac{1}{2}\left(1-\xi^{2}\right)(1-\eta) \\
N_{2}=\frac{1}{4}(1+\xi)(1-\eta)-\frac{1}{2}\left(N_{5}+N_{6}\right) & N_{6}=\frac{1}{2}(1+\xi)\left(1-\eta^{2}\right) \\
N_{3}=\frac{1}{4}(1+\xi)(1+\eta)-\frac{1}{2}\left(N_{6}+N_{7}\right) & N_{7}=\frac{1}{2}\left(1-\xi^{2}\right)(1+\eta)  \tag{eq.1}\\
N_{4}=\frac{1}{4}(1-\xi)(1+\eta)-\frac{1}{2}\left(N_{7}+N_{8}\right) & N_{8}=\frac{1}{2}(1-\xi)\left(1-\eta^{2}\right)
\end{array}
$$

Displacements $\{\boldsymbol{u}\}$ over an element are defined by shape function interpolation as given by equation 2 .

$$
\left\{\begin{array}{l}
u  \tag{eq.2}\\
v \\
w
\end{array}\right\}=\sum N_{i}\left(\left\{\begin{array}{c}
u_{i} \\
v_{i} \\
w_{i}
\end{array}\right\}+\zeta \frac{t_{i}}{2}\left[\mu_{i}\right]\left\{\begin{array}{c}
\alpha_{i} \\
\beta_{i}
\end{array}\right\}\right)
$$

Element geometry can be interpolated in a similar fashion where $\left[\boldsymbol{\mu}_{i}\right]$ is a matrix of directional cosines defining element shape, $\alpha$ and $\beta$ are nodal rotational degrees of freedom, and $i$ refers to the node number going from one through eight. It can be shown that:

$$
\left\lfloor\begin{array}{llllll}
\varepsilon_{x} & \varepsilon_{y} & \varepsilon_{z} & \gamma_{x y} & \gamma_{y z} & \gamma_{z x}
\end{array}\right]^{T}=\sum\left[B_{i}\right]\left\lfloor\begin{array}{lll}
u_{i} & v_{i} & w_{i} \tag{eq.3}
\end{array}\right]^{T}
$$

after making appropriate coordinate transformations, where $[\mathbf{B}]$ is the strain-displacement matrix dependent on the derivatives of $[\mathbf{N}]$ with respect to global coordinates $x, y$, and $z$.

According to the Mindlin-shell formulation, matrix $[\mathbf{B}]$ is defined as:

$$
[B]=[H]\left[\begin{array}{ccc}
J^{-1} & 0 & 0 \\
0 & J^{-1} & 0 \\
0 & 0 & J^{-1}
\end{array}\right] \sum\left[\begin{array}{ccccc}
N_{i, \xi} & 0 & 0 & -\zeta t_{i} N_{i, \xi} l_{2 i} / 2 & \zeta t_{i} N_{i, \xi} l_{1 i} / 2 \\
N_{i, \eta} & 0 & 0 & -\zeta t_{i} N_{i, \eta} l_{2 i} / 2 & \zeta t_{i} N_{i, \eta} l_{1 i} / 2 \\
0 & 0 & 0 & -t_{i} N_{i} l_{2 i} / 2 & t_{i} N_{i} l_{1 i} / 2 \\
0 & N_{i, \xi} & 0 & -\zeta t_{i} N_{i, \xi} m_{2 i} / 2 & \zeta t_{i} N_{i, \xi} m_{1 i} / 2 \\
\vdots & \vdots & \vdots & \vdots & \vdots \\
0 & 0 & 0 & -t_{i} N_{i} n_{2 i} / 2 & t_{i} N_{i} n_{1 i} / 2
\end{array}\right]\left\{\begin{array}{c}
u_{i} \\
v_{i} \\
w_{i} \\
\alpha_{i} \\
\beta_{i}
\end{array}\right\}
$$

(eq. 4)
where $l, m$, and $n$ are directional cosines contained on $\left[\boldsymbol{\mu}_{i}\right]$, and $[\mathbf{H}]$ and $[\mathbf{J}]$ are defined in the form:

$$
\begin{align*}
& {[\mathbf{J}]=\left[\begin{array}{lll}
x,{ }_{\xi} & y y_{,} & z,{ }_{\xi} \\
x,{ }_{\eta} & y,{ }_{\eta} & z,_{\eta} \\
x,{ }_{\zeta} & y,{ }_{\zeta} & z,_{\zeta}
\end{array}\right]=\sum_{i}\left[\begin{array}{lllllll}
N_{i, \xi} x_{i} & N_{i, \xi} y_{i} & N_{i, \xi} z_{i} \\
N_{i, \eta} x_{i} & N_{i, \eta} y_{i} & N_{i, \eta} z_{i} \\
N_{i, \zeta} x_{i} & N_{i, \zeta} y_{i} & N_{i, \zeta} z_{i}
\end{array}\right]} \\
& {[H]=\left[\begin{array}{lllllllll}
1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
0 & 0 & 0 & 0 & 1 & 0 & 0 & 0 & 0 \\
0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 \\
0 & 1 & 0 & 1 & 0 & 0 & 0 & 0 & 0 \\
0 & 0 & 0 & 0 & 0 & 1 & 0 & 1 & 0 \\
0 & 0 & 1 & 0 & 0 & 0 & 1 & 0 & 0
\end{array}\right]} \tag{eq.5}
\end{align*}
$$

The element stiffness matrix [ $k$ ] is defined as:

$$
\begin{equation*}
[k]=\int_{-1}^{1} \int_{-1-1}^{1} \int_{1}^{1}[B]^{T}[E][B] \operatorname{det}[J] d \xi d \eta d \zeta \tag{eq.6}
\end{equation*}
$$

where $[E]$ is the material property matrix defined for plane stress and homogeneous material by making the appropriate coordinate transformation for an equivalent state of stress and strain on local coordinates $\xi, \eta$, and $\zeta$.

### 3.4 Static Model

In the first part of this research, a finite element model of the proposed structure is constructed to identify the stresses and strains caused by the proposed wind pressure.

Geometry definition
The structure geometry is defined as a 3D shell with thickness varying from 0.0299 in. $(0.759 \mathrm{~mm})$ to 0.0478 in . $(1.214 \mathrm{~mm})$ as defined in Figure 3 b and Appendix A. Three basic cases are considered in the proposed geometry: screwed connections in valleys, screwed connections in crests and washer reinforced connections in valleys as shown in figures 3c and 3d. Because of symmetry considerations, only one fourth of the cladding defined in Figure $3 b$ is represented in the finite element model as shown in Figure 3a. The blue lines in Figure 3 represent partitions for the load and element distribution, and the red lines represent axes of symmetry. Center holes at the right of Figure 3a are defined by creating a circular extrusion cut with radius defined as the inner radius in Figure 4. Table 1 lists the different configurations modeled.


Figure 3a Geometry model


Figure 3b Geometry definition


Figure 3c Type B (WR) cladding with screwed connections at valleys


Figure 3d Type B (WR) cladding with screwed connections at crests

Table 1 Configuration definition

| Model | Config.* \# | Deck Gauge | Screw Locations (Figs. 3c-d) | Load P <br> (kip) |
| :---: | :---: | :---: | :---: | :---: |
| 1 | 3 | 22 | $\begin{gathered} \text { 1-2-3-4-5-6 } \\ \text { (valleys) } \end{gathered}$ | 2.0 |
| 2 | 4 | 22 | 1-3-4-6 <br> (valleys) | 1.0 |
| 3 | 5 | 22 | $\begin{gathered} \text { 1-2-3-4-5-6 } \\ \text { (valleys-washer) } \end{gathered}$ | 2.0 |
| 4 | 6 | 22 | 1-3-4-6 (valleys-washer) | 2.0 |
| 5 | 13 | 18 | 1-3-4-6 <br> (valleys) | 1.5 |
| 6 | 15 | 20 | $1-3-4-6$ <br> (valleys) | 1.5 |
| 7 | 17 | 20 | 1-3-4-6 (valleys-washer) | 1.5 |
| 8 | 18 | 18 | 1-3-4-6 (valleys-washer) | 2.5 |
| 9 | 19 | 18 | $\begin{gathered} 1-2-3-4-5 \\ \text { (crests) } \end{gathered}$ | 1.0 |
| 10 | 20 | 20 | $\begin{gathered} \text { 1-3-5 } \\ \text { (crests) } \end{gathered}$ | 1.2 |
| 11 | 21 | 22 | $\begin{gathered} \hline 1-3-5 \\ \text { (crests) } \end{gathered}$ | 1.2 |
| 12 | 24 | 18 | $\begin{gathered} \text { 1-3-5 } \\ \text { (crests) } \end{gathered}$ | 1.2 |

[^0]

Figure 4a Detail of a center connection from shaded region in Figure 3c


Figure 4 b Direction of moments and forces acting on the section

Material and analysis
Type BWR decks used in experimental study (García 2008) are roll formed from hot-dipped galvanized steel conforming to ASTM specification A653. Results are computed for two different material definitions. In the first one, the material is defined as elastic-perfectly plastic, with modulus of elasticity $E=29,000 \mathrm{ksi}$ (200Gpa), Poissons ratio $v=0.3$, and yield stress of $33 \mathrm{ksi}(228 \mathrm{Mpa})$. In the second one, a third linear strainhardening zone is added with hardening modulus of 2Gpa after a strain of 0.02 and extending to a failure strain of 0.20 and true fracture stress of 590 Mpa as shown in Figure 5 . The point at which the strain-hardening region begins is taken to be 15 to 20 times the maximum elastic strain (Salmon, Johnson, Malhas 2009). Although realistic material behavior has a proportional limit lower than the yield strength, the flow stress starts at $2 \%$ strain, which is in good agreement with the elastic-perfectly plastic assumption (ASTM-A653 2009).


Figure 5 Strain hardening material definition

In the linear analysis the material is defined as elastic and linear in geometry. Non-linearity in geometry is used to account for large displacements and geometric nonlinearity effects in the non-linear analysis.

## Loads and boundary conditions

As an approximation, the total pressure is defined by means of an equivalent overall force applied to the complete panel, $P$, divided by the total area of the valleys where the pressure is assumed to act $\left(3.5 \times 56.5 \times 12=2373 \mathrm{in}^{2}\right)\left(1.531 \mathrm{~m}^{2}\right)$, as shown in the shaded region in Figure 3b. It is believed that this force evaluation is consistent with previous experimental studies, in which air bag pressure was applied at the valleys (García 2008). Screw holes are defined only on the center where the plate screws take most of the load, which is where the plate fails. The remaining screw holes are not defined on the plate and displacement restrictions on the three axes are defined as boundary conditions on a point where the center of the hole would be located. Refinement of the geometry of the plate near the holes with less load than the center holes is not considered to be necessary because stress and strain variations near the hole do not affect the stress fields on the other holes. At the center holes, displacement restrictions on the axis of the load are defined on the circumference of the circle defining the partition of the part around the hole (outer radius defined in Figure 4). This circumference defines the circumference of the screw head, which is responsible for the lateral reaction force per unit length in the plate. On washer reinforced connections this circumference defines the circumference of the washer which has a radius of 0.75 in .
$(19.1 \mathrm{~mm})$. On the remaining screws, the reaction force is assumed to be a point load. Symmetry boundary conditions are defined as red lines in Figure 3.

### 3.5 Modeling Screwed Connections

In the previous section two different models were used on the same screwed connection. The type of model depends on the accuracy that is required for the field quantity near the connection. In the first less accurate model, the reaction force from the screw was simply modeled as a point load. Although point loads are easy to work with they are the source of singular fields in a structural response and prevent accurate representation of the field quantities. Point loads acting on a plate does not cause discontinuities in the displacement field but produce discontinuities in the stress field, which is a derivative of the displacement field. For the center connections, where the stress and strain fields need to be calculated, more accurate representations of loads and geometry are needed.

A better representation of the screw would involve an additional finite element model of the screw including different material properties and interaction between materials, such as contact stresses and friction forces. For the present case friction forces are neglected taking in consideration that the screws are not completely tight and the reaction force is assumed to be carried through the circumference of the screw head or washer by imposing displacement restrictions normal to the undeformed perimeter on a circle defining the circumference of the screw head or washer. Deformations in the other two directions in the plane of the plate are permitted as well as lateral deformation normal to the plate under the screw head or washer. In this way membrane action of the
plate under the screw head or washer, where stress concentration is caused by the screw hole, is modeled as well as bending action caused by the screw head or washer. Additional modeling of the screw is not carried out because failure occurs on the steel cladding and not on the screw itself.

Finite element mesh
Quadratic quadrilateral-dominated elements having a random distribution are defined in order to make a transition from the circular geometry of the hole to the rectangular geometry of the span, as shown in Figure 6.


Figure 6a Complete mesh for $1 / 4$ of the cladding shown in Figure 3a corresponding to $\phi_{5}$

The random distribution is necessary where changes in geometry do not allow a strictly defined mesh pattern. A more refined distribution is assigned to a square region of 88.9 mm (3.5in) defined in Figure 4 surrounding the center screwed connections. The
general trend of the mesh being more refined closer to the center holes is made to take into account the stress concentration near the holes, as suggested by previous studies (Mahendran and Mahaarachchi 2004). Refinement of the mesh is carried out by increasing the total number of elements on the partition and then on the squared regions along the center holes. Five mesh refinements are considered by increasing the total number of elements from 1,$082 ; 7,715 ; 12,827 ; 20,742$ and 30,929 corresponding to $\phi_{1}$, $\phi_{2}, \phi_{3}, \phi_{4}$, and $\phi_{5}$ respectively. STRI65 and S8R shell elements from the ABAQUS element library are used.


Figure 6b Mesh transition from a typical center screwed connection

## Convergence and Error

In order to test for convergence and discretization error, five different meshes are used in which an irregular refinement is made by adding more elements. Values of displacement and strain are evaluated at three different points within the mesh.

The approximate element size $h$ is plotted in figures 7 and 8 against the field quantity, which can be a displacement or one of its derivatives, such as a strain. Because the element distribution is not structured and subsequent mesh refinements do not preserve the original nodal positions, monotonic convergence is not guaranteed to occur. As the element size $h$ decreases and approaches zero corresponding to an infinitely refined mesh, it is expected that the field quantity should approach the exact value. An approximate estimate may be calculated by the least squares method as the $y$-intercept in figures 7 and 8 corresponding to an element of size zero. Given the approximation of the exact value and the value obtained by a mesh with element size $h$, an estimate of the percentage of error has been calculated in Table 2.

Points A, B and C are located near the center screw holes on the bottom face, as shown in Figure 4. Strains at points A and B were found to have a larger error than at point C, given that they are located closer to the hole where the gradient in the field quantity is larger. As expected, strains were found to have larger error than displacements. The trend of greater error in strains than in displacements is explained by the fact that strains depend on a higher derivative of the field quantity than displacements and thus converge at a lower rate.

Although mesh refinement has been performed, the results of figures 7 and 8 do not indicate convergence in a direct way.


Figure 7 Displacement convergence


Figure 8 Strain convergence

Table 2 Error estimate

| Error Estimate, $\% e=\frac{\phi_{5}-\phi_{\infty}}{\phi_{\infty}} 100 \%$ |  |  |
| :--- | :--- | :--- |
|  | Displacement | Principal Strain |
| Point A | 0.17 | 9.8 |
| Point B | 1.31 | 11.5 |
| Point C | 1.34 | 1.85 |

where $\phi_{5}$ corresponds to the mesh with $h=0.8149 \mathrm{~mm}$ and $\phi_{\infty}$ to the estimated condition approaching $h=0$.

Although singularities at the edge connections at the top of Figure $3 b$ can cause greater error on field quantity predictions, there are no singularities at the center connections given that more accurate representation of the screws prevents large load reactions to appear at element nodes. Boundary conditions imposed on the screw head or washer can be the source of high linear reaction loads, which may yield unusually high values in derivatives of the field quantity. The error is attributable to the lowest degree term omitted in the interpolation function and depends on element size $h$, the degree of the highest complete polynomial in element field quantity $p$, and $r^{\text {th }}$ derivative of the field quantity as suggested by Cook, Malkus, Plesha, and Witt (2002) in the form:

$$
O\left(h^{p+1-r}\right)
$$

where $O$ is a measure of the order of error.
For quadratic elements $p=2, r=0$ for displacements, and $r=1$ for strains. Considering the element formulation in Section 3.3, in which the field quantity is represented by interpolation functions in equation 1, the highest complete polynomial is of order 2 since
all the terms of order 2 are present, namely $\xi^{2}, \xi \eta$, and $\eta^{2}$. The error can then be traced to the omission of terms of order 3 in equation 1 given that a complete polynomial of order 3 cannot be defined. As a result a recursive equation can be defined in terms of the field quantity and the order of error.

$$
\begin{align*}
& \phi_{n}=\phi_{\infty}+e_{n} \\
& \phi_{n+1}=\phi_{\infty}+e_{n} * \frac{O \phi_{n+1}}{O \phi_{n}} \tag{eq.8}
\end{align*}
$$

Using an order of error analysis, the order of error in each derivative of the field quantity is given for each mesh refinement in tables 3 and 4. The right column in tables 3 and 4 gives how many times the error is reduced upon each subsequent mesh refinement where $h$ is plotted in figures 7 and 8 against the field quantity.

Table 3 Displacement order of error

| Order of Error $O\left(h^{p+1-r}\right) ; p=2, r=0$ |  |
| :--- | :--- |
| $O \phi_{1}=116.9$ | $O \phi_{1} / O \phi_{2}=27$ |
| $O \phi_{2}=4.33$ | $O \phi_{2} / O \phi_{3}=2.37$ |
| $O \phi_{3}=1.83$ | $O \phi_{3} / O \phi_{4}=1.95$ |
| $O \phi_{4}=0.935$ | $O \phi_{4} / O \phi_{5}=1.73$ |
| $O \phi_{5}=0.541$ |  |

Table 4 Strain order of error

| Order of Error $O\left(h^{p+1-r}\right) ; p=2, r=1$ |  |
| :--- | :--- |
| $O \phi_{1}=23.9$ | $O \phi_{1} / O \phi_{2}=9$ |
| $O \phi_{2}=2.66$ | $O \phi_{2} / O \phi_{3}=1.78$ |
| $O \phi_{3}=1.49$ | $O \phi_{3} / O \phi_{4}=1.56$ |
| $O \phi_{4}=0.956$ | $O \phi_{4} / O \phi_{5}=1.44$ |
| $O \phi_{5}=0.664$ |  |

Since $h$ is of degree 3 in displacements and 2 in strains in the order of error formulation, a linear relationship between error and element size $h$ is not expected. Given this proof the method of error estimation in Table 5 should provide a more accurate error estimate than the method used on figures 7 and 8 . The error in $\phi_{1}$ in Table 5 is calculated by using $n=$ 1 on equation 8 where $\phi_{1}$ and $\phi_{2}$ are displacements and strains from figures 7 and 8 .

Table 5 Percent error

| Error $e_{n}$, Percent Error \%e |  |
| :---: | :---: |
| Displacement Point A (mm) | Strain Point A (mm/mm) |
| $\begin{array}{cc} \hline \phi_{1} & 0.00293,1.1 \% \\ \phi_{2} & 0.000109,0.04 \% \\ & \\ \phi_{3} & 0.000046,0.02 \% \end{array}$ | $\begin{array}{cl} \hline \phi_{1} & 0.1895,26.2 \% \\ \phi_{2} & 0.0211,2.9 \% \\ \phi_{3} & 0.0118,1.6 \% \\ \phi_{4} & 0.0076,1.0 \% \end{array}$ |

By substituting the order of error from tables 3 and 4 and solving for $\phi_{\infty}$ and $e_{1}, e_{n}$ can then be taken as $e_{n-1} * \frac{O \phi_{n}}{O \phi_{n-1}}$. The largest reduction in error occurs by reducing element size from $\phi_{1}$ to $\phi_{2}$ as shown on figures 7 and 8 and tables 3 and 4. From Table 5 displacement converges in $\phi_{1}$ with a $1 \%$ error while strain converges on $\phi_{4}$. As a result an element size $h=1 \mathrm{~mm}$ corresponding to $\phi_{4}$ meets convergence requirements. Figure 6 is an example of $\phi_{5}$.

## Chapter 4: Results of Static Analysis

### 4.1 Introduction

A basic case has been initially solved in this Chapter. Three different models are considered in the finite element analysis to understand the effects of non-linearity in material and in geometry. In the first model the material is defined as elastic with no plastic zone defined and linear in geometry. In the second model, non-linear elasticity is considered. In the third model, plasticity as well as non-linearity in its geometry are considered.

### 4.2 Linear and Elastic Results

These results are calculated for configuration \#3 defined in Table 1. For the linear case, the von Mises stresses and out-of-plane displacements are calculated. Figure 9 shows out of plane displacements and von Mises stresses on the envelope (maximum values) of the top and bottom surfaces measured by the distance from the edge of the center hole on the $x$-axis, as defined in Figure 4. It can be seen that the out of plane displacement is zero at 5 mm on boundary 1 , corresponding to a boundary condition imposed on the screw head. Figure 9 also shows the stress concentration near to the hole at the peak of the curve. The values are unusually high (values higher than 1500 Mpa at 5 mm from the center hole) even for this dense finite element mesh; they will later be shown to be due to the assumption of elastic behavior made in the present model. Figure 10 shows the same results on an axis parallel to the previous one, with 1.75 in ( 44.5 mm )
offset (boundary 3). In this figure the slopes are less pronounced given that less deformation and hence stresses exist at some distance away from the hole.


Figure 9 Linear results at boundary 1 in Figure 4


Figure 10 Linear results, boundary 3 in Figure 4

Figure 4 b shows the direction of the bending moments and forces that are plotted in figures 11 and 12. Figure 11 shows membrane stresses acting perpendicular to the
boundaries of Figure 4, where $x=0$ represents the edge of the hole. Membrane stresses on boundaries one and three in Figure 11 are very similar except at a short distance from the hole, although boundary 1 is closer to the hole given that the reaction force of the screw is transmitted by bending and not by membrane stresses in a geometrically linear analysis not accounting for large defections. On the other hand, the stresses on boundary 2 are much lower because most of the bending action is taken in the longitudinal direction, as in the beam model where transverse deformation is zero. At the location of boundary 3 the membrane stress field is almost uniform, indicating that it is not affected by the connection. This value is well below the yield stress.

Figure 12 shows the bending moments on the axes of the boundaries. Figure 13 shows the shear stresses on the plane of the plate. The results may be interpreted as follows: the membrane stresses are larger on boundaries along the $x$-axis given that bending of the cladding in the longitudinal direction causes it to behave as a beam with tension and compression acting at the outermost fibers. The results of Figure 12 indicate that the moment along the boundary 1 grows fast as the distance from the hole is decreased because in a linear-elastic model not accounting for large displacements all of the load is transferred by bending action. Figure 13 shows zero shear on boundary 1 because of symmetry. The resultant force acting along the outer radius of a center connection in configuration \#3 is $814-\mathrm{N}$ acting perpendicular to the plate. This force represents the reaction force of the screw and displacements and forces in figures 11 to 13 are proportional to this force given a linear-elastic model. These results show large displacements and stresses well beyond the linear range.


Figure 11 Linear results, membrane stress for load $P$


Figure 12 Linear results, moment for load $P$


Figure 13 Linear results, shear stress for load $P$

According to thin plate theory, geometric linearity is fully justified provided displacements are smaller than the plate thickness. The material yield strength is 33 ksi $(228 \mathrm{Mpa})$ and the von Mises stresses are ten times larger, as shown in figures 9 and 10.

Because the results provide values that are in excess of linear theory, it is now justified to explore nonlinear behavior including geometric as well as material nonlinearity.

### 4.3 Geometrically Non-linear and Elastic Analysis

In this section, the von Mises stresses are calculated on boundaries 1 and 3 as in figures 9 and 10 but now for the geometrically non-linear case as shown in figures 14 and 15. Geometric non-linearity is a kinematic non-linear dependence of strains on displacements; this causes that the structure stiffness matrix becomes dependent on displacements. For plates with large displacements, membrane action may dominate over bending and is a consequence of the dependence of the stiffness matrix on the geometry of the structure.

In order to solve a problem where the stiffness matrix is not known in advance, it is necessary to perform a series of linear increments, in which the initial stiffness matrix for the case of zero displacement is modified. At each increment, an initial estimate of the displacement is used to calculate the new stiffness matrix and then a new displacement can be calculated based on the increment size, which is the change in load required in order to reach equilibrium. In the execution of the $A B A Q U S$ model, the increment size is set to "automatic," so that an algorithm determines an appropriate increment size based on the computed results.


Figure 14 Geometrically non-linear and elastic results at boundary 1 in Figure 4


Figure 15 Geometrically non-linear and elastic results at boundary 3 in Figure 4

The procedure is continued until equilibrium is reached and the calculated displacement is a small fraction of the original displacement. The solution technique used by ABAQUS is the full Newton's method, in which the stiffness matrix used at each increment is modified by the current displacement estimate and an equivalent or tangent stiffness in the load versus displacement curve is used to calculate the new estimate. By comparing both analyses it is seen that displacements and stresses on the geometrically non-linear analysis are less than half of those computed in the linear analysis. This can be explained by the fact that membrane action provides additional stiffness not considered on the geometrically linear analysis. Because stresses are still well over the yield strength a nonlinear analysis is justified.

### 4.4 Geometrically Non-linear and Elastic-Plastic Analysis

In this third analysis, two types of non-linearity are included simultaneously: namely material and geometric non-linearity. Material plasticity introduces additional complexity to the problem. When the yield strength is exceeded in an elastic-plastic material, the loading history plays an important part on the final state of the structure. For the present case the load is assumed to be statically applied and time independent.

In order to perform an elastic-plastic analysis, three aspects must be taken in consideration: the yield criterion, the flow rule, and the hardening rule. Normally the yield criterion determines when the material starts yielding and it is stated in terms of the von Mises or effective stress. Once this point is reached, the flow rule relates the state of stress to the state of strain in the material. For an elastic perfectly plastic material there is no hardening and increments in stress beyond the yield point are not possible. In order to
determine the response of the structure, an incremental analysis is used in which the elastic modulus depends on the state of stress and elastic and plastic strains are treated independently.

### 4.5 Geometrically Nonlinear Results Including Ideal Plasticity

The finite element analysis of configuration \#3 concluded with 23 increments, in which the occurrence of nonlinear behavior is evident as expected. Stress concentration occurred in the vicinity of the center holes with significant plasticity. Figures 16 and 17 show displacement and von Mises stresses computed at the same place as in the previous linear results but now for the nonlinear and elastic-plastic assumption. The displacements in Figure 16 are nearly half the displacements plotted in Figure 9 and many times more than the shell thickness, indicating that a nonlinear analysis is necessary in order to predict a more representative behavior of the structure. As compared to Figure 14, displacements in Figure 16 are a bit higher considering ideal plasticity. On the other hand, the values of von Mises stresses demonstrate that plasticity spreads in a zone extending about 15 mm from the hole on the $x$-axis and 65 mm on the $y$-axis as defined in Figure 4 and Figure 18 using orange color. The von Mises stresses in this case are computed as the envelope (maximum stresses) of the top and bottom surfaces in figures 16 and 17 and at the bottom surface in Figure 16.

Using the static strain criterion formulated by Mahaarachchi and Mahendran (2008), which states that failure occurs when the membrane tensile strain is $60 \%$ of the maximum surface tensile strength and the maximum surface tensile strain is the failure strain in a tensile coupon test, in conjunction with the finite element model, it is
concluded that static failure has not occurred. The maximum tensile strain is the failure strain or elongation as given on the ASTM standard, which is 0.20 , and the membrane tensile strain reached its peak at the edge of the hole as will be explained in the next section.


Figure 16 Non-linear results at boundary 1


Figure 17 Non-linear results along boundary 3, computed as the envelope of the top and bottom surfaces


Figure 18 Plastic zone for load $P=5.81 \mathrm{kPa}$ evaluated at the bottom surface


Figure 19 Plastic zone for load $0.25 P$ at the bottom surface

### 4.6 Typical Center Connection

In a typical screwed connection, the direction of the major principal strain results are shown in Figure 20, where $x$ is the distance from the edge of the hole in the $x$-axis defined in Figure 4. The major principal strain corresponds to the radial strain in the xaxis in every point, except for a small region around the circumference of the screw head, where the tangential strain corresponds to the major principal strain as can be seen in Figure 20 at a distance of about 5 mm from the hole. As a result, the maximum shear strain acts at 45 degrees from the principal direction in this configuration. The octahedral shear strain is defined in terms of the components of the strain tensor as:

$$
\Delta \varepsilon_{e q}=\frac{1}{\sqrt{2}(1+v)} \sqrt{\left(\Delta \varepsilon_{x}-\Delta \varepsilon_{y}\right)^{2}+\left(\Delta \varepsilon_{y}-\Delta \varepsilon_{z}\right)^{2}+\left(\Delta \varepsilon_{x}-\Delta \varepsilon_{z}\right)^{2}+\frac{3}{2}\left(\Delta \gamma_{x y}^{2}+\Delta \gamma_{y z}^{2}+\Delta \gamma_{x z}^{2}\right)}
$$

The equivalent or octahedral shear strain is expressed in terms of the normal and shear strain components of the strain tensor at a point on the structure as shown in Figure 21.

The peak of the radial strain curve corresponds to a very steep slope or change of the radial strain in the top and bottom surfaces, which is equivalent to a change in radial moment. Consequently this change in moment can be expressed as a shear stress, according to plate theory, corresponding to the screw head reaction force acting at the position of the peak of the curve. The direction of the mayor principal strain corresponds to the $x$-axis in Figure 4 and its magnitude with the bottom surface where the wind pressure is applied. From this information the mechanism by which the steel cladding resists the reaction force of the screw is determined.


Figure 20a Strain concentration (major principal) at center connection, boundary 1


Figure 20b Strain concentration (components) at center connection, boundary 1

If the diameter of the screw head were to be increased, as by adding a washer, it would be equivalent to shifting the radial strain curve in Figure 20 b to the right as discussed in Figure 20a where the peak of the curve is caused by the vertical reaction shear force of the screw head or washer. In other words, the washer has the effect of expanding the circumference where the vertical reaction force acts, thus lowering the vertical shear stress as it is distributed over a larger perimeter. Because shear can be expressed in terms of change in bending moment, the change in moment is also lowered. This would translate in a less pronounced peak on the radial strain curve.


Figure 21 Octahedral shear strain at center connection, boundary 1

### 4.7 Discussion of Results

In the previous section on modeling of screwed connections, a model of the connection was defined where bending as well as membrane stresses were taken into account and appropriate boundary conditions defined. Now we can see the role that
bending and membrane components play in resisting the reaction force from the screw and the effect of the screw hole on strains and stresses. Radial strains in the top and bottom surfaces, which are a measure of curvature, increase rapidly as they get closer to the circumference of the screw. Curvature in the radial direction (boundary 1) indicates that bending is responsible for high radial strains near the screw head. Strains on the midsurface are above the yield strain under the screw head and are amplified by a factor of three by stress concentration around the screw hole. Comparing the linear analysis results with the nonlinear results may highlight the effect that the membrane stresses have. The nonlinear results show that displacements are almost half of the displacements in the linear analysis when considering non-linearity in geometry given that membrane action provides additional stiffness not considered in a linear analysis.

## Chapter 5: Results of Fatigue Analysis

### 5.1 Fatigue Analysis Methodology

In order to predict fatigue life using total life approaches, several aspects need to be taken into consideration. First, it is necessary to determine if a stress-based or strainbased approach is appropriate. In order to use either approach, multiaxial stress effects, mean stresses, and variable amplitude loading have to be taken into account.


Figure 22 Completely reversed loading cycle $(\mathrm{R}=-1)$

In general a loading cycle is composed of elastic and plastic deformation components as shown in Figure 22. For the present case it is assumed that loading is not completely reversed since the load $P$ is defined to act always in the same direction and instead the stress ratio $R=0$ is used for tension loading in the bottom of the plate. From the finite element model the maximum stresses and strains occur on the bottom surface
because of stress concentration at the edge of the hole on the center connections as defined in Appendix B. When loading is removed the stresses return to zero and permanent plastic deformation remains, as shown in Figure 23. The fatigue process depends on several variables depending on the fatigue model to be used. These are maximum normal stress and strain amplitude, octahedral shear stress amplitude and mean stresses. It is noted that fatigue damage is independent of mean strains and hence, total plastic deformation. Since loading and unloading does not cause further plastic deformation on any given loading sublevel we are left with the elastic part of the cyclic hysteretic loop depicted by the dark colored line on the right of Figure 23. It is noted that some metals exhibit cyclic hardening while others exhibit cyclic softening when subjected to repetitive loading as shown on the light colored lines in Figure 23. The curved line to the right represents cyclic softening behavior characterized by having a yield strength lower than the nominal value and plastic deformation taking place as cycling continues. In comparison, the light colored line to the left represents cyclic hardening in which higher loads eventually increase the yield strength but without cyclic plasticity taking place on the same load level. This will have an effect on the monotonic behavior as compared to the cyclic stress-strain response for a given material.


Figure 23 Plastic deformation and hysteretic loop for tension loading $(\mathrm{R}=0)$ on one sublevel of the Sidgers protocol defined in Figure 24

If cyclic softening is considered one would expect to have a small increase on total plastic strain consisting of the sum of each individual plastic strain component taking place at each cycle until a stable behavior is achieved at a considerable portion of the fatigue life or by structural constraint along stress gradients or redundant equilibrium of forces on the structure. These plastic strain components are considered to be a small portion of the total strain amplitude and may be neglected on the fatigue damage predicted by equation 9 . On the other hand if cyclic hardening is considered, the elastic strain amplitude may increase by a small amount at some fraction of the fatigue life until
reaching a stable behavior without having cyclic plastic strain. For metals with an ultimate strength to yield strength ratio between 1.2 and 1.4 , which is the case here, a metal is said to be generally stable but may harden or soften (Mitchell 1992). A stable behavior in which there is no cyclic hardening or softening is considered for the present case based on the ultimate to yield strength ratio.

In the strain-life curve also called the Coffin-Manson relationship, the strain amplitude $\varepsilon_{a}$ is assumed as the sum of elastic and plastic components in the form:

$$
\begin{equation*}
\varepsilon_{a}=\frac{\sigma_{f}^{\prime}}{E}\left(2 N_{f}\right)^{b}+\varepsilon_{f}^{\prime}\left(2 N_{f}\right)^{c} \tag{eq.9}
\end{equation*}
$$

where $b$ is the slope of the elastic part and $c$ is the slope of the plastic part of the strainlife curve on a logarithmic plot (Coffin 1971). If we eliminate the plastic component on the right side and multiply both sides by $E$, we are left with the classical stress-life equation as:

$$
\begin{equation*}
\sigma_{a}=\sigma_{f}^{\prime}\left(2 N_{f}\right)^{b} \tag{eq.10}
\end{equation*}
$$

where $\sigma_{a}$ is the alternating stress and $\sigma_{f}{ }_{f}$ is the true fracture stress. In order to take into account mean stress effects, Morrow (1965) proposed to modify the number of cycles to failure as follows:

$$
\begin{equation*}
N_{f}=N_{M}\left(1-\frac{\sigma_{m}}{\sigma_{f}^{\prime}}\right)^{1 / b} \tag{eq.11}
\end{equation*}
$$

Substituting for $N_{f}$ in equation 9 gives:

$$
\begin{equation*}
\varepsilon_{a}=\frac{\sigma_{f}^{\prime}}{E}\left(1-\frac{\sigma_{m}}{\sigma_{f}^{\prime}}\right)\left(2 N_{M}\right)^{b}+\varepsilon_{f}^{\prime}\left(1-\frac{\sigma_{m}}{\sigma_{f}^{\prime}}\right)^{c / b}\left(2 N_{M}\right)^{c} \tag{eq.12}
\end{equation*}
$$

where $\sigma_{m}$ is the mean alternating stress. Upon eliminating the plastic component, we are left with the stress-life relationship that includes the mean stress effect.

$$
\begin{equation*}
\sigma_{a}=\left(\sigma_{f}^{\prime}-\sigma_{m}\right)\left(2 N_{M}\right)^{b} \tag{eq.13}
\end{equation*}
$$

Here the stress amplitude and mean stress are calculated from the major principal stress.
Goodman (1899) proposed the following relation for addressing mean stresses:

$$
\begin{equation*}
\frac{\sigma_{a}}{\sigma_{a m}}+\frac{\sigma_{m}}{\sigma_{u}}=1 \tag{eq.14}
\end{equation*}
$$

where $\sigma_{a m}$ is the equivalent stress amplitude for completely reverse loading that addresses the mean stress effect and the stress amplitude and mean stress are calculated from the Von Mises stress. Using the classical stress-life equation in terms of $\sigma_{a m}$ and a modified number of cycles to failure $N_{G}$ :

$$
\begin{equation*}
\sigma_{a m}=\sigma_{f}^{\prime}\left(2 N_{G}\right)^{b} \tag{eq.15}
\end{equation*}
$$

Substituting for $\sigma_{a m}$ from equation 14:

$$
\begin{equation*}
\sigma_{a}=\sigma_{f}^{\prime}\left(1-\frac{\sigma_{m}}{\sigma_{u}}\right)\left(2 N_{G}\right)^{b} \tag{eq.16}
\end{equation*}
$$

The Morrow equation is similar to that due to Goodman, except for using the true fracture stress instead of the ultimate strength.

Another approach to the mean stress effect is due to the Smith, Watson, and Topper parameter (1970), which was developed for materials that fail predominantly on maximum tensile stress or strain planes. According to SWT the life for any situation of mean stresses depends on the product of maximum stress times strain amplitude on the principal plane and the resulting equation is:

$$
\begin{equation*}
\sigma_{\max } \varepsilon_{a}=\frac{\left(\sigma_{f}^{\prime}\right)^{2}}{E}\left(2 N_{S W T}\right)^{2 b}+{\sigma^{\prime}}_{f}{\varepsilon^{\prime}}_{f}\left(2 N_{S W T}\right)^{b+c} \tag{eq.17}
\end{equation*}
$$

where $\sigma_{\max }$ is the major principal stress and $N_{S W T}$ is the corresponding number of cycles to failure. Upon eliminating the plastic component of strain the SWT equation results in:

$$
\begin{equation*}
\sigma_{\max } \varepsilon_{a}=\frac{\left(\sigma_{f}^{\prime}\right)^{2}}{E}\left(2 N_{S W T}\right)^{2 b} \tag{eq.18}
\end{equation*}
$$

or

$$
\begin{equation*}
\sigma_{a}=\frac{\left(\sigma_{f}^{\prime}\right)^{2}}{\sigma_{\max }}\left(2 N_{S W T}\right)^{2 b} \tag{eq.19}
\end{equation*}
$$

Fatigue data collected for different materials shows that the fatigue process may be divided into three regions: nucleation, shear, and tension. The extent of each region in the fatigue life depends on the material and type of loading. As a result the choice of the fatigue model will also depend on these factors. Evidence shows that at a high number of cycles fatigue is dominated by tension and at low numbers by shear (Hua and Socie 1985; Socie et al. 1989; Bannantine and Socie 1985). Also at high lives or number of cycles, fatigue is dominated by elastic deformation as compared to plastic deformation at low lives as predicted by equation 9. Upon this premise it can be concluded that tension damage is dominated by elastic deformation and shear damage by plastic deformation. This fact is also seen in brittle materials, which fail in tension with no plastic deformation, as opposed to more ductile materials, which fail predominantly in shear and exhibit large plastic deformation.

In our case, where there is a mean stress, there is an exception from the above discussion, which applies for completely reversed cycles with zero mean stress. In this case, considering only tension loading, fatigue will be dominated by elastic deformation
at low lives. This is a direct consequence of the mean stress effect given that mean stresses increase the fatigue damage caused by elastic deformation at low lives. As a result damage will be accumulated in the low cycle range and failure will occur before reaching the high cycle range. Tension models such as SWT should work better in predicting the fatigue life. Other critical plane models, such as Brown and Miller, and Fatemi and Socie, which were developed predominantly for shear damage require tension and torsion fatigue test data, which is not readily available.

Finally the Palmgren-Miner rule is used in order to take into account variable amplitude loading.

$$
\begin{equation*}
\sum_{i} \frac{n_{i}}{\left(N_{f}\right)_{i}}=1 \tag{eq.20}
\end{equation*}
$$

where $n$ is the number of cycles at the stress level $i$ at which $N_{f}$ cycles would cause failure. Palmgren-Miner's linear damage rule has some limitations because the order in which distinct amplitude cycles are applied is found to have an effect on fatigue damage. Mahendran and Mahaarachchi (2002) suggested that Miner's rule should be modified in order to take into account experimental evidence from multilevel cyclic tests performed on claddings. The modified Miner's rule is as follows:

$$
\begin{equation*}
\sum_{i} \frac{1}{K} \frac{n_{i}}{\left(N_{f}\right)_{i}}=F \tag{eq.21}
\end{equation*}
$$

where $K$ is a modification factor equal to 0.7 and $F$ equals the fatigue damage being zero for no damage and 1 for $100 \%$ damage, meaning that the fatigue life or number of cycles to failure has been reached. Although a better model can be constructed in which damage can be divided depending on the nature of loading, such as shear or tension, the total life is computed with just one model because in this case we are not dealing with combined
loading. In order to get the equivalent stress and strain amplitude, the load sequence shown in Figure 24 is used on the finite element model and the number of cycles in each sublevel is multiplied by 5 . For any level greater than E , the same sequence as on level E is used but the maximum load is increased by 0.25 on each subsequent level (e.g. level F has the same number of cycles and the same sequence as level E but with a load $\mathrm{P}=$ 2.25). For each load increment the cyclic stress-strain diagram has a specific path in which the order of load increments as well as loading history are necessary to determine the fatigue damage, as depicted in Figure 25.


Figure 24 Load sequence used in Sidgers protocol (Baskaran et al. 2006)


Figure 25 Cyclic stress-strain diagram
As the load increases in the first loading sequence, the equivalent strain passes the yield point until it reaches a plastic strain corresponding to a load of $0.25 P$. When the load is removed in the first cycle the unloading path follows the second line until zero stress and permanent deformation remains. The loading and unloading cycle of the first load increment will remain on the second line as depicted in Figure 25 and considering a stable behavior as discussed in Figure 23. In the next load increment the process is repeated by shifting another half cycle to the right until the maximum increment corresponding to load $P$ is reached on sublevel A4 of Figure 24 . When the load is decreased on sublevel A5 from $P$ to $0.25 P$ and then increased to $0.5 P$ the loading and unloading path remains as on sublevel A4, except that it will cover a smaller portion of stress and strain corresponding to 25 percent of the total trajectory. This is due to the strain hardening effect that the higher load had on the material and the stress and strain will be proportional to the load at any given unloading and reloading path for which the load does not exceeds the previously applied maximum load. The proportionality of the
stress and strain in the unloading and reloading paths can be proven since on any specific path the maximum strain range will not exceed the elastic strain range for the elasticplastic material definition on any given point in the structure. Because elastic strains are very small in comparison with plastic strains, the structure will not undergo large deformations beyond the previous permanent deformation in the elastic strain range. The element stiffness matrix will remain unchanged within any specific elastic strain range, and thus the proportionality assumption will remain valid. The remaining load increments on level A will remain on the same path but covering a greater portion depending on the amount the load is increased. For example consider load sublevel A6. Because the maximum load on sublevel A6 is $0.75 P$ which is less than $P$ from load sublevel A4, the loading and unloading path will remain as on sublevel A4 but the maximum stress will be $75 \%$ of the maximum stress on load sublevel A4 and the minimum stress would be $25 \%$ as proportional to the load. Sublevels B2 and B3 will also remain on the same path as no further plastic deformation is taking place. Upon reaching sublevel B4 corresponding to 1.25 $P$ the loading path is continued until the total strain reaches the next cyclic step. The cyclic stress-strain diagram can be resumed by plastic and elastic strain components, where plastic strains correspond to horizontal lines and elastic strains to vertical lines in Figure 25. Because cyclic loading is not completely reversed and instead only tension loading acts at the bottom of the plate, where stress concentration occurs, only elastic strain plays an important role on fatigue. The other half cycles corresponding to transitions between load increments, where plastic deformation occurs, are ignored for determining the fatigue life, first because plastic fatigue parameters are unknown, and
second because the fatigue damage done on these half cycles is not believed to be significant.

There are several methods for approximating fatigue parameters from monotonic tensile tests. The method of universal slopes (Manson 1965) is chosen in this thesis since it will be proven to yield good results later. From the method of universal slopes the fatigue strength coefficient and fatigue strength exponent are approximated from monotonic tensile tests parameters by the following equations.

$$
\begin{aligned}
& \sigma_{f}^{\prime}=1.9018 \sigma_{u} \\
& b=-0.12
\end{aligned}
$$

Another method, which only requires the ultimate strength, is the Uniform Material Law (Bäumel and Seeger 1990). From this method the fatigue parameters can be approximated by the following equations.

$$
\begin{gathered}
\sigma_{f}^{\prime}=1.5 \sigma_{u} \\
b=-0.087
\end{gathered}
$$

### 5.2 Fatigue Analysis Results

Using the total life approach, the strain-life method yields the same result as the stress-life method because of the load cycle occurring in the elastic regime. Results are compared for three different approaches of mean stress analysis and for two material definitions. Appendix C lists accumulated fatigue damage at each sublevel of the loading sequence for the elastic perfectly plastic material assumption and Appendix D is the same but with the strain hardening material assumption. These three approaches considered,
namely Morrow, Goodman, and SWT, are based on the stress life equation 10, which accounts for the elastic strain range term of the strain-life equation.

Twelve cladding configurations were studied and compared with results obtained by García (2008). In these configurations thickness, location and type of screwed connections, and load $P$ were varied as described in Section 3.4. From these, four are valley fixed, four are crest fixed and four are washer reinforced valley fixed connections. Appendix C and D list the fatigue damage calculated at each loading sublevel for the elastic-plastic and strain hardening material definitions respectively. From the finite element and fatigue models all configurations that yielded in the first loading sublevel yielded the same results for the von Mises stresses and the elastic-plastic material definition in contrast with the strain hardening material definition given that after first yield the calculated von Mises stress is equal to the yield stress for every configuration in the elastic-plastic material definition. For this reason the fatigue damage calculated with the Goodman approach, which is used with the von Mises stress, is expected to yield the same fatigue damage for every configuration on a given sublevel that yielded in the first loading sublevel.

For the comparison of fatigue models the reader is referred to Figure 26. The number of cycles to failure in this figure are $N_{M}$ for Morrow, $N_{G}$ for Goodman, and $N_{S W T}$ for the Smith, Watson and Topper methods. The figure uses a mean stress of 113.8 Mpa, which is half of the yield strength, and the universal slopes method. The horizontal line crosses the curves at an alternating stress equal to the mean stress of 113.8 Mpa for tension loading. At this alternating stress SWT falls between Goodman and Morrow. The Goodman approach is the most conservative taking into account that Goodman gives
good results for brittle materials and is more conservative for ductile materials. This should be the case given that the fatigue damage is being caused by elastic deformation and a brittle failure is expected.


Figure 26 Comparison of Fatigue Models for the universal slopes method

### 5.3 Discussion of Results and Parametric Studies

Equation 10 is the basic stress-life equation that is used to calculate the fatigue life. In order to use this equation, the two parameters $\sigma_{f}^{\prime}$ and $b$, which are not provided in the ASTM standard, need to be calculated. The first one is the true fracture strength or fatigue strength coefficient and the second one is the slope of the $\log -\log$ plot of the
elastic component of the equation or the fatigue strength exponent. These two parameters are approximated by the universal slopes method and the uniform material law and correlated to monotonic tensile test parameters. These parameters are usually used to fit the strain-life equation to the fatigue tests data.

Figure 27 shows the fatigue damage for the 12 configurations defined in Table 1 calculated and plotted at the middle of the loading sublevel in which failure was observed in experiments performed by García (2008). Figure 28 shows the same results but with the strain hardening material definition instead. From these two figures it is concluded that the elastic-plastic material definition provides a better correlation for all three of the fatigue models than the strain hardening material definition since Figure 27 shows less scattering around the expected value of $F=1$.


Figure 27 Comparison of fatigue models for the elastic-plastic material definition


Figure 28 Comparison of fatigue models for the strain hardening material definition

Figure 29 is an expansion of Figure 27 and Table 6 lists the average and standard deviation of the fatigue life computed for each model. From the last row of Table 6 it is concluded that SWT provides the best correlation with a value of dispersion of nearly half of the other two fatigue models since SWT uses two parameters from the stressstrain tensor instead of one.

Figure 30 shows the SWT approach on Figure 29 for the different types of connections. From Figure 30 it is concluded that the fatigue damage calculated is independent of the type of connection, number of screws and configuration meaning that the SWT - elastic-plastic model works well with all 12 configurations predicting an
average fatigue damage of 0.875 . This value suggests that the fatigue model should be modified in order to fit the experimental data for which the expected fatigue damage is 1 .


Figure 29 Comparison of fatigue models for the elastic-plastic material definition

Table 6 Fatigue damage average and standard deviation for the 12 configurations

|  | Goodman | Morrow | SWT |
| :---: | :---: | :---: | :---: |
| $\mu$ | 1.269 | 0.623 | 0.875 |
| $s$ | 0.1296 | 0.0671 | 0.0392 |
| $s / \mu$ | 0.1021 | 0.1078 | 0.0448 |

where $\mu$ is the sample mean, $s$ is the sample standard deviation and $s / \mu$ is a measure of dispersion of the fatigue life $F$.


Figure 30 Comparison of connection types and fatigue damage

### 5.4 Development of a Suitable Parameter Estimation Method

In the previous section a fatigue model and material definition was found that correlated with the experimental data. The fatigue parameters used in the previous section were calculated using the universal slopes method proposed by Manson (1965). This method was developed mainly for completely reversed loading $(R=-1)$, for a wide range in the number of cycles, and for the classic strain-life equation 9. In this section a new parameter estimation method is proposed for tension loading $(R=0)$, low cycle fatigue, and to be used with the SWT equation 19.

In the previous section the dispersion was defined as the ratio of sample standard deviation to the mean in the calculated fatigue life for the 12 configurations. This ratio is
used in order to compare the dispersion of different models relative to the mean. In order to adjust the SWT - elastic-plastic model to more precisely fit the experimental data, the fatigue strength exponent $b$ is changed using an iterative approach in order to minimize the dispersion. This parameter has to do with the shape of the curve in Figure 26. Once the dispersion is minimized, the true fracture stress $\sigma_{f}^{\prime}$ is changed using an iterative approach so that the mean of the calculated fatigue life of the 12 configurations reaches the expected value of 1 . Changing the true fracture stress has the effect of stretching the curve in Figure 26 since it is a coefficient on equation 19 without affecting dispersion. Using this approach the fatigue parameters $\sigma_{f}{ }_{f}$ and $b$ were found.

$$
\begin{gathered}
\sigma_{f}^{\prime}=1.87 \sigma_{u} \\
b=-0.12
\end{gathered}
$$

Since the dispersion did not decreased by a significant amount by changing the fatigue strength exponent $b$ then it is concluded that the value suggested by the universal slopes method of -0.12 is suitable to be used with the SWT fatigue model.

Figure 31 and Table 7 show the same results as Figure 30 and Table 6 but with the values of the fatigue parameters used to fit the experimental data proposed in this new method. Results are listed in Appendix E.

The fatigue damage calculated at the middle of the sublevel of the observed failure for the 12 configurations and for the proposed method varies between 0.948 and 1.089 damage with a mean of 1 and sample standard deviation of 0.0448 . This is the difference in fatigue damage between the experimental result and the computed result. By using the normal distribution and the mean and sample standard deviation of the
calculated fatigue damage, the range in fatigue damage for a $90 \%$ probability of failure is $\pm 0.074$. For the proposed model to predict failure with a $90 \%$ confidence the calculated fatigue damage should fall between 0.926 and 1.074 or if an upper limit is not used 0.943 or higher.


Figure 31 Comparison of connection types and fatigue damage for the proposed method

Table 7 Fatigue life average and standard deviation for the proposed method

| SWT |  |
| :---: | :---: |
| $\mu$ | 1 |
| $S$ | 0.0448 |
| $s / \mu$ | 0.0448 |

### 5.5 Fatigue failure

Figure 32 shows failures of two screwed connections in valleys due to fatigue. The left image shows the failure with a screw head diameter of $15.2 \mathrm{~mm}(0.6 \mathrm{in}$.) whereas the right image shows the same connection but reinforced with a washer with diameter of 38.1 mm ( 1.5 in .). The failure in the right is dominated by vertical and horizontal cracks extending from the screw hole and the failure in the left is composed of smaller diagonal cracks. Once a crack forms, it will grow in mode I as predicted by previous studies (Socie and Marquis 2000). Under mode I loading a crack grows perpendicular to the major principal strain direction. The major principal strain, which corresponds to the tangential strain in Section 4.6, is always tangent to the screw hole. The difference between the two cases in Figure 32 is at the angle where the tangential strain is maximum. The finite element model shows that this angle is not the same for all 12 configurations meaning that the angle of crack propagation is dependent on the number of screwed connections, the thickness of the cladding, and the type of connection as shown in Figure B. 1 in Appendix B.


Figure 32 Failure of Screwed connections by García, 2008, pp. 65.

Another finding is that loading sublevels that go after higher amplitude levels cause less damage because elastic deformation is less after permanent plastic deformation requiring a larger load in order to reach the yield strength than before. This can be seen in Appendix C, where most of the damage is on the first four loading sublevels. By comparing the fatigue damage between screwed connections in valleys and the same connections reinforced with washers on the same loading sublevel it is found that adding a washer increases the fatigue life of the connection by an average of $18 \%$. This can be seen by comparing the fatigue damage of configuration 5 with 3,17 with 15 , and 18 with 13 on the same loading sublevel in Appendix C.

### 5.6 Proposed Method for the Fatigue Life Prediction of Steel Thin-Walled Folded Plates Using an Analytical and Computational Approach

In this section a straightforward method of fatigue life prediction is proposed based on parametric studies done for 12 different cladding configurations. A step by step procedure is stated in terms of finite element modeling considerations and fatigue theory developed throughout this research.
A. Construct a finite element model of the structure.

1. Use symmetric boundary conditions when appropriate in order to reduce computational time as shown in Figure 3.
2. Define the material as elastic perfectly plastic.
3. Identify critical connections carrying a higher load (center connections in this case).
4. Model critical connections by defining the radius of the screw hole and the radius of the largest of the screw head or washer (if reinforced with washer) as defined in Figure 4.
5. Impose displacement restrictions in the direction of the load at the perimeter of the screw head or washer. The other two directions are not restricted in order to model stress concentration around the hole.
6. Define the load (pressure) acting normal to a single plane perpendicular to an axis passing through the center of the screws. The load magnitude has to be defined for each sublevel of the loading protocol for which the maximum load exceeds the previously applied maximum load starting from the first loading sublevel. As an example consider loading level A in Figure 24. In this case only loading sublevels A1, A2, A3 and A4 have to be defined since A5, A6, A7 and A8 do not exceed the load in sublevel A4. The next sublevel would be B4 since it exceeds the load in A4. A separate analysis will have to be run for each load magnitude considered.
7. Choose an appropriate element type and size and construct a coarse mesh in a region surrounding the critical connections and less dense in the rest of the structure in order to reduce computational time. A quadratic shell element is recomended.
8. Use a nonlinear geometric analysis in order to account for large displacements.
9. Run a separate analysis for each load described in step 6 and determine the location of the highest maximum principal stress around the hole at the critical
connections. Normally this location is at the bottom surface where tension exists near the hole and the same location should be used for each load magnitude. If the location changes by changing the load magnitude then a location is chosen so that the calculated fatigue damage in step 14 is maximum. Ignore high stresses at the perimeter of the screw head or washer since in reality the reaction load is transmitted as a pressure and not as a line load and failure is documented to occur at the edge of the hole as explained in Section 5.4. Record the values of the maximum principal stress and maximum principal strain at this location. Reading the values at the element integration point is recommended.
B. Calculate the fatigue damage.
10. Determine the fatigue parameters $\sigma_{f}^{\prime}$ and $b$ as a function of the ultimate tensile strength from the equations proposed in this research.

$$
\begin{gathered}
\sigma_{f}^{\prime}=1.87 \sigma_{u} \\
b=-0.12
\end{gathered}
$$

11. Determine the strain amplitude $\varepsilon_{a}$ for each loading sublevel.

$$
\begin{array}{ll}
\varepsilon_{a}=\frac{\varepsilon_{\max }}{2}\left(1-\frac{P_{\min }}{P_{s u b \max }}\right) ; & \varepsilon_{\max } \leq \varepsilon_{y} \\
\varepsilon_{a}=\frac{\varepsilon_{y}}{2}\left(1-\frac{P_{\min }}{P_{s u b \max }}\right) ; & \varepsilon_{\max }>\varepsilon_{y}, P_{s u b \max } \geq P_{\max } \\
\varepsilon_{a}=\frac{\varepsilon_{y}}{2} \frac{P_{s u b \max }-P_{\min }}{P_{\max }} \quad ; & \varepsilon_{\max }>\varepsilon_{y}, P_{s u b \max }<P_{\max }
\end{array}
$$

where $\varepsilon_{\text {max }}$ is the maximum principal strain recorded in step $9, P_{\text {max }}$ is the previously applied maximum load starting from the first loading sublevel being zero at the first sublevel as explained in step $6, P_{\text {sub max }}$ is the maximum sublevel load, and $P_{\text {min }}$ is the minimum sublevel load. It is noted that after first yield the strain amplitude is independent of the maximum principal strain and only depends on the loading sequence.
12. Determine the maximum stress $\sigma_{\max }$ for each loading sublevel.

$$
\begin{array}{ll}
\sigma_{\max }=\sigma_{\max p} ; & P_{\text {sub } \max } \geq P_{\max } \\
\sigma_{\max }=\sigma_{\max p r e v} \frac{P_{\text {sub } \max }}{P_{\max }} ; & P_{\text {sub } \max }<P_{\max }
\end{array}
$$

where $\sigma_{\max p}$ is the maximum principal stress recorded in step 9 and $\sigma_{\max p r e v}$ is the maximum previously applied maximum stress starting from the first loading sublevel.
13. Calculate the number of cycles to failure for each loading sublevel by solving for $N_{S W T}$ in equation 19.

$$
\sigma_{\max } \varepsilon_{a}=\frac{\left(\sigma_{f}^{\prime}\right)^{2}}{E}\left(2 N_{S W T}\right)^{2 b}
$$

where $\varepsilon_{a}$ and $\sigma_{\max }$ were found for each loading sublevel in steps 11 and 12
respectively and $\sigma_{f}{ }_{f}$ and $b$ were found in step 10 .
14. Calculate the fatigue damage $F$ by using equation 21.

$$
\sum_{i} \frac{1}{K} \frac{n_{i}}{\left(N_{f}\right)_{i}}=F
$$

where $n$ is the number of cycles at loading sublevel $i, K=0.7$ and $N_{f}=N_{S W T}$ for each loading sublevel. This summation is continued for each loading level until the fatigue damage $F$ exceeds 1 indicating that failure has occurred. Cracks will start at the location determined in step nine and will propagate in mode I (perpendicular to the maximum principal strain direction on the sublevel where $F$ $=1$ ), which in most cases is perpendicular to the screw hole.
15. Go to step 9 and check that the location chosen, which is the same for every loading sublevel, gives the highest fatigue damage $F$. The process may have to be repeated for several locations.

## Chapter 6: Conclusions

### 6.1 Summary of Contributions

In this research the stresses and strains that are responsible for steel cladding failures at the connections have been investigated using finite element analysis. The variable amplitude load associated with wind fluctuations has been reported in Figure 24. Previous studies by García (2008) approached this problem based on full-scale testing under alternating loads. A recent loading protocol established by the Canadian Standards Association has been implemented for rating a specific structure according to the level of wind damage it can withstand.

Structural analysis was used to evaluate stresses under an assumed representation of the loads in the testing procedure. Different fatigue criteria were compared and one was validated in predicting the fatigue damage associated with wind fluctuations.

### 6.2. Main Conclusions Derived from this Research

From the results obtained in this thesis, several conclusions can be stated about the specific findings:

- The low-cycle fatigue damage of steel cladding configurations is found to be caused by elastic deformation and not by plastic deformation according to the Sidgers protocol as shown in Section 5.1. Because for roof claddings pressure is only applied on the outside where the wind suction forces act, the load always acts in the same direction and only tension loading exists $(R=0)$.
- The order in which high amplitude and low amplitude cycles are applied is found to have an effect on the final fatigue damage predicted after initial yielding occurs. The results supporting this conclusion were presented in Section 5.5 of the thesis. If during the low amplitude cycles initial yielding is reached, then applying the low amplitude cycles first and the high amplitude cycles later will result in a higher fatigue damage than if the high amplitude cycles are applied first.
- A fatigue parameter estimation method is developed which correlates the fatigue parameters with the monotonic tensile properties of the material. This method is developed for low cycle fatigue, tension loading $(R=0)$, and to be used with the Smith, Watson, and Topper fatigue model as shown in Section 5.4.
- A method for the fatigue life prediction of steel thin-walled folded plates is developed using an analytical and computational approach. This method, which only requires the monotonic tensile properties of the material, can be used to predict a fatigue failure mechanism for different types of connections, thickness, and configurations as shown in Section 5.6.


### 6.3. Original Contributions of this Thesis

The approach of using analytical methods provides a new insight on the factors that are responsible for fatigue damage and is applicable to a larger selection of configurations and materials that would result too expensive and time consuming to be done by experimental methods. One of the contributions made in this research is to be able to predict the fatigue life of steel claddings under fluctuating wind loads by directly applying fatigue theory using total life approaches. This minimizes the need to perform
full-scale and small-scale testing. In order to perform this analysis a new methodology was developed, specifically the application of distinct fatigue and parameter estimation models.

### 6.4. Recommendations for Further Research

One of the limitations associated with fatigue analytical models is that one model is not applicable to all materials and kinds of loading since the methods presented here were developed specifically for low cycle fatigue and tension loading $(R=0)$. Future work should be aimed at exploring other materials and stress ratios ( $R$ ). An improved finite element model of the connections can be investigated for a wider range of screws, washers and types of connections. Another important field of investigation associated with this research is the quantification of wind loading fluctuations associated with tropical storms and hurricanes, which are the main cause of steel cladding failure in Puerto Rico and the US. Other variables associated with uncertainties in the prediction of fatigue life such as manufacturing process and microstructure can be investigated.

## References

ABAQUS v6.8. Simulia. Unified FEA. Dassault systemes. Warwick, Rhode Island. USA. 2008.

ASTM International. Standard Specification for Steel Sheet, Zinc-Coated (Galvanized) or Zinc-Iron Alloy-Coated (Galvannealed) by the Hot-Dip Process. ASTM-A653-09. American Society for Testing and Materials, West Conshohocken, PA, 2009.

Bannantine, J.A., and Socie, D.F., "Observations of Cracking Behavior in Tension and Torsion Low Cycle Fatigue," ASTM STP 942, American Society for Testing and Materials, West Conshohocken, PA, 1985, pp. 899-921.

Baskaran, A. Molleti, S. and Sexton, M. Wind performance evaluation of fully bonded roofing assemblies. Construction and Building Materials. Vol. 22, pp 1-21. 2006.

Bäumel, A., Seeger, T. Materials Data for Cyclic Loading, Supplement 1,Elsevier, Amsterdam, No. 61, 1990.

Boresi, A. P., Schmidt, R. J., Sidebottom, O. M. Advanced Mechanics of Materials. Fifth ed. John Wiley \& Sons, 1993.

Coffin, L.F., Jr., "A Note on Low Cycle Fatigue Laws,' J. Mater., Vol. 6, No. 2, pp. 388-402, 1971.

Cook, R. D., Malkus, D. S., Plesha, M. E. and Witt, R. J. Concepts and Applications of Finite Element Analysis. John Wiley \& Sons. Fifth ed. 2002.

CSA Number A123.21-04 2004. Standard test method for the dynamic wind uplift resistance of mechanically attached membrane-roofing systems, Canadian Standards Association, 2004.

Dowling, N. E. Estimating Fatigue Life. Fatigue and Fracture. In: ASM Handbook. Vol. 19, pp. 256-262, 1992.

Dowling, N. E. Parameters for Estimating Fatigue Life. Fatigue and Fracture. In: ASM Handbook. Vol. 19, pp. 963-968, 1992.

Dowling, N. E. Mechanical Behavior of Materials: Engineering Methods for Deformation, Fracture, and Fatigue, Prentice Hall, 1993.

García, A. J. Estimación de daños producidos por viento en edificaciones industriales. M. S. Thesis. Universidad de Puerto Rico Recinto Universitario de Mayagüez. 2008.

Goodman, J. Mechanics Applied to Engineering. Longman, Green \& Company. London. 1899.

Hua, C.T., and Socie, D.F., "Fatigue Damage in 1045 Steel Under Variable Amplitude Loading," Fatigue and Fracture Engineering Materials and Structures, Vol. 8, No. 2, pp. 101-104, 1985.

Lee, K. H. and Rosowsky, D. V. Fragility assessment for roof sheathing failure in high wind regions. Engineering Structures. Vol. 27, pp 857-868, 2004.

Mahaarachchi, D. and Mahendran, M. A strain criterion for pull-through failures in crestfixed steel claddings. Engineering Structures. Vol. 31, pp 498-506. 2008.

Mahaarachchi, D. and Mahendran, M. Wind uplift strength of trapezoidal steel cladding with closely spaced ribs. Journal of Wind Engineering and Industrial Aerodynamics. Vol. 97, pp 140-150, 2009.

Mahendran, M. Wind-resistant low-rise buildings in the tropics. ASCE J. of Performance of Constructed Facilities, Vol. 9, pp 330-346, 1995.

Mahendran, M. and Mahaarachchi, D. Splitting failures in trapezoidal steel roof cladding. ASCE Journal of Performance of Constructed Facilities. Vol. 18, pp 4-11, 2004.

Mahendran, M. and Tang, R. B. Pull-Out Strength of Steel Roof and Wall Cladding Systems. ASCE. J. Struct. Engrg. Volume 124, Issue 10, pp. 1192-1201. 1998.

Manson, S.S. "Fatigue: A Complex Subject - Some Simple Approximations," NASA-TM-X-52084, National Aeronautics and Space Administration, 1965.

Matcor. Rolled Formed Metal Building Components. Matcor, Inc. Guaynabo, PR. 2007.
Mitchell, M.R. Fundamentals of Modern Fatigue Analysis for Design. Fatigue and Fracture. In: ASM Handbook. Vol. 19, pp. 227-240, 1992.

Morrow, J. Cyclic Plastic Strain Energy and Fatigue of Metals. Internal Friction, Damping and Cyclic Plasticity. ASTM STP 378. American Society for Testing and Materials. West Conshohocken, PA, pp. 45-78, 1965.
Salmon, C. G., Johnson, J. E., Malhas, F. A. Steel Structures Design and Behavior. Fifth edition, Pearson Prentice Hall, New Jersey, 2009.

Smith, R.N., Watson, P., and Topper, T.H. A Stress-Strain Parameter for the Fatigue of Metals. Journal of Materials. Vol. 5, No. 4. pp. 767-778, 1970.

Socie, D.F., Kurath, P., and Koch, J.L., "A Multiaxial Fatigue Damage Parameter," European Group on Fracture, EGF Publication 3, Mechanical Engineering Publications, London, pp. 535-550, 1989.

Socie, D. F. and Marquis, G. B. Multiaxial Fatigue, Society of Automotive Engineers, Philadelphia, 2000.
$\mathrm{Xu}, \mathrm{Y} . \mathrm{L}$. Determination of wind-induced fatigue damage loading on roof cladding, Journal of Engineering Mechanics, pp 956-963, 1995.

## Appendix

## A. Specifications for Steel Cladding Geometry and Configuration

Table A. 1 Type B wide rib (WR) (Matcor, 2007) cladding specifications

| Deck <br> Gauge | Design <br> Thickness <br> (in.) | Fy | Weight <br> Galv. <br> $(\mathrm{ksi})$ | $\mathrm{I}_{\mathrm{e}}$ <br> min. <br> $\left(\mathrm{in}^{4}\right)$ | $\mathrm{S}_{\mathrm{p}}$ <br> Positive <br> Bending <br> $\left(\mathrm{in}^{3}\right)$ | $\mathrm{S}_{n}$ <br> Negative <br> Bending <br> $\left(\mathrm{in}^{3}\right)$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 22 | 0.0299 | 33 | 1.47 | 0.1400 | 0.1853 | 0.1918 |
| 20 | 0.0359 | 33 | 2.04 | 0.1833 | 0.2287 | 0.2390 |
| 18 | 0.0478 | 33 | 2.70 | 0.2600 | 0.3083 | 0.3160 |

## B. Finite Element Analysis Output Data for the Strain Hardening and Elastic-

Plastic Material Definitions (Abaqus 2008)

Values read at element integration point at bottom surface


Figure B. 1 Abaqus output points on center holes (Figs. 3c-d)

Table B. 1 Config. \#3 strain hardening

| Location: Point A1 in Figure B.1 at column \#5 in Figure 3c |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Load (Mpa) |  |  |  |  |  |
| S, Mises (Mpa) S, Max (Mpa) | E, Max |  |  |  |  |
| A1 | 0.001453 | 227.5 | 259.4 | 0.004049 |  |
| A2 | 0.002905 | 227.5 | 258.8 | 0.010550 |  |
| A3 | 0.004358 | 227.5 | 257.0 | 0.016600 |  |
| A4 | 0.005810 | 231.9 | 256.2 | 0.022080 |  |
| B4 | 0.007263 | 247.5 | 257.1 | 0.028160 |  |
| C4 | 0.008715 | 264.9 | 273.3 | 0.035280 |  |
| D4 | 0.010168 | 282.6 | 291.8 | 0.042580 |  |
|  |  |  |  |  |  |

Table B. 2 Config. \#3 elastic-plastic

| Location: Point A1 in Figure B. 1 at column \#5 in Figure 3c |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
|  | Load (Mpa) | S, Mises (Mpa) | S, Max (Mpa) | E, Max |
| A1 | 0.001453 | 227.5 | 259.4 | 0.004049 |
| A2 | 0.002905 | 227.5 | 258.8 | 0.010550 |
| A3 | 0.004358 | 227.5 | 258.8 | 0.016600 |
| A4 | 0.005810 | 227.5 | 258.8 |  |
| B4 | 0.007263 | 227.5 | 258.8 |  |
| C4 | 0.008715 | 227.5 | 258.8 |  |
| D4 | 0.010168 | 227.5 | 258.8 |  |

Table B. 3 Config. \#4 strain hardening

| Location: Point C1 in Figure B1 at column \#6 in Figure 3c |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
|  | Load (Mpa) | S, Mises (Mpa) | S, Max (Mpa) | E, Max |
| A1 | 0.000726 | 227.5 | 254.6 | 0.002796 |
| A2 | 0.001453 | 227.5 | 252.5 | 0.005641 |
| A3 | 0.002179 | 227.5 | 252.5 | 0.009122 |
| A4 | 0.002905 | 227.5 | 252.3 | 0.016290 |
| B4 | 0.003631 | 238.5 | 255.1 | 0.024430 |
| C4 | 0.004358 | 258.6 | 271.0 | 0.032660 |
| D4 | 0.005084 | 274.8 | 283.2 | 0.039240 |
| E4 | 0.005810 | 289.2 | 296.8 | 0.045020 |

Table B. 4 Config. \#4 elastic-plastic

| Location: Point A1 in Figure B1 at column \#6 in Figure 3c |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
|  | Load (Mpa) | S, Mises (Mpa) | S, Max (Mpa) | E, Max |
| A1 | 0.000726 | 227.5 | 258.9 | 0.003924 |
| A2 | 0.001453 | 227.5 | 257.6 | 0.008394 |
| A3 | 0.002179 | 227.5 | 254.5 | 0.010350 |
| A4 | 0.002905 | 227.5 | 255.4 | 0.011600 |
| B4 | 0.003631 | 227.5 | 255.4 |  |
| C4 | 0.004358 | 227.5 | 255.4 |  |
| D4 | 0.005084 | 227.5 | 255.4 |  |
| E4 | 0.005810 | 227.5 | 255.4 |  |

Table B. 5 Config. \#5 strain hardening

| Location: Point A1 in Figure B1 at column \#5 in Figure 3c |  |  |  |  |  |  |  |  |
| :--- | :--- | :--- | :--- | ---: | :---: | :---: | :---: | :---: |
| Load (Mpa) |  |  |  |  |  | S, Mises (Mpa) | S, Max (Mpa) | E, Max |
| A1 | 0.001453 | 227.6 | 240.6 | 0.001408 |  |  |  |  |
| A2 | 0.002906 | 227.6 | 242.4 | 0.002347 |  |  |  |  |
| A3 | 0.004359 | 227.6 | 244.1 | 0.003285 |  |  |  |  |
| A4 | 0.005812 | 227.6 | 245.8 | 0.004224 |  |  |  |  |
| B4 | 0.007266 | 227.6 | 247.5 | 0.005162 |  |  |  |  |
| C4 | 0.008719 | 227.6 | 249.3 | 0.006101 |  |  |  |  |
| D4 | 0.010172 | 227.6 | 249.5 | 0.019230 |  |  |  |  |
| E4 | 0.011625 | 250.3 | 261.3 | 0.029250 |  |  |  |  |
| F4 | 0.013078 | 275.8 | 280.1 | 0.039710 |  |  |  |  |
| G4 | 0.014531 | 296.0 | 303.4 | 0.047980 |  |  |  |  |
| H4 | 0.015984 | 315.3 | 327.0 | 0.055900 |  |  |  |  |
|  |  |  |  |  |  |  |  |  |

Table B. 6 Config. \#5 elastic-plastic

| Location: Point B1 in Figure B1 at column \#6 in Figure 3c |  |  |  |
| :--- | :--- | :--- | :--- |
| Load (Mpa) S, Mises (Mpa) S, Max (Mpa) |  |  |  |
| A1 | 0.001453 |  | 199.5 |
| A2 | 0.002906 | 227.6 | 262.7 |
| A3 | 0.004359 | 227.6 | 267.7 |
| A4 | 0.005812 | 227.6 | 262.6 |
| B4 | 0.007266 | 227.6 | 262.6 |
| C4 | 0.008719 | 227.6 | 262.6 |
| D4 | 0.010172 | 227.6 | 262.6 |
| E4 | 0.011625 | 227.6 | 262.6 |
| F4 | 0.013078 | 22.6 | 262.6 |
| G4 | 0.014531 | 227.6 | 262.6 |
| H4 | 0.015984 | 227.6 | 262.6 |
|  |  |  |  |

Table B. 7 Config. \#6 strain hardening

| Location: Point A1 in Figure B1 at column \#4 in Figure 3c |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Load (Mpa) |  |  |  |  |  |
| A1 , Mises (Mpa) | S, Max (Mpa) | E, Max |  |  |  |
| A1 | 0.001089 | 227.5 | 254.0 | 0.001834 |  |
| A2 | 0.002179 | 227.5 | 252.8 | 0.006096 |  |
| A3 | 0.003268 | 227.5 | 251.6 | 0.010358 |  |
| A4 | 0.004358 | 227.5 | 250.4 | 0.014620 |  |
| B4 | 0.005447 | 250.4 | 258.8 | 0.029330 |  |
| C4 | 0.006537 | 276.5 | 283.5 | 0.04030 |  |
| D4 | 0.007626 | 296.6 | 309.7 | 0.048210 |  |
| E4 | 0.008716 | 319.0 | 332.9 | 0.057410 |  |
| F4 | 0.009805 | 341.6 | 355.6 | 0.066740 |  |

Table B. 8 Config. \#6 elastic-plastic

| Location: Point B1 in Figure B1 at column \#4 in Figure 3c |  |  |  |  |
| :---: | :---: | :---: | :---: | ---: |
| Load (Mpa) |  |  |  |  |
| S, Mises (Mpa) S, Max (Mpa) | E, Max |  |  |  |
| A1 | 0.001089 | 227.5 | 262.1 | 0.001834 |
| A2 | 0.002179 | 227.5 | 261.4 | 0.006096 |
| A3 | 0.003268 | 227.5 | 260.8 | 0.010358 |
| A4 | 0.004358 | 227.5 | 260.2 | 0.014620 |
| B4 | 0.005447 | 227.5 | 260.2 |  |
| C4 | 0.006537 | 227.5 | 260.2 |  |
| D4 | 0.007626 | 227.5 | 260.2 |  |
| E4 | 0.008716 | 227.5 | 260.2 |  |
| F4 | 0.009805 | 227.5 | 260.2 |  |

Table B. 9 Config. \#13 strain hardening

| Location: Point B1 in Figure B.1 at column \#4 in Figure 3c |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Load (Mpa) |  |  |  |  |  |
| S, Mises (Mpa) | S, Max (Mpa) | E, Max |  |  |  |
| A1 | 0.001089 | 227.5 | 260.8 | 0.002283 |  |
| A2 | 0.002179 | 227.5 | 261.2 | 0.007413 |  |
| A3 | 0.003268 | 227.5 | 260.6 | 0.015480 |  |
| A4 | 0.004358 | 231.6 | 260.0 | 0.021920 |  |
| B4 | 0.005447 | 241.4 | 259.1 | 0.026810 |  |
| C4 | 0.006537 | 250.3 | 265.8 | 0.031250 |  |
| D4 | 0.007626 | 260.2 | 276.0 | 0.036170 |  |
| E4 | 0.008715 | 270.5 | 291.3 | 0.041310 |  |
|  |  |  |  |  |  |

Table B. 10 Config. \#13 elastic-plastic


Table B. 11 Config. \#15 strain hardening

| Location: Point C1 in Figure B.1 at column \#6 in Figure 3c |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Load (Mpa) S, Mises (Mpa) S, Max (Mpa) |  |  |  |  |  |
| A1 | 0.001089 | 227.5 | 258.2 | 0.003757 |  |
| A2 | 0.002179 | 227.5 | 257.4 | 0.007514 |  |
| A3 | 0.003268 | 227.5 | 256.6 | 0.011271 |  |
| A4 | 0.004358 | 235.0 | 255.8 | 0.023640 |  |
| B4 | 0.005447 | 255.7 | 268.6 | 0.033980 |  |
| C4 | 0.006537 | 273.7 | 285.0 | 0.042970 |  |
|  |  |  |  |  |  |

Table B. 12 Config. \#15 elastic-plastic

| Location: Point B1 in Figure B.1 at column \#4 in Figure 3c |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Load (Mpa) |  |  |  |  |  |
| S, Mises (Mpa) S, Max (Mpa) | E, Max |  |  |  |  |
| A1 | 0.001089 | 227.5 | 262.6 | 0.003267 |  |
| A2 | 0.002179 | 227.5 | 262.6 | 0.018200 |  |
| A3 | 0.003268 | 227.5 | 262.6 | 0.033133 |  |
| A4 | 0.004358 | 227.5 | 262.6 | 0.048066 |  |
| B4 | 0.005447 | 227.5 | 262.6 | 0.062999 |  |
| C4 | 0.006537 | 227.5 | 262.6 | 0.018200 |  |
|  |  |  |  |  |  |

Table B. 13 Config. \#17 strain hardening

| Location: Point B1 in Figure B.1 at column \#4 in Figure 3c |  |  |  |  |  |
| :---: | :---: | :---: | :---: | ---: | :---: |
| Load (Mpa) S, Mises (Mpa) S, Max (Mpa) |  |  |  |  |  |
| E, Max |  |  |  |  |  |
| A1 | 0.001089 | 227.5 | 247.4 | 0.001428 |  |
| A2 | 0.002179 | 227.5 | 248.0 | 0.003973 |  |
| A3 | 0.003268 | 227.5 | 248.7 | 0.006519 |  |
| A4 | 0.004358 | 227.5 | 249.3 | 0.009064 |  |
| B4 | 0.005447 | 227.5 | 249.9 | 0.011609 |  |
| C4 | 0.006537 | 227.5 | 250.5 | 0.014155 |  |
| D4 | 0.007626 | 227.5 | 251.1 | 0.016700 |  |
| E4 | 0.008716 | 241.8 | 252.9 | 0.025800 |  |
| F4 | 0.009805 | 259.4 | 264.1 | 0.033010 |  |
| G4 | 0.010894 | 274.5 | 280.6 | 0.039160 |  |
| H4 | 0.011984 | 287.9 | 298.1 | 0.044660 |  |
| 14 | 0.013073 | 300.6 | 313.0 | 0.04989 |  |

Table B. 14 Config. \#17 elastic-plastic

| Location: Point B1 in Figure B.1 at column \#4 in Figure 3c |  |  |  |  |  |
| :---: | :---: | :---: | :---: | ---: | :---: |
| Load (Mpa) |  |  |  |  |  |
| S, Mises (Mpa) S, Max (Mpa) | E, Max |  |  |  |  |
| A1 | 0.001089 | 227.5 | 247.4 | 0.001428 |  |
| A2 | 0.002179 | 227.5 | 248.0 | 0.003973 |  |
| A3 | 0.003268 | 227.5 | 248.7 | 0.006519 |  |
| A4 | 0.004358 | 227.5 | 249.3 | 0.009064 |  |
| B4 | 0.005447 | 227.5 | 249.9 | 0.011609 |  |
| C4 | 0.006537 | 227.5 | 250.5 | 0.014155 |  |
| D4 | 0.007626 | 227.5 | 251.1 | 0.016700 |  |
| E4 | 0.008716 | 227.5 | 251.1 |  |  |
| F4 | 0.009805 | 227.5 | 251.1 |  |  |
| G4 | 0.010894 | 227.5 | 251.1 |  |  |
| H4 | 0.011984 | 227.5 | 251.1 |  |  |
| 14 | 0.013073 | 227.5 | 251.1 |  |  |

Table B. 15 Config. \#18 strain hardening

| Location: Point B1 in Figure B.1 at column \#4 in Figure 3c |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Load (Mpa) S, Mises (Mpa) S, Max (Mpa) |  |  |  |  |  |
| E, Max |  |  |  |  |  |
| A1 | 0.001816 | 227.5 | 252.6 | 0.001540 |  |
| A2 | 0.003631 | 227.5 | 252.6 | 0.003680 |  |
| A3 | 0.005447 | 227.5 | 252.5 | 0.005820 |  |
| A4 | 0.007263 | 227.5 | 252.5 | 0.007960 |  |
| B4 | 0.009079 | 227.5 | 252.4 | 0.010100 |  |
| C4 | 0.010894 | 227.5 | 253.0 | 0.015850 |  |
| D4 | 0.012710 | 241.0 | 252.0 | 0.025480 |  |
| E4 | 0.014526 | 261.2 | 263.7 | 0.033720 |  |
| F4 | 0.016342 | 279.5 | 286.0 | 0.041240 |  |
| G4 | 0.018157 | 295.6 | 306.2 | 0.047840 |  |

Table B. 16 Config. \#18 elastic-plastic

| Location: Point B1 in Figure B.1 at column \#4 in Figure 3c |  |  |  |  |
| :---: | :---: | :---: | :---: | ---: |
| Load (Mpa) S, Mises (Mpa) S, Max (Mpa) |  |  |  |  |
| E, Max |  |  |  |  |
| A1 | 0.001816 | 227.5 | 252.6 | 0.001540 |
| A2 | 0.003631 | 227.5 | 252.6 | 0.003680 |
| A3 | 0.005447 | 227.5 | 252.5 | 0.005820 |
| A4 | 0.007263 | 227.5 | 252.5 | 0.007960 |
| B4 | 0.009079 | 227.5 | 252.4 | 0.010100 |
| C4 | 0.010894 | 227.5 | 252.4 | 0.015850 |
| D4 | 0.012710 | 241.0 | 252.4 |  |
| E4 | 0.014526 | 261.2 | 252.4 |  |
| F4 | 0.016342 | 279.5 | 252.4 |  |
| G4 | 0.018157 | 295.6 | 252.4 |  |

Table B. 17 Config. \#19 strain hardening

| Location: Point B1 in Figure B. 1 at column \#5 in Figure 3d |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
|  | Load (Mpa) | S, Mises (Mpa) | S, Max (Mpa) | E, Max |
| A1 | 0.000726 | 218.9 | 226.9 | 0.001110 |
| A2 | 0.001453 | 227.5 | 259.3 | 0.002451 |
| A3 | 0.002179 | 227.5 | 259.0 | 0.004311 |
| A4 | 0.002905 | 227.5 | 259.2 | 0.008580 |
| B4 | 0.003631 | 227.5 | 259.3 | 0.014070 |
| C4 | 0.004358 | 234.3 | 257.4 | 0.022710 |
| D4 | 0.005084 | 250.9 | 270.1 | 0.029500 |
| E4 | 0.005810 | 267.5 | 282.8 | 0.036290 |
| F4 | 0.006537 | 284.0 | 295.5 | 0.043080 |
| G4 | 0.007263 | 300.6 | 308.1 | 0.049860 |
| H4 | 0.007989 | 307.6 | 316.1 | 0.052710 |
| 14 | 0.008716 | 314.5 | 324.1 | 0.055550 |
| J4 | 0.009442 | 321.5 | 332.1 | 0.058390 |
| K4 | 0.010168 | 328.3 | 340.1 | 0.061240 |

Table B. 18 Config. \#19 elastic-plastic

| Location: Point B1 in Figure B.1 at column \#5 in Figure 3d |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
| Load (Mpa) |  |  |  |  |
| S, Mises (Mpa) | S, Max (Mpa) | E, Max |  |  |
| A1 | 0.000726 | 218.9 | 226.9 | 0.001110 |
| A2 | 0.001453 | 227.5 | 259.3 | 0.002451 |
| A3 | 0.002179 | 227.5 | 259.0 | 0.004311 |
| A4 | 0.002905 | 227.5 | 259.2 | 0.008580 |
| B4 | 0.003631 | 227.5 | 259.3 | 0.014070 |
| C4 | 0.004358 | 227.5 | 257.2 |  |
| D4 | 0.005084 | 227.5 | 255.0 |  |
| E4 | 0.005810 | 227.5 | 252.9 |  |
| F4 | 0.006537 | 227.5 | 250.7 |  |
| G4 | 0.007263 | 227.5 | 250.1 |  |
| H4 | 0.007989 | 227.5 | 249.5 |  |
| 14 | 0.008716 | 227.5 | 248.9 |  |
| J4 | 0.009442 | 227.5 | 248.3 |  |
| K4 | 0.010168 | 227.5 | 247.7 |  |

Table B19 Config. \#20 strain hardening

| Location: Point B1 in Figure B.1 at column \#3 in Figure 3d |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Load (Mpa) |  |  |  |  |  | S, Mises (Mpa) | S, Max (Mpa) | E, Max |
| A1 | 0.000872 | 227.5 | 257.8 | 0.001847 |  |  |  |  |
| A2 | 0.001743 | 227.5 | 259.8 | 0.006436 |  |  |  |  |
| A3 | 0.002615 | 227.5 | 257.8 | 0.014860 |  |  |  |  |
| A4 | 0.003486 | 234.9 | 257.0 | 0.023600 |  |  |  |  |
| B4 | 0.004358 | 243.8 | 258.6 | 0.028050 |  |  |  |  |
| C4 | 0.005229 | 245.8 | 261.2 | 0.029060 |  |  |  |  |

Table B20 Config. \#20 elastic-plastic

| Location: Point B1 in Figure B.1 at column \#3 in Figure 3d |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
| Load (Mpa) |  |  |  |  |
| A1, Mises (Mpa) | S, Max (Mpa) | E, Max |  |  |
| A2 | 0.000872 | 227.5 | 257.8 | 0.001847 |
| A3 | 0.001743 | 227.5 | 259.8 | 0.006436 |
| A4 | 0.002615 | 227.5 | 257.8 | 0.014860 |
| B4 | 0.003486 | 227.5 | 257.0 |  |
| C4 | 0.004358 | 227.5 | 257.0 |  |
|  |  |  | 257.0 |  |

Table B. 21 Config. \#21 strain hardening

| Location: Point D1 in Figure B.1 at column \#5 in Figure 3d |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | ---: | :---: | :---: |
| Load (Mpa) S, Mises (Mpa) S, Max (Mpa) |  |  |  |  |  | E, Max |
| A1 | 0.000872 | 227.5 | 262.7 | 0.002666 |  |  |
| A2 | 0.001743 | 227.5 | 262.7 | 0.007885 |  |  |
| A3 | 0.002615 | 227.5 | 262.7 | 0.013300 |  |  |
| A4 | 0.003486 | 230.0 | 262.9 | 0.019090 |  |  |
| B4 | 0.004358 | 236.1 | 269.3 | 0.021410 |  |  |
| C4 | 0.005229 | 235.9 | 269.2 | 0.021340 |  |  |
|  |  |  |  |  |  |  |

Table B. 22 Config. \#21 elastic-plastic

| Location: Point D1 in Figure B. 1 at column \#5 in Figure 3d |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
| Load (Mpa) |  |  |  |  |
| S, Mises (Mpa) S, Max (Mpa) | E, Max |  |  |  |
| A1 | 0.000872 | 227.5 | 262.7 | 0.002666 |
| A2 | 0.001743 | 227.5 | 262.7 | 0.007885 |
| A3 | 0.002615 | 227.5 | 262.7 | 0.013300 |
| A4 | 0.003486 | 227.5 | 262.7 | 0.019090 |
| B4 | 0.004358 | 227.5 | 262.7 |  |
| C4 | 0.005229 | 227.5 | 262.7 |  |
|  |  |  |  |  |

Table B. 23 Config. \#24 strain hardening

| Location: Point B1 in Figure B. 1 at column \#5 in Figure 3d |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
|  | Load (Mpa) | S, Mises (Mpa) | S, Max (Mpa) | E, Max |
| A1 | 0.000872 | 227.5 | 235.9 | 0.001230 |
| A2 | 0.001743 | 227.5 | 258.3 | 0.003021 |
| A3 | 0.002615 | 227.5 | 257.8 | 0.006977 |
| A4 | 0.003486 | 227.5 | 257.9 | 0.014200 |
| B4 | 0.004358 | 245.7 | 259.2 | 0.027390 |
| C4 | 0.005229 | 278.0 | 288.6 | 0.040640 |
| D4 | 0.006101 | 300.7 | 307.5 | 0.049930 |
| E4 | 0.006972 | 317.1 | 326.7 | 0.056610 |
| F4 | 0.007844 | 332.3 | 343.5 | 0.062910 |
| G4 | 0.008716 | 346.7 | 358.9 | 0.068810 |

Table B. 24 Config. \#24 elastic-plastic

| Location: Point B1 in Figure B.1 at column \#5 in Figure 3d |  |  |  |  |  |
| :--- | :--- | :--- | :--- | :--- | :---: |
| Load (Mpa) S, Mises (Mpa) S, Max (Mpa) |  |  |  |  |  |
| E, Max |  |  |  |  |  |
| A1 | 0.000872 |  | 209.4 | 0.001230 |  |
| A2 | 0.001743 | 227.5 | 262.6 | 0.003021 |  |
| A3 | 0.002615 | 227.5 | 262.4 | 0.006977 |  |
| A4 | 0.003486 | 227.5 | 262.3 | 0.013940 |  |
| B4 | 0.004358 | 227.5 | 262.2 |  |  |
| C4 | 0.005229 | 227.5 | 262.2 |  |  |
| D4 | 0.006101 | 227.5 | 262.2 |  |  |
| E4 | 0.006972 | 227.5 | 262.1 |  |  |
| F4 | 0.007844 | 227.5 | 262.1 |  |  |
| G4 | 0.008716 | 227.5 | 262.1 |  |  |

## C. Fatigue Damage Results, Elastic-Plastic

| Table C.1 Config. \#3 |  |  |  |  |  |  |  |  |  |  |
| :--- | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
|  | $\Delta \varepsilon_{\max }$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{S W T}$ | $F_{G}$ |
|  | $\times 10^{-3}$ |  |  |  |  |  |  |  |  |  |
| A1 | 1.138 | 129.7 | 259.4 | 2000 | 19182 | 14576 | 10065 | 0.149 | 0.196 | 0.284 |
| A2 | 1.138 | 129.4 | 258.8 | 3500 | 19652 | 14715 | 10065 | 0.403 | 0.536 | 0.781 |
| A3 | 1.138 | 129.4 | 258.8 | 1000 | 19652 | 14715 | 10065 | 0.476 | 0.633 | 0.923 |
| A4 | 1.138 | 129.4 | 258.8 | 250 | 19652 | 14715 | 10065 | 0.494 | 0.657 | 0.958 |
| A5 | 0.285 | 97.1 | 129.4 | 2000 | 3599611629 | 85235994 | $\#$ | 0.494 | 0.657 | 0.958 |
| A6 | 0.569 | 129.4 | 194.1 | 2000 | 6338445 | 876229 | 3246250 | 0.495 | 0.660 | 0.959 |
| A7 | 0.854 | 161.8 | 258.8 | 125 | 117751 | 48789 | 30063 | 0.496 | 0.664 | 0.965 |
| A8 | 0.569 | 194.1 | 258.8 | 125 | 1795078 | 264265 | 188072 | 0.496 | 0.665 | 0.966 |
| B2 | 0.717 | 80.9 | 161.8 | 2500 | 2274833 | 715038 | 2599950 | 0.498 | 0.670 | 0.967 |
| B3 | 1.070 | 121.3 | 242.6 | 750 | 38902 | 24917 | 23176 | 0.525 | 0.713 | 1.013 |
| B4 | 1.138 | 129.4 | 258.8 | 250 | 19652 | 14715 | 10065 | 0.544 | 0.737 | 1.049 |
| B6 | 0.569 | 129.4 | 194.1 | 1750 | 6338445 | 876229 | 3246250 | 0.544 | 0.740 | 1.050 |
| B7 | 0.854 | 161.8 | 258.8 | 125 | 117751 | 48789 | 30063 | 0.546 | 0.744 | 1.056 |
| B8 | 0.569 | 194.1 | 258.8 | 125 | 1795078 | 264265 | 188072 | 0.546 | 0.744 | 1.057 |
| C2 | 0.683 | 77.6 | 155.3 | 1250 | 3369887 | 1038694 | 4031757 | 0.546 | 0.746 | 1.057 |
| C3 | 1.029 | 116.5 | 232.9 | 750 | 59566 | 34756 | 38710 | 0.564 | 0.777 | 1.085 |
| C4 | 1.138 | 129.4 | 258.8 | 250 | 19652 | 14715 | 10065 | 0.582 | 0.801 | 1.120 |
| C6 | 0.569 | 129.4 | 194.1 | 1500 | 6338445 | 876229 | 3246250 | 0.583 | 0.804 | 1.121 |
| C7 | 0.854 | 161.8 | 258.8 | 125 | 117751 | 48789 | 30063 | 0.584 | 0.807 | 1.127 |
| C8 | 0.569 | 194.1 | 258.8 | 125 | 1795078 | 264265 | 188072 | 0.584 | 0.808 | 1.128 |
| D2 | 0.668 | 75.5 | 151.0 | 1250 | 4415806 | 1283022 | 5444514 | 0.585 | 0.809 | 1.128 |
| D3 | 0.994 | 113.2 | 226.5 | 500 | 79725 | 45130 | 54824 | 0.594 | 0.825 | 1.141 |
| D4 | 1.138 | 129.4 | 258.8 | 250 | 19652 | 14715 | 10065 | 0.612 | 0.849 | 1.177 |
|  |  |  |  |  |  |  |  |  |  |  |

D4 - first failure observed (García 2008)

Tables C. 1 through C. 12 were calculated using the procedure in Section 5.6 where Column one is the strain range as defined in Section 5.6, columns two and three are the mean and maximum stresses as defined in Section 5.6, Column four is the number of cycles on each loading sublevel, columns five, six, and seven are calculated from equations 13,16 and 19, and columns eight, nine and ten are calculated from equation 21.

| Table C. 2 Config. \#4 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{aligned} & \Delta \mathcal{E}_{\text {max }} \\ & \times 10^{-3} \end{aligned}$ | $\sigma_{m}$ | $\overline{\sigma_{\text {max }}}$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| A1 | 1.138 | 129.5 | 258.9 | 2000 | 19571 | 14691 | 10066 | 0.146 | 0.194 | 0.284 |
| A2 | 1.138 | 128.8 | 257.6 | 3500 | 20651 | 15002 | 10066 | 0.388 | 0.528 | 0.781 |
| A3 | 1.138 | 127.3 | 254.5 | 1000 | 23491 | 15779 | 10066 | 0.449 | 0.618 | 0.923 |
| A4 | 1.138 | 127.3 | 254.5 | 250 | 23491 | 15779 | 10066 | 0.464 | 0.641 | 0.958 |
| A5 | 0.285 | 95.4 | 127.3 | 2000 | 4253180041 | 91399058 | \# | 0.464 | 0.641 | 0.958 |
| A6 | 0.569 | 127.3 | 190.9 | 2000 | 7576637 | 939586 | 3246626 | 0.465 | 0.644 | 0.959 |
| A7 | 0.854 | 159.1 | 254.5 | 125 | 142642 | 52317 | 30067 | 0.466 | 0.647 | 0.965 |
| A8 | 0.569 | 190.9 | 254.5 | 125 | 2208448 | 283373 | 188103 | 0.466 | 0.648 | 0.966 |
| B2 | 0.717 | 79.5 | 159.1 | 2500 | 2673792 | 766739 | 2600197 | 0.467 | 0.653 | 0.967 |
| B3 | 1.070 | 119.3 | 238.6 | 750 | 46360 | 26719 | 23178 | 0.490 | 0.693 | 1.013 |
| B4 | 1.138 | 127.3 | 254.5 | 250 | 23491 | 15779 | 10066 | 0.505 | 0.715 | 1.049 |
| B6 | 0.569 | 127.3 | 190.9 | 1750 | 7576637 | 939586 | 3246626 | 0.506 | 0.718 | 1.050 |
| B7 | 0.854 | 159.1 | 254.5 | 125 | 142642 | 52317 | 30067 | 0.507 | 0.722 | 1.056 |
| B8 | 0.569 | 190.9 | 254.5 | 125 | 2208448 | 283373 | 188103 | 0.507 | 0.722 | 1.056 |
| C2 | 0.683 | 76.4 | 152.7 | 1250 | 3956894 | 1113798 | 4032136 | 0.508 | 0.724 | 1.057 |
| C3 | 1.029 | 114.5 | 229.1 | 750 | 70859 | 37270 | 38715 | 0.523 | 0.753 | 1.085 |
| C4 | 1.138 | 127.3 | 254.5 | 250 | 23491 | 15779 | 10066 | 0.538 | 0.775 | 1.120 |
| C6 | 0.569 | 127.3 | 190.9 | 1500 | 7576637 | 939586 | 3246626 | 0.538 | 0.777 | 1.121 |
| C7 | 0.854 | 159.1 | 254.5 | 125 | 142642 | 52317 | 30067 | 0.539 | 0.781 | 1.127 |
| C8 | 0.569 | 190.9 | 254.5 | 125 | 2208448 | 283373 | 188103 | 0.540 | 0.781 | 1.128 |
| D2 | 0.668 | 74.2 | 148.4 | 1250 | 5181542 | 1375792 | 5445022 | 0.540 | 0.783 | 1.128 |
| D3 | 0.994 | 111.3 | 222.7 | 500 | 94730 | 48393 | 54830 | 0.547 | 0.798 | 1.141 |
| D4 | 1.138 | 127.3 | 254.5 | 250 | 23491 | 15779 | 10066 | 0.563 | 0.820 | 1.176 |
| D6 | 0.569 | 127.3 | 190.9 | 250 | 7576637 | 939586 | 3246626 | 0.563 | 0.821 | 1.177 |
| D7 | 0.854 | 159.1 | 254.5 | 125 | 142642 | 52317 | 30067 | 0.564 | 0.824 | 1.183 |
| D8 | 0.569 | 190.9 | 254.5 | 125 | 2208448 | 283373 | 188103 | 0.564 | 0.825 | 1.183 |
| E2 | 0.650 | 72.7 | 145.4 | 1000 | 6304752 | 1672917 | 6770673 | 0.564 | 0.825 | 1.184 |
| E3 | 0.975 | 109.1 | 218.1 | 500 | 117078 | 57023 | 70572 | 0.570 | 0.838 | 1.194 |
| E4 | 1.138 | 127.3 | 254.5 | 250 | 23491 | 15779 | 10066 | 0.586 | 0.861 | 1.229 |
| E7 | 0.854 | 159.1 | 254.5 | 125 | 142642 | 52317 | 30067 | 0.587 | 0.864 | 1.235 |
| E8 | 0.569 | 190.9 | 254.5 | 125 | 2208448 | 283373 | 188103 | 0.587 | 0.865 | 1.236 |


| Cont. Table C. 2 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{gathered} \Delta \varepsilon_{\max } \\ \times 10^{-3} \end{gathered}$ | $\sigma_{m}$ | $\sigma_{\text {max }}$ | $n$ | $N_{M}$ | $N_{\text {SWT }}$ | $N_{G}$ | $F_{M}$ | $F_{S W T}$ | $F_{G}$ |
| F2 | 0.643 | 61.6 | 123.2 | 1000 | 29872449 | 3494340 | 37401242 | 0.587 | 0.865 | 1.236 |
| *F3 | 0.962 | 92.2 | 184.3 | 500 | 635401 | 122076 | 508814 | 0.588 | 0.871 | 1.238 |

* first failure observed (García 2008)

| Table C. 3 Config. \#5 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{array}{r} \Delta \varepsilon_{\max } \\ \times 10^{-3} \\ \hline \end{array}$ |  | $\sigma_{\text {max }}$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| A1 | 1.138 | 99.8 | 199.5 | 2000 | 289104 | 43518 | 10066 | 0.010 | 0.066 | 0.284 |
| A2 | 1.138 | 131.4 | 262.7 | 3500 | 16746 | 13826 | 10066 | 0.308 | 0.427 | 0.781 |
| A3 | 1.138 | 131.4 | 262.7 | 1000 | 16746 | 13826 | 10066 | 0.394 | 0.531 | 0.923 |
| A4 | 1.138 | 131.3 | 262.6 | 250 | 16814 | 13847 | 10066 | 0.415 | 0.556 | 0.958 |
| A5 | 0.285 | 98.5 | 131.3 | 2000 | 3111884196 | 80213258 | \# | 0.415 | 0.556 | 0.958 |
| A6 | 0.569 | 131.3 | 197.0 | 2000 | 5423343 | 824595 | 3246626 | 0.416 | 0.560 | 0.959 |
| A7 | 0.854 | 164.1 | 262.6 | 125 | 99559 | 45914 | 30067 | 0.417 | 0.564 | 0.965 |
| A8 | 0.569 | 197.0 | 262.6 | 125 | 1496849 | 248693 | 188103 | 0.417 | 0.565 | 0.966 |
| B2 | 0.717 | 82.1 | 164.1 | 2500 | 1975803 | 672903 | 2600197 | 0.419 | 0.570 | 0.967 |
| B3 | 1.070 | 123.1 | 246.2 | 750 | 33376 | 23449 | 23178 | 0.451 | 0.616 | 1.013 |
| B4 | 1.138 | 131.3 | 262.6 | 250 | 16814 | 13847 | 10066 | 0.473 | 0.641 | 1.049 |
| B6 | 0.569 | 131.3 | 197.0 | 1750 | 5423343 | 824595 | 3246626 | 0.473 | 0.644 | 1.050 |
| B7 | 0.854 | 164.1 | 262.6 | 125 | 99559 | 45914 | 30067 | 0.475 | 0.648 | 1.056 |
| B8 | 0.569 | 197.0 | 262.6 | 125 | 1496849 | 248693 | 188103 | 0.475 | 0.649 | 1.056 |
| C2 | 0.683 | 78.8 | 157.6 | 1250 | 2929538 | 977487 | 4032136 | 0.476 | 0.651 | 1.057 |
| C3 | 1.029 | 118.2 | 236.3 | 750 | 51186 | 32708 | 38715 | 0.497 | 0.684 | 1.085 |
| C4 | 1.138 | 131.3 | 262.6 | 250 | 16814 | 13847 | 10066 | 0.518 | 0.709 | 1.120 |
| C6 | 0.569 | 131.3 | 197.0 | 1500 | 5423343 | 824595 | 3246626 | 0.518 | 0.712 | 1.121 |
| C7 | 0.854 | 164.1 | 262.6 | 125 | 99559 | 45914 | 30067 | 0.520 | 0.716 | 1.127 |
| C8 | 0.569 | 197.0 | 262.6 | 125 | 1496849 | 248693 | 188103 | 0.520 | 0.717 | 1.128 |
| D2 | 0.668 | 76.6 | 153.2 | 1250 | 3841063 | 1207416 | 5445022 | 0.521 | 0.718 | 1.128 |
| D3 | 0.994 | 114.9 | 229.8 | 500 | 68580 | 42471 | 54830 | 0.531 | 0.735 | 1.141 |
| D4 | 1.138 | 131.3 | 262.6 | 250 | 16814 | 13847 | 10066 | 0.552 | 0.761 | 1.176 |
| D6 | 0.569 | 131.3 | 197.0 | 250 | 5423343 | 824595 | 3246626 | 0.552 | 0.761 | 1.177 |
| D7 | 0.854 | 164.1 | 262.6 | 125 | 99559 | 45914 | 30067 | 0.554 | 0.765 | 1.183 |
| D8 | 0.569 | 197.0 | 262.6 | 125 | 1496849 | 248693 | 188103 | 0.554 | 0.766 | 1.183 |
| E2 | 0.650 | 75.0 | 150.0 | 1000 | 4677868 | 1468178 | 6770673 | 0.554 | 0.767 | 1.184 |
| E3 | 0.975 | 112.5 | 225.1 | 500 | 84891 | 50044 | 70572 | 0.563 | 0.781 | 1.194 |
| E4 | 1.138 | 131.3 | 262.6 | 250 | 16814 | 13847 | 10066 | 0.584 | 0.807 | 1.229 |
| E7 | 0.854 | 164.1 | 262.6 | 125 | 99559 | 45914 | 30067 | 0.586 | 0.811 | 1.235 |
| E8 | 0.569 | 197.0 | 262.6 | 125 | 1496849 | 248693 | 188103 | 0.586 | 0.811 | 1.236 |


| Cont. Table C. 3 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{aligned} & \Delta \varepsilon_{\max } \\ & \times 10^{-3} \end{aligned}$ | $\sigma_{m}$ | $\sigma_{\text {max }}$ | $n$ | $N_{M}$ | $N_{\text {SWT }}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| F2 | 0.643 | 63.6 | 127.2 | 1000 | 22305887 | 3066688 | 37401242 | 0.586 | 0.812 | 1.236 |
| F3 | 0.962 | 95.1 | 190.2 | 500 | 465873 | 107136 | 508814 | 0.588 | 0.818 | 1.238 |
| F4 | 1.138 | 131.3 | 262.6 | 250 | 16814 | 13847 | 10066 | 0.609 | 0.844 | 1.273 |
| F7 | 0.854 | 164.1 | 262.6 | 125 | 99559 | 45914 | 30067 | 0.611 | 0.848 | 1.279 |
| F8 | 0.569 | 197.0 | 262.6 | 125 | 1496849 | 248693 | 188103 | 0.611 | 0.849 | 1.280 |
| G2 | 0.632 | 52.8 | 105.7 | 1000 | 123637216 | 7119365 | \# | 0.611 | 0.849 | 1.280 |
| G3 | 0.951 | 79.5 | 158.9 | 500 | 2698186 | 237343 | 3679806 | 0.611 | 0.852 | 1.280 |
| G4 | 1.138 | 131.3 | 262.6 | 250 | 16814 | 13847 | 10066 | 0.632 | 0.878 | 1.316 |
| G7 | 0.854 | 163.8 | 262.1 | 125 | 101773 | 46280 | 30067 | 0.634 | 0.882 | 1.322 |
| G8 | 0.569 | 196.6 | 262.1 | 125 | 1532948 | 250675 | 188103 | 0.634 | 0.882 | 1.323 |
| H2 | 0.628 | 43.9 | 87.7 | 1000 | 669166868 | 15877453 | \# | 0.634 | 0.883 | 1.323 |
| H3 | 0.938 | 65.5 | 130.9 | 500 | 16962434 | 563482 | 27776585 | 0.634 | 0.884 | 1.323 |
| H4 | 1.138 | 131.3 | 262.6 | 250 | 16814 | 13847 | 10066 | 0.655 | 0.910 | 1.358 |
| H7 | 0.854 | 163.8 | 262.1 | 125 | 101773 | 46280 | 30067 | 0.657 | 0.913 | 1.364 |
| H8 | 0.569 | 196.6 | 262.1 | 125 | 1532948 | 250675 | 188103 | 0.657 | 0.914 | 1.365 |

H - first failure observed (García 2008)

| Table C. 4 Config. \#6 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{array}{r} \Delta \varepsilon_{\max } \\ \times 10^{-3} \\ \hline \end{array}$ |  | $\sigma_{\text {max }}$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| A1 | 1.138 | 131.1 | 262.1 | 2000 | 17161 | 13958 | 10066 | 0.166 | 0.205 | 0.284 |
| A2 | 1.138 | 130.7 | 261.4 | 3500 | 17660 | 14114 | 10066 | 0.450 | 0.559 | 0.781 |
| A3 | 1.138 | 130.4 | 260.8 | 1000 | 18100 | 14250 | 10066 | 0.529 | 0.659 | 0.923 |
| A4 | 1.138 | 130.1 | 260.2 | 250 | 18551 | 14388 | 10066 | 0.548 | 0.684 | 0.958 |
| A5 | 0.285 | 97.6 | 130.1 | 2000 | 3410935578 | 83341334 | \# | 0.548 | 0.684 | 0.958 |
| A6 | 0.569 | 130.1 | 195.2 | 2000 | 5983461 | 856752 | 3246626 | 0.548 | 0.687 | 0.959 |
| A7 | 0.854 | 162.6 | 260.2 | 125 | 110672 | 47705 | 30067 | 0.550 | 0.691 | 0.965 |
| A8 | 0.569 | 195.2 | 260.2 | 125 | 1678593 | 258391 | 188103 | 0.550 | 0.692 | 0.966 |
| B2 | 0.717 | 81.3 | 162.6 | 2500 | 2159299 | 699144 | 2600197 | 0.552 | 0.697 | 0.967 |
| B3 | 1.070 | 122.0 | 243.9 | 750 | 36760 | 24363 | 23178 | 0.581 | 0.741 | 1.013 |
| B4 | 1.138 | 130.1 | 260.2 | 250 | 18551 | 14388 | 10066 | 0.600 | 0.766 | 1.049 |
| B6 | 0.569 | 130.1 | 195.2 | 1750 | 5983461 | 856752 | 3246626 | 0.600 | 0.769 | 1.050 |
| B7 | 0.854 | 162.6 | 260.2 | 125 | 110672 | 47705 | 30067 | 0.602 | 0.772 | 1.056 |
| B8 | 0.569 | 195.2 | 260.2 | 125 | 1678593 | 258391 | 188103 | 0.602 | 0.773 | 1.056 |
| C2 | 0.683 | 78.1 | 156.1 | 1250 | 3199793 | 1015606 | 4032136 | 0.603 | 0.775 | 1.057 |
| C3 | 1.029 | 117.1 | 234.2 | 750 | 56319 | 33984 | 38715 | 0.622 | 0.806 | 1.085 |
| C4 | 1.138 | 130.1 | 260.2 | 250 | 18551 | 14388 | 10066 | 0.641 | 0.831 | 1.120 |
| C6 | 0.569 | 130.1 | 195.2 | 1500 | 5983461 | 856752 | 3246626 | 0.641 | 0.834 | 1.121 |
| C7 | 0.854 | 162.6 | 260.2 | 125 | 110672 | 47705 | 30067 | 0.643 | 0.837 | 1.127 |
| C8 | 0.569 | 195.2 | 260.2 | 125 | 1678593 | 258391 | 188103 | 0.643 | 0.838 | 1.128 |
| D2 | 0.668 | 75.9 | 151.8 | 1250 | 4193835 | 1254502 | 5445022 | 0.644 | 0.840 | 1.128 |
| D3 | 0.994 | 113.8 | 227.7 | 500 | 75408 | 44127 | 54830 | 0.653 | 0.856 | 1.141 |
| D4 | 1.138 | 130.1 | 260.2 | 250 | 18551 | 14388 | 10066 | 0.672 | 0.881 | 1.176 |
| D6 | 0.569 | 130.1 | 195.2 | 250 | 5983461 | 856752 | 3246626 | 0.672 | 0.881 | 1.177 |
| D7 | 0.854 | 162.6 | 260.2 | 125 | 110672 | 47705 | 30067 | 0.674 | 0.885 | 1.183 |
| D8 | 0.569 | 195.2 | 260.2 | 125 | 1678593 | 258391 | 188103 | 0.674 | 0.885 | 1.183 |
| E2 | 0.650 | 74.3 | 148.7 | 1000 | 5106139 | 1525433 | 6770673 | 0.674 | 0.886 | 1.184 |
| E3 | 0.975 | 111.5 | 223.0 | 500 | 93300 | 51996 | 70572 | 0.682 | 0.900 | 1.194 |
| E4 | 1.138 | 130.1 | 260.2 | 250 | 18551 | 14388 | 10066 | 0.701 | 0.925 | 1.229 |
| E7 | 0.854 | 162.6 | 260.2 | 125 | 110672 | 47705 | 30067 | 0.703 | 0.929 | 1.235 |
| E8 | 0.569 | 195.2 | 260.2 | 125 | 1678593 | 258391 | 188103 | 0.703 | 0.929 | 1.236 |


| Cont. Table C. 4 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{gathered} \Delta \varepsilon_{\max } \\ \times 10^{-3} \end{gathered}$ | $\sigma_{m}$ | $\sigma_{\text {max }}$ | $n$ | $N_{M}$ | $N_{\text {SWT }}$ | $N_{G}$ | $F_{M}$ | $F_{S W T}$ | $F_{G}$ |
| F2 | 0.643 | 63.0 | 126.0 | 1000 | 24301972 | 3186280 | 37401242 | 0.703 | 0.930 | 1.236 |
| *F3 | 0.962 | 94.2 | 188.4 | 500 | 510325 | 111314 | 508814 | 0.704 | 0.936 | 1.236 |

* first failure observed (García 2008)

| Table C. 5 Config. \#13 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{gathered} \Delta \varepsilon_{\max } \\ \times 10^{-3} \end{gathered}$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| A1 | 1.138 | 130.4 | 260.8 | 2000 | 18130 | 14260 | 10066 | 0.158 | 0.200 | 0.284 |
| A2 | 1.138 | 130.6 | 261.2 | 3500 | 17770 | 14149 | 10066 | 0.439 | 0.554 | 0.781 |
| A3 | 1.138 | 130.3 | 260.6 | 1000 | 18249 | 14296 | 10066 | 0.517 | 0.654 | 0.923 |
| A4 | 1.138 | 130.0 | 260.0 | 250 | 18704 | 14434 | 10066 | 0.536 | 0.678 | 0.958 |
| A5 | 0.285 | 97.5 | 130.0 | 2000 | 3437222388 | 83608779 | 3230672125 | 0.536 | 0.678 | 0.958 |
| A6 | 0.569 | 130.0 | 195.0 | 2000 | 6032847 | 859502 | 3246626 | 0.537 | 0.682 | 0.959 |
| A7 | 0.854 | 162.5 | 260.0 | 125 | 111655 | 47858 | 30067 | 0.538 | 0.686 | 0.965 |
| A8 | 0.569 | 195.0 | 260.0 | 125 | 1694740 | 259220 | 188103 | 0.539 | 0.686 | 0.966 |
| B2 | 0.717 | 81.3 | 162.5 | 2500 | 2175406 | 701387 | 2600197 | 0.540 | 0.691 | 0.967 |
| B3 | 1.070 | 121.9 | 243.8 | 750 | 37058 | 24442 | 23178 | 0.569 | 0.735 | 1.013 |
| B4 | 1.138 | 129.6 | 259.1 | 250 | 19410 | 14644 | 10066 | 0.587 | 0.760 | 1.049 |
| B6 | 0.569 | 129.6 | 194.3 | 1750 | 6260528 | 872010 | 3246626 | 0.588 | 0.762 | 1.050 |
| B7 | 0.854 | 161.9 | 259.1 | 125 | 116195 | 48555 | 30067 | 0.589 | 0.766 | 1.056 |
| B8 | 0.569 | 194.3 | 259.1 | 125 | 1769427 | 262992 | 188103 | 0.590 | 0.767 | 1.056 |
| C2 | 0.683 | 77.7 | 155.5 | 1250 | 3332626 | 1033692 | 4032136 | 0.590 | 0.768 | 1.057 |
| C3 | 1.029 | 116.6 | 233.2 | 750 | 58854 | 34589 | 38715 | 0.608 | 0.799 | 1.085 |
| C4 | 1.138 | 129.6 | 259.1 | 250 | 19410 | 14644 | 10066 | 0.627 | 0.824 | 1.120 |
| C6 | 0.569 | 129.6 | 194.3 | 1500 | 6260528 | 872010 | 3246626 | 0.627 | 0.826 | 1.121 |
| C7 | 0.854 | 161.9 | 259.1 | 125 | 116195 | 48555 | 30067 | 0.629 | 0.830 | 1.127 |
| C8 | 0.569 | 194.3 | 259.1 | 125 | 1769427 | 262992 | 188103 | 0.629 | 0.831 | 1.128 |
| D2 | 0.668 | 75.6 | 151.1 | 1250 | 4367185 | 1276843 | 5445022 | 0.629 | 0.832 | 1.128 |
| D3 | 0.994 | 113.4 | 226.7 | 500 | 78778 | 44913 | 54830 | 0.638 | 0.848 | 1.141 |
| D4 | 1.138 | 129.6 | 259.1 | 250 | 19410 | 14644 | 10066 | 0.657 | 0.872 | 1.176 |
| D6 | 0.569 | 129.6 | 194.3 | 250 | 6260528 | 872010 | 3246626 | 0.657 | 0.873 | 1.177 |
| D7 | 0.854 | 161.9 | 259.1 | 125 | 116195 | 48555 | 30067 | 0.658 | 0.876 | 1.183 |
| D8 | 0.569 | 194.3 | 259.1 | 125 | 1769427 | 262992 | 188103 | 0.658 | 0.877 | 1.183 |
| E2 | 0.650 | 74.0 | 148.0 | 1000 | 5316554 | 1552599 | 6770673 | 0.658 | 0.878 | 1.184 |
| E3 | 0.975 | 111.0 | 222.1 | 500 | 97449 | 52921 | 70572 | 0.666 | 0.892 | 1.194 |
| *E4 | 1.138 | 129.6 | 259.1 | 250 | 19410 | 14644 | 10066 | 0.684 | 0.916 | 1.229 |


| Table C. 6 Config. \#15 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{array}{r} \Delta \varepsilon_{\max } \\ \times 10^{-3} \\ \hline \end{array}$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| A1 | 1.138 | 131.3 | 262.6 | 2000 | 16814 | 13847 | 10066 | 0.170 | 0.206 | 0.284 |
| A2 | 1.138 | 131.3 | 262.6 | 3500 | 16814 | 13847 | 10066 | 0.467 | 0.567 | 0.781 |
| A3 | 1.138 | 131.3 | 262.6 | 1000 | 16814 | 13847 | 10066 | 0.552 | 0.671 | 0.923 |
| A4 | 1.138 | 131.3 | 262.6 | 250 | 16814 | 13847 | 10066 | 0.573 | 0.696 | 0.958 |
| A5 | 0.285 | 98.5 | 131.3 | 2000 | 3111884196 | 80213258 | \# | 0.573 | 0.696 | 0.958 |
| A6 | 0.569 | 131.3 | 197.0 | 2000 | 5423343 | 824595 | 3246626 | 0.574 | 0.700 | 0.959 |
| A7 | 0.854 | 164.1 | 262.6 | 125 | 99559 | 45914 | 30067 | 0.576 | 0.704 | 0.965 |
| A8 | 0.569 | 197.0 | 262.6 | 125 | 1496849 | 248693 | 188103 | 0.576 | 0.704 | 0.966 |
| B2 | 0.717 | 82.1 | 164.1 | 2500 | 1975803 | 672903 | 2600197 | 0.578 | 0.710 | 0.967 |
| B3 | 1.070 | 123.1 | 246.2 | 750 | 33376 | 23449 | 23178 | 0.610 | 0.755 | 1.013 |
| B4 | 1.138 | 131.3 | 262.6 | 250 | 16814 | 13847 | 10066 | 0.631 | 0.781 | 1.049 |
| B6 | 0.569 | 131.3 | 197.0 | 1750 | 5423343 | 824595 | 3246626 | 0.632 | 0.784 | 1.050 |
| B7 | 0.854 | 164.1 | 262.6 | 125 | 99559 | 45914 | 30067 | 0.633 | 0.788 | 1.056 |
| B8 | 0.569 | 197.0 | 262.6 | 125 | 1496849 | 248693 | 188103 | 0.633 | 0.789 | 1.056 |
| C2 | 0.683 | 78.8 | 157.6 | 1250 | 2929538 | 977487 | 4032136 | 0.634 | 0.791 | 1.057 |
| C3 | 1.029 | 118.2 | 236.3 | 750 | 51186 | 32708 | 38715 | 0.655 | 0.823 | 1.085 |
| C4 | 1.138 | 131.3 | 262.6 | 250 | 16814 | 13847 | 10066 | 0.676 | 0.849 | 1.120 |
| C7 | 0.569 | 131.3 | 197.0 | 125 | 5423343 | 824595 | 3246626 | 0.677 | 0.852 | 1.121 |
| C8 | 0.854 | 164.1 | 262.6 | 125 | 99559 | 45914 | 30067 | 0.678 | 0.856 | 1.127 |

C - first failure observed (García 2008)

| Table C. 7 Config. \#17 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{array}{r} \Delta \varepsilon_{\max } \\ \times 10^{-3} \\ \hline \end{array}$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| A1 | 1.138 | 123.7 | 247.4 | 2000 | 31692 | 17753 | 10066 | 0.090 | 0.161 | 0.284 |
| A2 | 1.138 | 124.3 | 248.5 | 3500 | 30243 | 17428 | 10066 | 0.255 | 0.448 | 0.781 |
| A3 | 1.138 | 124.8 | 249.5 | 1000 | 28987 | 17138 | 10066 | 0.305 | 0.531 | 0.923 |
| A4 | 1.138 | 125.3 | 250.5 | 250 | 27787 | 16855 | 10066 | 0.318 | 0.552 | 0.958 |
| A5 | 0.285 | 93.9 | 125.3 | 2000 | 4977474649 | 97635695 | \# | 0.318 | 0.552 | 0.958 |
| A6 | 0.569 | 125.3 | 187.9 | 2000 | 8962320 | 1003699 | 3246626 | 0.318 | 0.555 | 0.959 |
| A7 | 0.854 | 156.6 | 250.5 | 125 | 170812 | 55887 | 30067 | 0.319 | 0.558 | 0.965 |
| A8 | 0.569 | 187.9 | 250.5 | 125 | 2682417 | 302709 | 188103 | 0.319 | 0.559 | 0.966 |
| B2 | 0.717 | 78.3 | 156.6 | 2500 | 3113966 | 819058 | 2600197 | 0.320 | 0.563 | 0.967 |
| B3 | 1.070 | 117.4 | 234.8 | 750 | 54685 | 28542 | 23178 | 0.340 | 0.601 | 1.013 |
| B4 | 1.138 | 125.8 | 251.5 | 250 | 26639 | 16578 | 10066 | 0.353 | 0.622 | 1.049 |
| B6 | 0.569 | 125.8 | 188.6 | 1750 | 8592220 | 987175 | 3246626 | 0.353 | 0.625 | 1.050 |
| B7 | 0.854 | 157.2 | 251.5 | 125 | 163258 | 54967 | 30067 | 0.355 | 0.628 | 1.056 |
| B8 | 0.569 | 188.6 | 251.5 | 125 | 2554760 | 297725 | 188103 | 0.355 | 0.629 | 1.056 |
| C2 | 0.683 | 75.5 | 150.9 | 1250 | 4432063 | 1170210 | 4032136 | 0.355 | 0.630 | 1.057 |
| C3 | 1.029 | 113.2 | 226.4 | 750 | 80089 | 39157 | 38715 | 0.368 | 0.658 | 1.085 |
| C4 | 1.138 | 126.3 | 252.5 | 250 | 25542 | 16306 | 10066 | 0.382 | 0.680 | 1.120 |
| C6 | 0.569 | 126.3 | 189.4 | 1500 | 8238409 | 970986 | 3246626 | 0.383 | 0.682 | 1.121 |
| C7 | 0.854 | 157.8 | 252.5 | 125 | 156057 | 54066 | 30067 | 0.384 | 0.685 | 1.127 |
| C8 | 0.569 | 189.4 | 252.5 | 125 | 2433428 | 292843 | 188103 | 0.384 | 0.686 | 1.128 |
| D2 | 0.668 | 73.6 | 147.3 | 1250 | 5586033 | 1421770 | 5445022 | 0.384 | 0.687 | 1.128 |
| D3 | 0.994 | 110.5 | 220.9 | 500 | 102719 | 50010 | 54830 | 0.391 | 0.701 | 1.141 |
| D4 | 1.138 | 126.4 | 252.8 | 250 | 25223 | 16225 | 10066 | 0.405 | 0.723 | 1.176 |
| D6 | 0.569 | 126.4 | 189.6 | 250 | 8135327 | 966194 | 3246626 | 0.405 | 0.724 | 1.177 |
| D7 | 0.854 | 158.0 | 252.8 | 125 | 153963 | 53799 | 30067 | 0.407 | 0.727 | 1.183 |
| D8 | 0.569 | 189.6 | 252.8 | 125 | 2398213 | 291398 | 188103 | 0.407 | 0.728 | 1.183 |
| E2 | 0.650 | 72.2 | 144.4 | 1000 | 6719250 | 1720293 | 6770673 | 0.407 | 0.728 | 1.184 |
| E3 | 0.975 | 108.3 | 216.7 | 500 | 125375 | 58637 | 70572 | 0.413 | 0.741 | 1.194 |
| E4 | 1.138 | 126.6 | 253.2 | 250 | 24803 | 16119 | 10066 | 0.427 | 0.763 | 1.229 |
| E7 | 0.854 | 158.3 | 253.2 | 125 | 151216 | 53446 | 281401 | 0.428 | 0.766 | 1.230 |
| E8 | 0.569 | 189.9 | 253.2 | 125 | 2352084 | 289485 | 2601114 | 0.428 | 0.767 | 1.230 |


| Cont. Table C. 7 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{gathered} \Delta \varepsilon_{\max } \\ \times 10^{-3} \end{gathered}$ | $\sigma_{m}$ | $\sigma_{\text {max }}$ | $n$ | $N_{M}$ | $N_{\text {SWT }}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| F2 | 0.643 | 61.2 | 122.4 | 1000 | 31794020 | 3593297 | 37401242 | 0.428 | 0.767 | 1.230 |
| F3 | 0.962 | 91.5 | 183.1 | 500 | 678855 | 125533 | 508814 | 0.429 | 0.773 | 1.231 |
| F4 | 1.138 | 126.8 | 253.5 | 250 | 24493 | 16040 | 10066 | 0.444 | 0.795 | 1.267 |
| F7 | 0.854 | 158.4 | 253.5 | 125 | 149190 | 53183 | 2539689 | 0.445 | 0.798 | 1.267 |
| F8 | 0.569 | 190.1 | 253.5 | 125 | 2318094 | 288060 | 31141824 | 0.445 | 0.799 | 1.267 |
| G2 | 0.632 | 50.9 | 101.7 | 1000 | 175007528 | 8341896 | \# | 0.445 | 0.799 | 1.267 |
| G3 | 0.951 | 76.5 | 153.0 | 500 | 3887592 | 278099 | 3679806 | 0.445 | 0.802 | 1.267 |
| G4 | 1.138 | 126.9 | 253.8 | 250 | 24188 | 15961 | 10066 | 0.460 | 0.824 | 1.303 |
| G7 | 0.854 | 158.6 | 253.8 | 125 | 147193 | 52921 | 21841432 | 0.461 | 0.828 | 1.303 |
| G8 | 0.569 | 190.4 | 253.8 | 125 | 2284617 | 286644 | \# | 0.461 | 0.828 | 1.303 |
| H2 | 0.628 | 42.2 | 84.4 | 1000 | 941918293 | 18603917 | \# | 0.461 | 0.828 | 1.303 |
| H3 | 0.938 | 63.0 | 126.1 | 500 | 24207877 | 660242 | 27776585 | 0.461 | 0.829 | 1.303 |
| H4 | 1.138 | 127.1 | 254.1 | 250 | 23886 | 15882 | 10066 | 0.476 | 0.852 | 1.338 |
| H7 | 0.854 | 158.8 | 254.1 | 125 | 145224 | 52661 | \# | 0.478 | 0.855 | 1.338 |
| H8 | 0.569 | 190.6 | 254.1 | 125 | 2251643 | 285236 | \# | 0.478 | 0.856 | 1.338 |
| I2 | 0.621 | 34.4 | 68.8 | 1000 | 5878308923 | 46033988 | \# | 0.478 | 0.856 | 1.338 |
| I3 | 0.931 | 51.6 | 103.1 | 500 | 154197241 | 1569103 | \# | 0.478 | 0.856 | 1.338 |
| *I4 | 1.138 | 127.2 | 254.4 | 250 | 23589 | 15804 | 10066 | 0.493 | 0.879 | 1.374 |

* first failure observed (García 2008)

| Table C. 8 Config. \#18 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{aligned} & \Delta \varepsilon_{\max } \\ & \times 10^{-3} \end{aligned}$ |  | $\sigma_{\text {max }}$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| A1 | 1.138 | 126.3 | 252.6 | 2000 | 25435 | 16279 | 10066 | 0.112 | 0.176 | 0.284 |
| A2 | 1.138 | 126.3 | 252.6 | 3500 | 25435 | 16279 | 10066 | 0.309 | 0.483 | 0.781 |
| A3 | 1.138 | 126.3 | 252.5 | 1000 | 25542 | 16306 | 10066 | 0.365 | 0.570 | 0.923 |
| A4 | 1.138 | 126.2 | 252.4 | 250 | 25650 | 16333 | 10066 | 0.379 | 0.592 | 0.958 |
| A5 | 0.285 | 94.7 | 126.2 | 2000 | 4618060881 | 94609599 | \# | 0.379 | 0.592 | 0.958 |
| A6 | 0.569 | 126.2 | 189.3 | 2000 | 8273079 | 972590 | 3246626 | 0.379 | 0.595 | 0.959 |
| A7 | 0.854 | 157.8 | 252.4 | 125 | 156762 | 54155 | 30067 | 0.380 | 0.598 | 0.965 |
| A8 | 0.569 | 189.3 | 252.4 | 125 | 2445287 | 293327 | 188103 | 0.380 | 0.599 | 0.966 |
| B2 | 0.717 | 78.9 | 157.8 | 2500 | 2895775 | 793672 | 2600197 | 0.382 | 0.604 | 0.967 |
| B3 | 1.070 | 118.3 | 236.6 | 750 | 50547 | 27658 | 23178 | 0.403 | 0.642 | 1.013 |
| B4 | 1.138 | 126.2 | 252.4 | 250 | 25650 | 16333 | 10066 | 0.417 | 0.664 | 1.049 |
| B6 | 0.569 | 126.2 | 189.3 | 1750 | 8273079 | 972590 | 3246626 | 0.417 | 0.667 | 1.050 |
| B7 | 0.854 | 157.8 | 252.4 | 125 | 156762 | 54155 | 30067 | 0.418 | 0.670 | 1.056 |
| B8 | 0.569 | 189.3 | 252.4 | 125 | 2445287 | 293327 | 188103 | 0.418 | 0.671 | 1.056 |
| C2 | 0.683 | 75.7 | 151.4 | 1250 | 4283296 | 1152922 | 4032136 | 0.419 | 0.672 | 1.057 |
| C3 | 1.029 | 113.6 | 227.2 | 750 | 77191 | 38579 | 38715 | 0.432 | 0.700 | 1.085 |
| C4 | 1.138 | 126.2 | 252.4 | 250 | 25650 | 16333 | 10066 | 0.446 | 0.722 | 1.120 |
| C6 | 0.569 | 126.2 | 189.3 | 1500 | 8273079 | 972590 | 3246626 | 0.447 | 0.724 | 1.121 |
| C7 | 0.854 | 157.8 | 252.4 | 125 | 156762 | 54155 | 30067 | 0.448 | 0.727 | 1.127 |
| C8 | 0.569 | 189.3 | 252.4 | 125 | 2445287 | 293327 | 188103 | 0.448 | 0.728 | 1.128 |
| D2 | 0.668 | 73.6 | 147.2 | 1250 | 5607141 | 1424119 | 5445022 | 0.448 | 0.729 | 1.128 |
| D3 | 0.994 | 110.4 | 220.9 | 500 | 103137 | 50093 | 54830 | 0.455 | 0.743 | 1.141 |
| D4 | 1.138 | 126.2 | 252.4 | 250 | 25650 | 16333 | 10066 | 0.469 | 0.765 | 1.176 |
| D6 | 0.569 | 126.2 | 189.3 | 250 | 8273079 | 972590 | 3246626 | 0.469 | 0.766 | 1.177 |
| D7 | 0.854 | 157.8 | 252.4 | 125 | 156762 | 54155 | 30067 | 0.470 | 0.769 | 1.183 |
| D8 | 0.569 | 189.3 | 252.4 | 125 | 2445287 | 293327 | 188103 | 0.470 | 0.770 | 1.183 |
| E2 | 0.650 | 72.1 | 144.2 | 1000 | 6821040 | 1731681 | 6770673 | 0.471 | 0.770 | 1.184 |
| E3 | 0.975 | 108.2 | 216.3 | 500 | 127417 | 59026 | 70572 | 0.476 | 0.782 | 1.194 |
| E4 | 1.138 | 126.2 | 252.4 | 250 | 25650 | 16333 | 10066 | 0.490 | 0.804 | 1.229 |
| E7 | 0.854 | 157.8 | 252.4 | 125 | 156762 | 54155 | 281401 | 0.491 | 0.808 | 1.230 |
| E8 | 0.569 | 189.3 | 252.4 | 125 | 2445287 | 293327 | 2601114 | 0.491 | 0.808 | 1.230 |


| Cont. Table C. 8 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{gathered} \Delta \varepsilon_{\max } \\ \times 10^{-3} \\ \hline \end{gathered}$ | $\sigma_{m}$ | $\sigma_{\text {max }}$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| F2 | 0.643 | 61.1 | 122.2 | 1000 | 32265568 | 3617084 | 37401242 | 0.491 | 0.809 | 1.230 |
| F3 | 0.962 | 91.4 | 182.8 | 500 | 689540 | 126364 | 508814 | 0.492 | 0.814 | 1.231 |
| F4 | 1.138 | 126.2 | 252.4 | 250 | 25650 | 16333 | 10066 | 0.506 | 0.836 | 1.267 |
| F7 | 0.854 | 157.8 | 252.4 | 125 | 156762 | 54155 | 2539689 | 0.507 | 0.839 | 1.267 |
| F8 | 0.569 | 189.3 | 252.4 | 125 | 2445287 | 293327 | 31141824 | 0.507 | 0.840 | 1.267 |
| G2 | 0.632 | 50.8 | 101.6 | 1000 | 177552966 | 8397118 | \# | 0.507 | 0.840 | 1.267 |
| G3 | 0.951 | 76.4 | 152.7 | 500 | 3946977 | 279940 | 3679806 | 0.508 | 0.843 | 1.267 |
| G4 | 1.138 | 126.0 | 252.0 | 250 | 26085 | 16441 | 10066 | 0.521 | 0.865 | 1.303 |
| G7 | 0.854 | 157.5 | 252.0 | 125 | 159615 | 54514 | 21841432 | 0.522 | 0.868 | 1.303 |
| G8 | 0.569 | 189.0 | 252.0 | 125 | 2493325 | 295272 | \# | 0.523 | 0.868 | 1.303 |



| Cont. Table C. 9 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{gathered} \Delta \varepsilon_{\max } \\ \times 10^{-3} \end{gathered}$ | $\sigma_{m}$ | $\sigma_{\text {max }}$ | $n$ | $N_{M}$ | $N_{\text {SWT }}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| F2 | 0.643 | 61.5 | 122.9 | 1000 | 30567429 | 3530516 | 39119332 | 0.526 | 0.800 | 1.124 |
| F3 | 0.962 | 92.2 | 184.4 | 500 | 632012 | 121801 | 517083 | 0.527 | 0.806 | 1.126 |
| F4 | 1.138 | 125.4 | 250.7 | 250 | 27553 | 16799 | 10066 | 0.540 | 0.827 | 1.161 |
| F7 | 0.854 | 156.7 | 250.7 | 125 | 169272 | 55702 | 30067 | 0.541 | 0.831 | 1.167 |
| F8 | 0.569 | 188.0 | 250.7 | 125 | 2656364 | 301704 | 188103 | 0.541 | 0.831 | 1.168 |
| G2 | 0.632 | 51.2 | 102.5 | 1000 | 163696927 | 8091183 | \# | 0.541 | 0.831 | 1.168 |
| G3 | 0.951 | 76.8 | 153.7 | 500 | 3722142 | 272887 | 3845889 | 0.541 | 0.834 | 1.168 |
| G4 | 1.138 | 125.1 | 250.1 | 250 | 28260 | 16968 | 10066 | 0.554 | 0.855 | 1.204 |
| G7 | 0.854 | 156.3 | 250.1 | 125 | 173936 | 56260 | 30067 | 0.555 | 0.858 | 1.210 |
| G8 | 0.569 | 187.6 | 250.1 | 125 | 2735328 | 304731 | 188103 | 0.555 | 0.859 | 1.211 |
| H2 | 0.628 | 42.3 | 84.5 | 1000 | 934248552 | 18533443 | \# | 0.555 | 0.859 | 1.211 |
| H3 | 0.938 | 63.4 | 126.8 | 500 | 22942212 | 644602 | 28613377 | 0.555 | 0.860 | 1.211 |
| H4 | 1.138 | 124.8 | 249.5 | 250 | 28987 | 17138 | 10066 | 0.567 | 0.881 | 1.246 |
| H7 | 0.854 | 155.9 | 249.5 | 125 | 178735 | 56826 | 30067 | 0.568 | 0.884 | 1.252 |
| H8 | 0.569 | 187.1 | 249.5 | 125 | 2816745 | 307796 | 188103 | 0.568 | 0.885 | 1.253 |
| I2 | 0.621 | 34.6 | 69.2 | 1000 | 5582348775 | 44927942 | \# | 0.568 | 0.885 | 1.253 |
| I3 | 0.931 | 51.9 | 103.7 | 500 | 146273281 | 1531793 | \# | 0.568 | 0.885 | 1.253 |
| I4 | 1.138 | 124.5 | 248.9 | 250 | 29734 | 17311 | 10066 | 0.580 | 0.906 | 1.288 |
| I7 | 0.854 | 155.6 | 248.9 | 125 | 183675 | 57399 | 30067 | 0.581 | 0.909 | 1.294 |
| I8 | 0.569 | 186.7 | 248.9 | 125 | 2900696 | 310900 | 188103 | 0.581 | 0.909 | 1.295 |
| J2 | 0.618 | 28.1 | 56.2 | 1000 | 34697325916 | 108468242 | \# | 0.581 | 0.909 | 1.295 |
| J3 | 0.926 | 42.1 | 84.3 | 500 | 958220241 | 3729848 | \# | 0.581 | 0.910 | 1.295 |
| J4 | 1.138 | 124.2 | 248.3 | 250 | 30501 | 17486 | 10066 | 0.593 | 0.930 | 1.331 |
| J7 | 0.854 | 155.2 | 248.3 | 125 | 188759 | 57979 | 30067 | 0.594 | 0.933 | 1.337 |
| J8 | 0.569 | 186.2 | 248.3 | 125 | 2987262 | 314042 | 188103 | 0.594 | 0.934 | 1.338 |
| K2 | 0.613 | 22.7 | 45.4 | 1000 | 222765457835 | 274194362 | \# | 0.594 | 0.934 | 1.338 |
| K3 | 0.921 | 34.0 | 68.1 | 500 | 6419921317 | 9274700 | \# | 0.594 | 0.934 | 1.338 |
| K4 | 1.138 | 123.9 | 247.7 | 250 | 31290 | 17663 | 10066 | 0.606 | 0.954 | 1.373 |
| K7 | 0.854 | 154.8 | 247.7 | 125 | 193992 | 58567 | 30067 | 0.606 | 0.957 | 1.379 |
| K8 | 0.569 | 185.8 | 247.7 | 125 | 30567429 | 3530516 | 188103 | 0.607 | 0.958 | 1.380 |

[^1]| Table C.10 Config. \#20 |  |  |  |  |  |  |  |  |  |  |
| :--- | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
|  | $\Delta \varepsilon_{\text {max }}$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{S W T}$ | $F_{G}$ |
|  | $\times 10^{-3}$ |  |  |  |  |  |  |  |  |  |
| A1 | 1.138 | 131.4 | 262.7 | 2000 | 16746 | 13826 | 10066 | 0.171 | 0.207 | 0.284 |
| A2 | 1.138 | 131.3 | 262.5 | 3500 | 16883 | 13869 | 10066 | 0.467 | 0.567 | 0.781 |
| A3 | 1.138 | 131.2 | 262.4 | 1000 | 16952 | 13892 | 10066 | 0.551 | 0.670 | 0.923 |
| A4 | 1.138 | 131.2 | 262.3 | 250 | 17022 | 13914 | 10066 | 0.572 | 0.696 | 0.958 |
| A5 | 0.285 | 98.4 | 131.2 | 2000 | 314766449 | 80596210 | $\#$ | 0.572 | 0.696 | 0.958 |
| A6 | 0.569 | 131.2 | 196.7 | 2000 | 5490191 | 828532 | 3246626 | 0.573 | 0.699 | 0.959 |
| A7 | 0.854 | 163.9 | 262.3 | 125 | 100881 | 46134 | 30067 | 0.574 | 0.703 | 0.965 |
| A8 | 0.569 | 196.7 | 262.3 | 125 | 1518401 | 249880 | 188103 | 0.574 | 0.704 | 0.966 |
| B2 | 0.717 | 82.0 | 163.9 | 2500 | 1997784 | 676115 | 2600197 | 0.576 | 0.709 | 0.967 |
| B3 | 1.070 | 123.0 | 245.9 | 750 | 33781 | 23561 | 23178 | 0.608 | 0.754 | 1.013 |
| B4 | 1.138 | 131.1 | 262.1 | 250 | 17161 | 13958 | 10066 | 0.629 | 0.780 | 1.049 |
| B6 | 0.569 | 131.1 | 196.6 | 1750 | 5535244 | 831170 | 3246626 | 0.629 | 0.783 | 1.050 |
| B7 | 0.854 | 163.8 | 262.1 | 125 | 101773 | 46280 | 30067 | 0.631 | 0.787 | 1.056 |
| B8 | 0.569 | 196.6 | 262.1 | 125 | 1532948 | 250675 | 188103 | 0.631 | 0.788 | 1.056 |
| C2 | 0.683 | 78.6 | 157.3 | 1250 | 2983723 | 985280 | 4032136 | 0.632 | 0.789 | 1.057 |
| C3 | 1.029 | 117.9 | 235.9 | 750 | 52212 | 32969 | 38715 | 0.652 | 0.822 | 1.085 |
| C4 | 1.138 | 131.0 | 262.0 | 250 | 17232 | 13980 | 10066 | 0.673 | 0.848 | 1.120 |
| C6 | 0.569 | 131.0 | 196.5 | 1500 | 5557918 | 832492 | 3246626 | 0.673 | 0.850 | 1.121 |
| C7 | 0.854 | 163.8 | 262.0 | 125 | 102222 | 46354 | 30067 | 0.675 | 0.854 | 1.127 |
| C8 | 0.569 | 196.5 | 262.0 | 125 | 1540275 | 251074 | 188103 | 0.675 | 0.855 | 1.128 |
| D2 | 0.668 | 76.4 | 152.8 | 1250 | 3926119 | 1218979 | 5445022 | 0.676 | 0.856 | 1.128 |
| *D3 | 0.994 | 114.6 | 229.3 | 500 | 70222 | 42877 | 54830 | 0.686 | 0.873 | 1.141 |

* first failure observed (García 2008)

| Table C. 11 Config. \#21 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{array}{r} \Delta \varepsilon_{\max } \\ \times 10^{-3} \\ \hline \end{array}$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{S W T}$ | $F_{G}$ |
| A1 | 1.138 | 131.4 | 262.7 | 2000 | 16746 | 13826 | 10066 | 0.171 | 0.207 | 0.284 |
| A2 | 1.138 | 131.4 | 262.7 | 3500 | 16746 | 13826 | 10066 | 0.469 | 0.568 | 0.781 |
| A3 | 1.138 | 131.4 | 262.7 | 1000 | 16746 | 13826 | 10066 | 0.555 | 0.672 | 0.923 |
| A4 | 1.138 | 131.4 | 262.7 | 250 | 16746 | 13826 | 10066 | 0.576 | 0.697 | 0.958 |
| A5 | 0.285 | 98.5 | 131.4 | 2000 | 3100054596 | 80086109 | \#\#\#\#\#\#\#\#\# | 0.576 | 0.698 | 0.958 |
| A6 | 0.569 | 131.4 | 197.0 | 2000 | 5401253 | 823288 | 3246626 | 0.576 | 0.701 | 0.959 |
| A7 | 0.854 | 164.2 | 262.7 | 125 | 99122 | 45842 | 30067 | 0.578 | 0.705 | 0.965 |
| A8 | 0.569 | 197.0 | 262.7 | 125 | 1489736 | 248298 | 188103 | 0.578 | 0.706 | 0.966 |
| B2 | 0.717 | 82.1 | 164.2 | 2500 | 1968534 | 671836 | 2600197 | 0.580 | 0.711 | 0.967 |
| B3 | 1.070 | 123.1 | 246.3 | 750 | 33243 | 23412 | 23178 | 0.612 | 0.757 | 1.013 |
| B4 | 1.138 | 131.4 | 262.7 | 250 | 16746 | 13826 | 10066 | 0.634 | 0.782 | 1.049 |
| B6 | 0.569 | 131.4 | 197.0 | 1750 | 5401253 | 823288 | 3246626 | 0.634 | 0.786 | 1.050 |
| B7 | 0.854 | 164.2 | 262.7 | 125 | 99122 | 45842 | 30067 | 0.636 | 0.789 | 1.056 |
| B8 | 0.569 | 197.0 | 262.7 | 125 | 1489736 | 248298 | 188103 | 0.636 | 0.790 | 1.056 |
| C2 | 0.683 | 78.8 | 157.6 | 1250 | 2918830 | 975937 | 4032136 | 0.637 | 0.792 | 1.057 |
| C3 | 1.029 | 118.2 | 236.4 | 750 | 50983 | 32657 | 38715 | 0.658 | 0.825 | 1.085 |
| C4 | 1.138 | 131.4 | 262.7 | 250 | 16746 | 13826 | 10066 | 0.679 | 0.851 | 1.120 |
| C6 | 0.569 | 131.4 | 197.0 | 1500 | 5401253 | 823288 | 3246626 | 0.679 | 0.853 | 1.121 |
| C7 | 0.854 | 164.2 | 262.7 | 125 | 99122 | 45842 | 30067 | 0.681 | 0.857 | 1.127 |
| C8 | 0.569 | 197.0 | 262.7 | 125 | 1489736 | 248298 | 188103 | 0.681 | 0.858 | 1.128 |
| D2 | 0.668 | 76.6 | 153.2 | 1250 | 3827083 | 1205503 | 5445022 | 0.682 | 0.859 | 1.128 |
| *D3 | 0.994 | 114.9 | 229.9 | 500 | 68310 | 42403 | 54830 | 0.692 | 0.876 | 1.141 |

* first failure observed (García 2008)

| Table C. 12 Config. \#24 |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{aligned} & \Delta \varepsilon_{\max } \\ & \times 10^{-3} \end{aligned}$ | $\sigma_{m}$ | $\overline{\sigma_{\max }}$ | $n$ | $N_{M}$ | $N_{S W T}$ | $F_{M}$ | $F_{S W T}$ |
| A1 | 0.874 | 104.7 | 209.4 | 2000 | 177435 | 106618 | 0.016 | 0.027 |
| A2 | 1.138 | 131.3 | 262.6 | 3500 | 16814 | 13847 | 0.313 | 0.388 |
| A3 | 1.138 | 131.2 | 262.4 | 1000 | 16952 | 13892 | 0.398 | 0.491 |
| A4 | 1.138 | 131.2 | 262.3 | 250 | 17022 | 13914 | 0.419 | 0.516 |
| A5 | 0.285 | 98.4 | 131.2 | 2000 | 3147666449 | 80596210 | 0.419 | 0.516 |
| A6 | 0.569 | 131.2 | 196.7 | 2000 | 5490191 | 828532 | 0.419 | 0.520 |
| A7 | 0.854 | 163.9 | 262.3 | 125 | 100881 | 46134 | 0.421 | 0.524 |
| A8 | 0.569 | 196.7 | 262.3 | 125 | 1518401 | 249880 | 0.421 | 0.524 |
| B2 | 0.717 | 82.0 | 163.9 | 2500 | 1997784 | 676115 | 0.423 | 0.530 |
| B3 | 1.070 | 123.0 | 245.9 | 750 | 33781 | 23561 | 0.455 | 0.575 |
| B4 | 1.138 | 131.1 | 262.2 | 250 | 17091 | 13936 | 0.476 | 0.601 |
| B6 | 0.569 | 131.1 | 196.7 | 1750 | 5512668 | 829850 | 0.476 | 0.604 |
| B7 | 0.854 | 163.9 | 262.2 | 125 | 101326 | 46207 | 0.478 | 0.608 |
| B8 | 0.569 | 196.7 | 262.2 | 125 | 1525656 | 250277 | 0.478 | 0.608 |
| C2 | 0.683 | 78.7 | 157.3 | 1250 | 2972799 | 983715 | 0.478 | 0.610 |
| C3 | 1.029 | 118.0 | 236.0 | 750 | 52005 | 32917 | 0.499 | 0.643 |
| C4 | 1.138 | 131.1 | 262.2 | 250 | 17091 | 13936 | 0.520 | 0.668 |
| C6 | 0.569 | 131.1 | 196.7 | 1500 | 5512668 | 829850 | 0.520 | 0.671 |
| C7 | 0.854 | 163.9 | 262.2 | 125 | 101326 | 46207 | 0.522 | 0.675 |
| C8 | 0.569 | 196.7 | 262.2 | 125 | 1525656 | 250277 | 0.522 | 0.676 |
| D2 | 0.668 | 76.5 | 152.9 | 1250 | 3897541 | 1215110 | 0.523 | 0.677 |
| D3 | 0.994 | 114.7 | 229.4 | 500 | 69670 | 42741 | 0.533 | 0.694 |
| D4 | 1.138 | 131.1 | 262.2 | 250 | 17091 | 13936 | 0.554 | 0.719 |
| D6 | 0.569 | 131.1 | 196.7 | 250 | 5512668 | 829850 | 0.554 | 0.720 |
| D7 | 0.854 | 163.9 | 262.2 | 125 | 101326 | 46207 | 0.556 | 0.724 |
| D8 | 0.569 | 196.7 | 262.2 | 125 | 1525656 | 250277 | 0.556 | 0.724 |
| E2 | 0.650 | 74.9 | 149.8 | 1000 | 4746440 | 1477533 | 0.556 | 0.725 |
| E3 | 0.975 | 112.4 | 224.7 | 500 | 86234 | 50363 | 0.564 | 0.740 |
| E4 | 1.138 | 131.1 | 262.1 | 250 | 17161 | 13958 | 0.585 | 0.765 |
| E7 | 0.854 | 163.8 | 262.1 | 125 | 101773 | 46280 | 0.587 | 0.769 |
| E8 | 0.569 | 196.6 | 262.1 | 125 | 1532948 | 250675 | 0.587 | 0.770 |


| Cont. Table C. 12 |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{gathered} \Delta \varepsilon_{\max } \\ \times 10^{-3} \end{gathered}$ | $\sigma_{m}$ | $\sigma_{\text {max }}$ | $n$ | $N_{M}$ | $N_{\text {SWT }}$ | $F_{M}$ | $F_{S W T}$ |
| F2 | 0.643 | 63.2 | 126.4 | 1000 | 23582091 | 3143784 | 0.587 | 0.770 |
| F3 | 0.962 | 94.8 | 189.6 | 500 | 479704 | 108459 | 0.589 | 0.777 |
| F4 | 1.138 | 131.1 | 262.1 | 250 | 17161 | 13958 | 0.609 | 0.802 |
| F7 | 0.854 | 163.8 | 262.1 | 125 | 101773 | 46280 | 0.611 | 0.806 |
| F8 | 0.569 | 196.6 | 262.1 | 125 | 1532948 | 250675 | 0.611 | 0.807 |
| G2 | 0.632 | 52.7 | 105.4 | 1000 | 126922418 | 7204877 | 0.611 | 0.807 |
| G3 | 0.951 | 79.0 | 158.0 | 500 | 2848962 | 242995 | 0.612 | 0.810 |
| *G4 | 1.138 | 131.1 | 262.1 | 250 | 17161 | 13958 | 0.632 | 0.836 |

* first failure observed (García 2008)
D. Fatigue Damage Results, Strain Hardening

| Table D.1 Config. \#3 |  |  |  |  |  |  |  |  |  |  |
| :--- | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
|  | $\Delta \varepsilon_{\max }$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{S W T}$ | $F_{G}$ |
|  | $\times 10^{-3}$ |  |  |  |  |  |  |  |  |  |
| A1 | 1.138 | 129.7 | 259.4 | 2000 | 19182 | 14576 | 10065 | 0.149 | 0.196 | 0.284 |
| A2 | 1.138 | 129.4 | 258.8 | 3500 | 19685 | 14724 | 10065 | 0.403 | 0.536 | 0.781 |
| A3 | 1.138 | 128.5 | 257.0 | 1000 | 21170 | 15149 | 10065 | 0.470 | 0.630 | 0.923 |
| A4 | 1.159 | 128.1 | 256.2 | 250 | 21885 | 14221 | 7812 | 0.487 | 0.655 | 0.968 |
| A5 | 0.290 | 96.1 | 128.1 | 2000 | 3980618657 | 82377053 | $\#$ | 0.487 | 0.655 | 0.968 |
| A6 | 0.580 | 128.1 | 192.2 | 2000 | 7058741 | 846839 | 2519629 | 0.487 | 0.658 | 0.969 |
| A7 | 0.869 | 160.1 | 256.2 | 125 | 132195 | 47153 | 22337 | 0.489 | 0.662 | 0.977 |
| A8 | 0.580 | 192.2 | 256.2 | 125 | 2034282 | 255401 | 131359 | 0.489 | 0.663 | 0.979 |
| B2 | 0.730 | 80.1 | 160.1 | 2500 | 2507634 | 691054 | 2112509 | 0.490 | 0.668 | 0.981 |
| B3 | 1.089 | 120.1 | 240.2 | 750 | 43243 | 24082 | 18150 | 0.515 | 0.713 | 1.040 |
| B4 | 1.220 | 128.6 | 257.1 | 250 | 21083 | 11318 | 3227 | 0.532 | 0.744 | 1.150 |
| B6 | 0.610 | 128.6 | 192.8 | 1750 | 6799984 | 673965 | 1040887 | 0.532 | 0.748 | 1.153 |
| B7 | 0.915 | 160.7 | 257.1 | 125 | 126995 | 37527 | 7822 | 0.534 | 0.753 | 1.175 |
| B8 | 0.610 | 192.8 | 257.1 | 125 | 1947930 | 203263 | 36139 | 0.534 | 0.753 | 1.180 |
| C2 | 0.732 | 77.1 | 154.3 | 1250 | 3589782 | 798927 | 1620714 | 0.534 | 0.756 | 1.181 |
| C3 | 1.103 | 115.7 | 231.4 | 750 | 63781 | 26733 | 13265 | 0.551 | 0.796 | 1.262 |
| C4 | 1.292 | 136.7 | 273.3 | 250 | 10931 | 6909 | 1230 | 0.584 | 0.847 | 1.553 |
| C6 | 0.646 | 136.7 | 205.0 | 1500 | 3525827 | 411436 | 396802 | 0.584 | 0.853 | 1.558 |
| C7 | 0.969 | 170.8 | 273.3 | 125 | 62553 | 22909 | 2427 | 0.587 | 0.860 | 1.632 |
| C8 | 0.646 | 205.0 | 273.3 | 125 | 903583 | 124087 | 8158 | 0.587 | 0.862 | 1.653 |
| D2 | 0.758 | 79.7 | 159.4 | 1250 | 2617235 | 602447 | 1046023 | 0.588 | 0.865 | 1.655 |
| D3 | 1.128 | 119.6 | 239.1 | 500 | 45271 | 21191 | 7863 | 0.604 | 0.899 | 1.746 |
| D4 | 1.365 | 145.9 | 291.8 | 250 | 5334 | 4183 | 469 | 0.671 | 0.984 | 2.508 |
|  |  |  |  |  |  |  |  |  |  |  |

D4 - first failure observed (García 2008)

Tables D. 1 through D. 12 were calculated using the procedure in Section 5.6 where Column one is the strain range as defined in Section 5.6, columns two and three are the mean and maximum stresses as defined in Section 5.6, Column four is the number of cycles on each loading sublevel, columns five, six, and seven are calculated from equations 13,16 and 19, and columns eight, nine and ten are calculated from equation 21.


| Cont. Table D. 2 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{gathered} \Delta \varepsilon_{\max } \\ \times 10^{-3} \end{gathered}$ | $\sigma_{m}$ | $\sigma_{\text {max }}$ | $n$ | $N_{M}$ | $N_{\text {SWT }}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| F2 | 0.785 | 68.6 | 137.1 | 1000 | 10980928 | 972837 | 5284778 | 0.654 | 1.049 | 3.971 |
| *F3 | 1.175 | 102.6 | 205.1 | 500 | 218763 | 33986 | 54961 | 0.658 | 1.070 | 3.984 |

* first failure observed (García 2008)

| Table D. 3 Config. \#5 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{array}{r} \Delta \varepsilon_{\max } \\ \times 10^{-3} \\ \hline \end{array}$ |  | $\sigma_{\text {max }}$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| A1 | 1.138 | 120.3 | 240.5 | 2000 | 42587 | 19960 | 10065 | 0.067 | 0.143 | 0.284 |
| A2 | 1.138 | 121.1 | 242.3 | 3500 | 39518 | 19375 | 10065 | 0.194 | 0.401 | 0.781 |
| A3 | 1.138 | 122.0 | 244.0 | 1000 | 36673 | 18809 | 10065 | 0.233 | 0.477 | 0.923 |
| A4 | 1.138 | 122.9 | 245.7 | 250 | 34080 | 18270 | 10065 | 0.243 | 0.497 | 0.958 |
| A5 | 0.285 | 92.1 | 122.9 | 2000 | 6027547108 | 105832570 | \# | 0.243 | 0.497 | 0.958 |
| A6 | 0.569 | 122.9 | 184.3 | 2000 | 10992226 | 1087963 | 3246250 | 0.243 | 0.499 | 0.959 |
| A7 | 0.854 | 153.6 | 245.7 | 125 | 212572 | 60579 | 30063 | 0.244 | 0.502 | 0.965 |
| A8 | 0.569 | 184.3 | 245.7 | 125 | 3394755 | 328123 | 188072 | 0.244 | 0.503 | 0.966 |
| B2 | 0.717 | 76.8 | 153.6 | 2500 | 3749124 | 887821 | 2599950 | 0.245 | 0.507 | 0.967 |
| B3 | 1.070 | 115.2 | 230.3 | 750 | 66847 | 30938 | 23176 | 0.261 | 0.542 | 1.013 |
| B4 | 1.138 | 123.7 | 247.4 | 250 | 31630 | 17739 | 10065 | 0.272 | 0.562 | 1.049 |
| B6 | 0.569 | 123.7 | 185.6 | 1750 | 10201909 | 1056326 | 3246250 | 0.273 | 0.564 | 1.050 |
| B7 | 0.854 | 154.7 | 247.4 | 125 | 196250 | 58817 | 30063 | 0.274 | 0.567 | 1.056 |
| B8 | 0.569 | 185.6 | 247.4 | 125 | 3115102 | 318581 | 188072 | 0.274 | 0.568 | 1.057 |
| C2 | 0.683 | 74.2 | 148.5 | 1250 | 5175571 | 1252183 | 4031757 | 0.274 | 0.569 | 1.057 |
| C3 | 1.029 | 111.4 | 222.7 | 750 | 94668 | 41900 | 38710 | 0.285 | 0.595 | 1.085 |
| C4 | 1.138 | 124.6 | 249.2 | 250 | 29396 | 17233 | 10065 | 0.297 | 0.615 | 1.120 |
| C6 | 0.569 | 124.6 | 186.9 | 1500 | 9481245 | 1026214 | 3246250 | 0.298 | 0.617 | 1.121 |
| C7 | 0.854 | 155.7 | 249.2 | 125 | 181436 | 57141 | 30063 | 0.299 | 0.621 | 1.127 |
| C8 | 0.569 | 186.9 | 249.2 | 125 | 2862621 | 309499 | 188072 | 0.299 | 0.621 | 1.128 |
| D2 | 0.668 | 72.7 | 145.3 | 1250 | 6338032 | 1502637 | 5444514 | 0.299 | 0.622 | 1.128 |
| D3 | 0.994 | 109.0 | 218.0 | 500 | 117674 | 52855 | 54824 | 0.305 | 0.636 | 1.141 |
| D4 | 1.138 | 124.7 | 249.4 | 250 | 29088 | 17162 | 10065 | 0.317 | 0.657 | 1.177 |
| D6 | 0.569 | 124.7 | 187.1 | 250 | 9382037 | 1021966 | 3246250 | 0.317 | 0.657 | 1.177 |
| D7 | 0.854 | 155.9 | 249.4 | 125 | 179402 | 56904 | 30063 | 0.318 | 0.660 | 1.183 |
| D8 | 0.569 | 187.1 | 249.4 | 125 | 2828064 | 308218 | 188072 | 0.318 | 0.661 | 1.184 |
| E2 | 0.650 | 71.3 | 142.5 | 1000 | 7635063 | 1819592 | 6770046 | 0.319 | 0.661 | 1.184 |
| E3 | 0.975 | 106.9 | 213.8 | 500 | 143823 | 62022 | 70564 | 0.324 | 0.673 | 1.194 |
| E4 | 1.231 | 130.6 | 261.2 | 250 | 17795 | 10205 | 2774 | 0.344 | 0.708 | 1.323 |
| E7 | 0.923 | 133.6 | 213.8 | 125 | 984071 | 77989 | 6523 | 0.345 | 0.713 | 1.350 |
| E8 | 0.616 | 160.3 | 213.8 | 125 | 17460773 | 422420 | 28791 | 0.346 | 0.714 | 1.356 |


| Cont. Table D.3 |  |  |  |  |  |  |  |  |  |  |
| :---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
|  | $\Delta \varepsilon_{\max }$ <br> $\mathrm{x} 10^{-3}$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{S W T}$ | $F_{G}$ |
| F2 | 0.696 | 60.4 | 120.8 | 1000 | 36032127 | 2739781 | 37397911 | 0.346 | 0.715 | 1.356 |
| F3 | 1.040 | 90.3 | 180.6 | 500 | 775181 | 95715 | 508763 | 0.346 | 0.722 | 1.358 |
| F4 | 1.336 | 140.0 | 280.0 | 250 | 8400 | 5433 | 683 | 0.389 | 0.788 | 1.881 |
| F7 | 1.002 | 175.0 | 280.0 | 125 | 47031 | 18015 | 6523 | 0.393 | 0.798 | 1.908 |
| F8 | 0.668 | 210.0 | 280.0 | 125 | 662071 | 97576 | 28791 | 0.393 | 0.800 | 1.914 |
| G2 | 0.742 | 50.2 | 100.4 | 1000 | 197863294 | 4522401 | $\#$ | 0.393 | 0.800 | 1.914 |
| G3 | 1.116 | 75.5 | 150.9 | 500 | 4422114 | 150766 | 3679460 | 0.393 | 0.805 | 1.915 |
| G4 | 1.420 | 151.7 | 303.3 | 250 | 3463 | 3019 | 229 | 0.496 | 0.923 | 3.473 |
| G7 | 1.065 | 189.6 | 303.3 | 125 | 17913 | 10010 | 6523 | 0.506 | 0.941 | 3.500 |
| G8 | 0.710 | 227.5 | 303.3 | 125 | 229352 | 54218 | 28791 | 0.507 | 0.944 | 3.507 |
| H2 | 0.784 | 41.7 | 83.3 | 1000 | 1062887580 | 7822895 | $\#$ | 0.507 | 0.944 | 3.507 |
| H3 | 1.170 | 62.2 | 124.4 | 500 | 27446518 | 277630 | 27774095 | 0.507 | 0.947 | 3.507 |
| *H4 | 1.500 | 163.4 | 326.9 | 250 | 1479 | 1759 | 81 | 0.749 | 1.150 | 7.906 |


| Table D. 4 Config. \#6 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{aligned} & \Delta \varepsilon_{\max } \\ & \times 10^{-3} \end{aligned}$ |  | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{\text {SWT }}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| A1 | 1.138 | 127.0 | 254.0 | 2000 | 23986 | 15908 | 10065 | 0.119 | 0.180 | 0.284 |
| A2 | 1.138 | 126.4 | 252.8 | 3500 | 25223 | 16225 | 10065 | 0.317 | 0.488 | 0.781 |
| A3 | 1.138 | 125.8 | 251.6 | 1000 | 26527 | 16550 | 10065 | 0.371 | 0.574 | 0.923 |
| A4 | 1.138 | 125.2 | 250.4 | 250 | 27904 | 16883 | 10065 | 0.384 | 0.595 | 0.958 |
| A5 | 0.285 | 93.9 | 125.2 | 2000 | 4997211332 | 97798264 | \# | 0.384 | 0.595 | 0.958 |
| A6 | 0.569 | 125.2 | 187.8 | 2000 | 9000257 | 1005370 | 3246250 | 0.384 | 0.598 | 0.959 |
| A7 | 0.854 | 156.5 | 250.4 | 125 | 171587 | 55980 | 30063 | 0.385 | 0.601 | 0.965 |
| A8 | 0.569 | 187.8 | 250.4 | 125 | 2695544 | 303213 | 188072 | 0.385 | 0.602 | 0.966 |
| B2 | 0.717 | 78.3 | 156.5 | 2500 | 3125935 | 820422 | 2599950 | 0.387 | 0.606 | 0.967 |
| B3 | 1.070 | 117.4 | 234.8 | 750 | 54913 | 28590 | 23176 | 0.406 | 0.644 | 1.013 |
| B4 | 1.232 | 129.4 | 258.8 | 250 | 19652 | 10571 | 2737 | 0.424 | 0.677 | 1.144 |
| B6 | 0.616 | 129.4 | 194.1 | 1750 | 6338445 | 629505 | 882861 | 0.425 | 0.681 | 1.147 |
| B7 | 0.924 | 161.8 | 258.8 | 125 | 117751 | 35052 | 6418 | 0.426 | 0.687 | 1.175 |
| B8 | 0.616 | 194.1 | 258.8 | 125 | 1795078 | 189855 | 28211 | 0.426 | 0.687 | 1.181 |
| C2 | 0.739 | 77.6 | 155.3 | 1250 | 3369887 | 746224 | 1423362 | 0.427 | 0.690 | 1.182 |
| C3 | 1.114 | 116.5 | 232.9 | 750 | 59566 | 24970 | 11368 | 0.445 | 0.733 | 1.276 |
| C4 | 1.340 | 141.8 | 283.5 | 250 | 7330 | 5095 | 651 | 0.494 | 0.803 | 1.825 |
| C6 | 0.670 | 141.8 | 212.6 | 1500 | 2364281 | 303380 | 210048 | 0.494 | 0.810 | 1.835 |
| C7 | 1.005 | 177.2 | 283.5 | 125 | 40566 | 16893 | 1103 | 0.499 | 0.821 | 1.997 |
| C8 | 0.670 | 212.6 | 283.5 | 125 | 563265 | 91497 | 2889 | 0.499 | 0.822 | 2.059 |
| D2 | 0.786 | 82.7 | 165.4 | 1250 | 1836827 | 444225 | 646653 | 0.500 | 0.827 | 2.062 |
| D3 | 1.170 | 124.0 | 248.1 | 500 | 30811 | 15625 | 4402 | 0.523 | 0.872 | 2.224 |
| D4 | 1.422 | 154.9 | 309.7 | 250 | 2741 | 2752 | 221 | 0.654 | 1.002 | 3.843 |
| D6 | 0.711 | 154.9 | 232.3 | 250 | 883973 | 163892 | 71139 | 0.654 | 1.004 | 3.848 |
| D7 | 1.067 | 193.6 | 309.7 | 125 | 13860 | 9126 | 278 | 0.667 | 1.024 | 4.491 |
| D8 | 0.711 | 232.3 | 309.7 | 125 | 172682 | 49429 | 432 | 0.668 | 1.027 | 4.904 |
| E2 | 0.813 | 88.5 | 177.0 | 1000 | 948759 | 291807 | 369307 | 0.669 | 1.032 | 4.908 |
| E3 | 1.219 | 132.7 | 265.4 | 500 | 14978 | 9946 | 2222 | 0.717 | 1.104 | 5.229 |
| E4 | 1.515 | 166.5 | 332.9 | 250 | 1198 | 1564 | 66 | 1.015 | 1.332 | 10.629 |
| E7 | 1.136 | 208.1 | 332.9 | 125 | 5565 | 5187 | 56 | 1.047 | 1.367 | 13.808 |
| E8 | 0.758 | 249.7 | 332.9 | 125 | 62419 | 28095 | 40 | 1.050 | 1.373 | 18.260 |


| Cont. Table D. 4 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{array}{r} \Delta \varepsilon_{\max } \\ \times 10^{-3} \\ \hline \end{array}$ | $\sigma_{m}$ | $\sigma_{\text {max }}$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{S W T}$ | $F_{G}$ |
| F2 | 0.856 | 75.0 | 150.0 | 1000 | 4699906 | 468112 | 2332955 | 1.050 | 1.376 | 18.261 |
| *F3 | 1.280 | 112.2 | 224.3 | 500 | 87957 | 16354 | 21186 | 1.058 | 1.420 | 18.295 |

* first failure observed (García 2008)

| Table D. 5 Config. \#13 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{aligned} & \Delta \varepsilon_{\max } \\ & \times 10^{-3} \end{aligned}$ | $\bar{\sigma}_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{S W T}$ | $F_{G}$ |
| A1 | 1.138 | 130.4 | 260.8 | 2000 | 18130 | 14260 | 10065 | 0.158 | 0.200 | 0.284 |
| A2 | 1.138 | 130.6 | 261.2 | 3500 | 17770 | 14149 | 10065 | 0.439 | 0.554 | 0.781 |
| A3 | 1.138 | 130.3 | 260.6 | 1000 | 18249 | 14296 | 10065 | 0.517 | 0.654 | 0.923 |
| A4 | 1.157 | 130.0 | 260.0 | 250 | 18704 | 13471 | 7936 | 0.536 | 0.680 | 0.968 |
| A5 | 0.289 | 97.5 | 130.0 | 2000 | 3437222388 | 78034918 | 2625958138 | 0.536 | 0.680 | 0.968 |
| A6 | 0.579 | 130.0 | 195.0 | 2000 | 6032847 | 802202 | 2559828 | 0.537 | 0.684 | 0.969 |
| A7 | 0.868 | 162.5 | 260.0 | 125 | 111655 | 44668 | 22756 | 0.538 | 0.688 | 0.977 |
| A8 | 0.579 | 195.0 | 260.0 | 125 | 1694740 | 241939 | 134355 | 0.539 | 0.689 | 0.978 |
| B2 | 0.729 | 81.3 | 162.5 | 2500 | 2175406 | 654629 | 2140013 | 0.540 | 0.694 | 0.980 |
| B3 | 1.088 | 121.9 | 243.8 | 750 | 37058 | 22812 | 18430 | 0.569 | 0.741 | 1.038 |
| B4 | 1.207 | 129.6 | 259.1 | 250 | 19410 | 11459 | 4538 | 0.587 | 0.772 | 1.116 |
| B6 | 0.604 | 129.6 | 194.3 | 1750 | 6260528 | 682340 | 1463594 | 0.588 | 0.776 | 1.118 |
| B7 | 0.905 | 161.9 | 259.1 | 125 | 116195 | 37993 | 11751 | 0.589 | 0.780 | 1.133 |
| B8 | 0.604 | 194.3 | 259.1 | 125 | 1769427 | 205789 | 59885 | 0.590 | 0.781 | 1.136 |
| C2 | 0.724 | 77.7 | 155.5 | 1250 | 3332626 | 808854 | 2124070 | 0.590 | 0.784 | 1.137 |
| C3 | 1.091 | 116.6 | 233.2 | 750 | 58854 | 27066 | 18268 | 0.608 | 0.823 | 1.196 |
| C4 | 1.251 | 132.9 | 265.8 | 250 | 14761 | 8874 | 2753 | 0.632 | 0.863 | 1.326 |
| C6 | 0.626 | 132.9 | 199.4 | 1500 | 4760964 | 528416 | 887971 | 0.633 | 0.867 | 1.328 |
| C7 | 0.938 | 166.1 | 265.8 | 125 | 86519 | 29423 | 6463 | 0.635 | 0.874 | 1.356 |
| C8 | 0.626 | 199.4 | 265.8 | 125 | 1285489 | 159367 | 28458 | 0.635 | 0.875 | 1.362 |
| D2 | 0.734 | 77.5 | 155.0 | 1250 | 3419408 | 773735 | 1947534 | 0.636 | 0.877 | 1.363 |
| D3 | 1.093 | 116.3 | 232.6 | 500 | 60478 | 27216 | 16477 | 0.647 | 0.903 | 1.406 |
| D4 | 1.301 | 138.0 | 276.0 | 250 | 9808 | 6439 | 1593 | 0.684 | 0.959 | 1.630 |
| D6 | 0.651 | 138.0 | 207.0 | 250 | 3163425 | 383418 | 513915 | 0.684 | 0.960 | 1.631 |
| D7 | 0.976 | 172.5 | 276.0 | 125 | 55626 | 21349 | 3331 | 0.687 | 0.968 | 1.685 |
| D8 | 0.651 | 207.0 | 276.0 | 125 | 795133 | 115636 | 12274 | 0.687 | 0.970 | 1.699 |
| E2 | 0.743 | 78.9 | 157.7 | 1000 | 2899172 | 682669 | 1599529 | 0.688 | 0.972 | 1.700 |
| E3 | 1.115 | 118.3 | 236.6 | 500 | 50611 | 23269 | 13059 | 0.702 | 1.002 | 1.755 |
| *E4 | 1.353 | 145.6 | 291.3 | 250 | 5443 | 4373 | 906 | 0.768 | 1.084 | 2.149 |


| Table D.6 Config. \#15 |  |  |  |  |  |  |  |  |  |  |
| :--- | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
|  | $\Delta \varepsilon_{\text {max }}$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{S W T}$ | $F_{G}$ |
|  | $\times 10^{-3}$ |  |  |  |  |  |  |  |  |  |
| A1 | 1.138 | 129.1 | 258.2 | 2000 | 20178 | 14867 | 10065 | 0.142 | 0.192 | 0.284 |
| A2 | 1.138 | 128.7 | 257.4 | 3500 | 20821 | 15051 | 10065 | 0.382 | 0.524 | 0.781 |
| A3 | 1.138 | 128.3 | 256.6 | 1000 | 21486 | 15237 | 10065 | 0.448 | 0.618 | 0.923 |
| A4 | 1.175 | 127.9 | 255.8 | 250 | 22269 | 13523 | 6542 | 0.464 | 0.645 | 0.977 |
| A5 | 0.294 | 95.9 | 127.9 | 2000 | 4045967212 | 78335143 | $\#$ | 0.464 | 0.645 | 0.977 |
| A6 | 0.588 | 127.9 | 191.8 | 2000 | 7182719 | 805288 | 2110048 | 0.465 | 0.648 | 0.979 |
| A7 | 0.881 | 159.9 | 255.8 | 125 | 134692 | 44839 | 18125 | 0.466 | 0.652 | 0.988 |
| A8 | 0.588 | 191.8 | 255.8 | 125 | 2075816 | 242870 | 101891 | 0.466 | 0.653 | 0.990 |
| B2 | 0.740 | 79.9 | 159.9 | 2500 | 2547500 | 657147 | 1827970 | 0.467 | 0.658 | 0.992 |
| B3 | 1.105 | 119.9 | 239.8 | 750 | 43990 | 22900 | 15298 | 0.492 | 0.705 | 1.062 |
| B4 | 1.279 | 134.3 | 268.6 | 250 | 13164 | 7741 | 2044 | 0.519 | 0.751 | 1.237 |
| B6 | 0.640 | 134.3 | 201.5 | 1750 | 4245841 | 460985 | 659127 | 0.520 | 0.757 | 1.241 |
| B7 | 0.959 | 167.9 | 268.6 | 125 | 76463 | 25668 | 4509 | 0.522 | 0.764 | 1.280 |
| B8 | 0.640 | 201.5 | 268.6 | 125 | 1124108 | 139030 | 18063 | 0.522 | 0.765 | 1.290 |
| C2 | 0.767 | 80.6 | 161.2 | 1250 | 2352682 | 546458 | 1131913 | 0.523 | 0.768 | 1.292 |
| C3 | 1.156 | 120.9 | 241.8 | 750 | 40350 | 18285 | 8649 | 0.549 | 0.827 | 1.416 |
| C4 | 1.369 | 142.5 | 285.0 | 250 | 6907 | 4556 | 760 | 0.601 | 0.905 | 1.885 |
|  |  |  |  |  |  |  |  |  |  |  |

C4 - first failure observed (García 2008)

| Table D. 7 Config. \#17 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{array}{r} \Delta \varepsilon_{\max } \\ \times 10^{-3} \\ \hline \end{array}$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| A1 | 1.138 | 123.7 | 247.4 | 2000 | 31692 | 17753 | 10065 | 0.090 | 0.161 | 0.284 |
| A2 | 1.138 | 124.0 | 248.0 | 3500 | 30893 | 17575 | 10065 | 0.252 | 0.445 | 0.781 |
| A3 | 1.138 | 124.4 | 248.7 | 1000 | 29987 | 17369 | 10065 | 0.300 | 0.528 | 0.923 |
| A4 | 1.138 | 124.7 | 249.3 | 250 | 29234 | 17196 | 10065 | 0.312 | 0.548 | 0.958 |
| A5 | 0.285 | 93.5 | 124.7 | 2000 | 5219991587 | 99608867 | \# | 0.312 | 0.548 | 0.958 |
| A6 | 0.569 | 124.7 | 187.0 | 2000 | 9429077 | 1023983 | 3246250 | 0.312 | 0.551 | 0.959 |
| A7 | 0.854 | 155.8 | 249.3 | 125 | 180366 | 57017 | 30063 | 0.313 | 0.554 | 0.965 |
| A8 | 0.569 | 187.0 | 249.3 | 125 | 2844443 | 308827 | 188072 | 0.313 | 0.555 | 0.966 |
| B2 | 0.717 | 77.9 | 155.8 | 2500 | 3260950 | 835611 | 2599950 | 0.314 | 0.559 | 0.967 |
| B3 | 1.070 | 116.9 | 233.7 | 750 | 57485 | 29119 | 23176 | 0.333 | 0.596 | 1.013 |
| B4 | 1.138 | 125.0 | 249.9 | 250 | 28500 | 17024 | 10065 | 0.345 | 0.617 | 1.049 |
| B6 | 0.569 | 125.0 | 187.4 | 1750 | 9192532 | 1013778 | 3246250 | 0.346 | 0.620 | 1.050 |
| B7 | 0.854 | 156.2 | 249.9 | 125 | 175520 | 56448 | 30063 | 0.347 | 0.623 | 1.056 |
| B8 | 0.569 | 187.4 | 249.9 | 125 | 2762191 | 305749 | 188072 | 0.347 | 0.623 | 1.057 |
| C2 | 0.683 | 75.0 | 149.9 | 1250 | 4710626 | 1201746 | 4031757 | 0.347 | 0.625 | 1.057 |
| C3 | 1.029 | 112.5 | 224.9 | 750 | 85533 | 40212 | 38710 | 0.360 | 0.651 | 1.085 |
| C4 | 1.138 | 125.3 | 250.5 | 250 | 27787 | 16855 | 10065 | 0.373 | 0.673 | 1.120 |
| C6 | 0.569 | 125.3 | 187.9 | 1500 | 8962320 | 1003699 | 3246250 | 0.373 | 0.675 | 1.121 |
| C7 | 0.854 | 156.6 | 250.5 | 125 | 170812 | 55887 | 30063 | 0.374 | 0.678 | 1.127 |
| C8 | 0.569 | 187.9 | 250.5 | 125 | 2682417 | 302709 | 188072 | 0.374 | 0.678 | 1.128 |
| D2 | 0.668 | 73.1 | 146.1 | 1250 | 6025185 | 1469669 | 5444514 | 0.374 | 0.680 | 1.128 |
| D3 | 0.994 | 109.6 | 219.2 | 500 | 111437 | 51695 | 54824 | 0.381 | 0.694 | 1.141 |
| D4 | 1.138 | 125.6 | 251.1 | 250 | 27092 | 16688 | 10065 | 0.394 | 0.715 | 1.177 |
| D6 | 0.569 | 125.6 | 188.3 | 250 | 8738262 | 993743 | 3246250 | 0.394 | 0.715 | 1.177 |
| D7 | 0.854 | 156.9 | 251.1 | 125 | 166237 | 55333 | 30063 | 0.395 | 0.719 | 1.183 |
| D8 | 0.569 | 188.3 | 251.1 | 125 | 2605045 | 299707 | 188072 | 0.395 | 0.719 | 1.184 |
| E2 | 0.650 | 71.7 | 143.5 | 1000 | 7163646 | 1769343 | 6770046 | 0.395 | 0.720 | 1.184 |
| E3 | 0.975 | 107.6 | 215.2 | 500 | 134307 | 60309 | 70564 | 0.401 | 0.732 | 1.194 |
| E4 | 1.196 | 126.5 | 252.9 | 250 | 25117 | 13168 | 4452 | 0.415 | 0.759 | 1.274 |
| E7 | 0.897 | 158.1 | 252.9 | 125 | 153271 | 43662 | 281367 | 0.416 | 0.763 | 1.275 |
| E8 | 0.598 | 189.7 | 252.9 | 125 | 2386593 | 236492 | 2600754 | 0.416 | 0.764 | 1.275 |


| Cont. Table D.7 |  |  |  |  |  |  |  |  |  |  |
| :---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
|  | $\Delta \varepsilon_{\max }$ <br> x $10^{-3}$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{S W T}$ | $F_{G}$ |
| F2 | 0.676 | 60.8 | 121.6 | 1000 | 33851781 | 3004340 | 37397911 | 0.416 | 0.764 | 1.275 |
| F3 | 1.011 | 90.9 | 181.9 | 500 | 725545 | 104958 | 508763 | 0.417 | 0.771 | 1.276 |
| F4 | 1.269 | 132.1 | 264.1 | 250 | 15818 | 8588 | 1661 | 0.440 | 0.813 | 1.491 |
| F7 | 0.952 | 165.1 | 264.1 | 125 | 93216 | 28476 | 2539410 | 0.442 | 0.819 | 1.491 |
| F8 | 0.635 | 198.1 | 264.1 | 125 | 1393762 | 154238 | 31138032 | 0.442 | 0.820 | 1.491 |
| G2 | 0.705 | 50.5 | 101.0 | 1000 | 186110857 | 5448922 | $\#$ | 0.442 | 0.820 | 1.491 |
| G3 | 1.060 | 76.0 | 152.0 | 500 | 4146906 | 181654 | 3679460 | 0.442 | 0.824 | 1.492 |
| G4 | 1.331 | 140.3 | 280.6 | 250 | 8204 | 5469 | 730 | 0.485 | 0.890 | 1.981 |
| G7 | 0.998 | 175.4 | 280.6 | 125 | 45839 | 18134 | 21839218 | 0.489 | 0.899 | 1.981 |
| G8 | 0.666 | 210.5 | 280.6 | 125 | 643764 | 98222 | $\#$ | 0.490 | 0.901 | 1.981 |
| H2 | 0.735 | 41.9 | 83.9 | 1000 | 1000710705 | 9961660 | $\#$ | 0.490 | 0.901 | 1.981 |
| H3 | 1.097 | 62.6 | 125.2 | 500 | 25780207 | 353534 | 27774095 | 0.490 | 0.903 | 1.981 |
| H4 | 1.386 | 149.1 | 298.1 | 250 | 4207 | 3591 | 353 | 0.574 | 1.003 | 2.993 |
| H7 | 1.040 | 186.3 | 298.1 | 125 | 22157 | 11906 | $\#$ | 0.583 | 1.018 | 2.993 |
| H8 | 0.693 | 223.6 | 298.1 | 125 | 289993 | 64485 | $\#$ | 0.583 | 1.021 | 2.993 |
| I2 | 0.756 | 34.1 | 68.3 | 1000 | 6239902288 | 20822573 | $\#$ | 0.583 | 1.021 | 2.993 |
| I3 | 1.134 | 51.2 | 102.4 | 500 | 163993528 | 709753 | $\#$ | 0.583 | 1.022 | 2.993 |
| *I4 | 1.439 | 156.5 | 313.0 | 250 | 2431 | 2506 | 178 | 0.730 | 1.164 | 5.005 |

* first failure observed (García 2008)

| Table D. 8 Config. \#18 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{array}{r} \Delta \varepsilon_{\max } \\ \times 10^{-3} \\ \hline \end{array}$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| A1 | 1.138 | 126.3 | 252.6 | 2000 | 25435 | 16279 | 10065 | 0.112 | 0.176 | 0.284 |
| A2 | 1.138 | 126.3 | 252.6 | 3500 | 25435 | 16279 | 10065 | 0.309 | 0.483 | 0.781 |
| A3 | 1.138 | 126.3 | 252.6 | 1000 | 25435 | 16279 | 10065 | 0.365 | 0.570 | 0.923 |
| A4 | 1.138 | 126.3 | 252.6 | 250 | 25435 | 16279 | 10065 | 0.379 | 0.592 | 0.958 |
| A5 | 0.285 | 94.7 | 126.3 | 2000 | 4581893072 | 94297871 | \# | 0.379 | 0.592 | 0.958 |
| A6 | 0.569 | 126.3 | 189.5 | 2000 | 8203894 | 969386 | 3246250 | 0.379 | 0.595 | 0.959 |
| A7 | 0.854 | 157.9 | 252.6 | 125 | 155356 | 53977 | 30063 | 0.381 | 0.599 | 0.965 |
| A8 | 0.569 | 189.5 | 252.6 | 125 | 2421630 | 292360 | 188072 | 0.381 | 0.599 | 0.966 |
| B2 | 0.717 | 78.9 | 157.9 | 2500 | 2873793 | 791057 | 2599950 | 0.382 | 0.604 | 0.967 |
| B3 | 1.070 | 118.4 | 236.8 | 750 | 50131 | 27566 | 23176 | 0.403 | 0.643 | 1.013 |
| B4 | 1.138 | 126.2 | 252.4 | 250 | 25650 | 16333 | 10065 | 0.417 | 0.664 | 1.049 |
| B6 | 0.569 | 126.2 | 189.3 | 1750 | 8273079 | 972590 | 3246250 | 0.418 | 0.667 | 1.050 |
| B7 | 0.854 | 157.8 | 252.4 | 125 | 156762 | 54155 | 30063 | 0.419 | 0.670 | 1.056 |
| B8 | 0.569 | 189.3 | 252.4 | 125 | 2445287 | 293327 | 188072 | 0.419 | 0.671 | 1.057 |
| C2 | 0.683 | 75.7 | 151.4 | 1250 | 4283296 | 1152922 | 4031757 | 0.419 | 0.673 | 1.057 |
| C3 | 1.029 | 113.6 | 227.2 | 750 | 77191 | 38579 | 38710 | 0.433 | 0.700 | 1.085 |
| C4 | 1.138 | 126.5 | 253.0 | 250 | 25012 | 16172 | 10065 | 0.447 | 0.722 | 1.120 |
| C6 | 0.569 | 126.5 | 189.8 | 1500 | 8067372 | 963016 | 3246250 | 0.448 | 0.725 | 1.121 |
| C7 | 0.854 | 158.1 | 253.0 | 125 | 152583 | 53622 | 30063 | 0.449 | 0.728 | 1.127 |
| C8 | 0.569 | 189.8 | 253.0 | 125 | 2375031 | 290439 | 188072 | 0.449 | 0.729 | 1.128 |
| D2 | 0.668 | 73.8 | 147.6 | 1250 | 5481780 | 1410099 | 5444514 | 0.449 | 0.730 | 1.128 |
| D3 | 0.994 | 110.7 | 221.4 | 500 | 100656 | 49600 | 54824 | 0.456 | 0.744 | 1.141 |
| D4 | 1.193 | 126.0 | 252.0 | 250 | 26085 | 13506 | 4652 | 0.470 | 0.771 | 1.218 |
| D6 | 0.597 | 126.0 | 189.0 | 250 | 8413327 | 804250 | 1500592 | 0.470 | 0.771 | 1.218 |
| D7 | 0.895 | 157.5 | 252.0 | 125 | 159615 | 44782 | 12105 | 0.471 | 0.775 | 1.233 |
| D8 | 0.597 | 189.0 | 252.0 | 125 | 2493325 | 242556 | 62118 | 0.471 | 0.776 | 1.236 |
| E2 | 0.682 | 72.0 | 144.0 | 1000 | 6924514 | 1431953 | 3669625 | 0.471 | 0.777 | 1.236 |
| E3 | 1.023 | 108.0 | 216.0 | 500 | 129496 | 48809 | 34690 | 0.477 | 0.791 | 1.257 |
| E4 | 1.276 | 131.9 | 263.7 | 250 | 16077 | 8447 | 1508 | 0.499 | 0.834 | 1.494 |
| E7 | 0.957 | 164.8 | 263.7 | 125 | 94865 | 28007 | 125599 | 0.501 | 0.840 | 1.495 |
| E8 | 0.638 | 197.8 | 263.7 | 125 | 1420509 | 151698 | 1022624 | 0.501 | 0.841 | 1.495 |


| Cont. Table D.8 |  |  |  |  |  |  |  |  |  |  |
| :--- | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
|  | $\Delta \varepsilon_{\max }$ <br> $\mathrm{x} 10^{-3}$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{S W T}$ | $F_{G}$ |
| F2 | 0.721 | 61.0 | 122.0 | 1000 | 32744787 | 2260010 | 20777085 | 0.501 | 0.842 | 1.495 |
| F3 | 1.078 | 91.3 | 182.5 | 500 | 700408 | 78954 | 262676 | 0.502 | 0.851 | 1.498 |
| F4 | 1.352 | 143.0 | 286.0 | 250 | 6656 | 4733 | 555 | 0.556 | 0.926 | 2.142 |
| F7 | 1.014 | 178.8 | 286.0 | 125 | 36531 | 15692 | 1224105 | 0.561 | 0.938 | 2.142 |
| F8 | 0.676 | 214.5 | 286.0 | 125 | 502237 | 84994 | 13792369 | 0.561 | 0.940 | 2.142 |
| G2 | 0.751 | 50.7 | 101.4 | 1000 | 180139165 | 4122780 | $\#$ | 0.561 | 0.940 | 2.142 |
| G3 | 1.130 | 76.2 | 152.5 | 500 | 4007352 | 137444 | 1974338 | 0.561 | 0.945 | 2.142 |
| G4 | 1.418 | 153.1 | 306.2 | 250 | 3115 | 2920 | 232 | 0.676 | 1.068 | 3.679 |

E4 - visible cracking (García 2008)

| Table D. 9 Config. \#19 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{array}{r} \Delta \varepsilon_{\max } \\ \times 10^{-3} \\ \hline \end{array}$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| A1 | 1.110 | 113.5 | 226.9 | 2000 | 78109 | 28240 | 16622 | 0.037 | 0.101 | 0.172 |
| A2 | 1.138 | 129.7 | 259.3 | 3500 | 19251 | 14597 | 10066 | 0.296 | 0.444 | 0.669 |
| A3 | 1.138 | 129.5 | 259.0 | 1000 | 19490 | 14667 | 10066 | 0.370 | 0.541 | 0.811 |
| A4 | 1.138 | 129.6 | 259.2 | 250 | 19330 | 14620 | 10066 | 0.388 | 0.566 | 0.846 |
| A5 | 0.285 | 97.2 | 129.6 | 2000 | 3544579326 | 84689261 | \# | 0.388 | 0.566 | 0.846 |
| A6 | 0.569 | 129.6 | 194.4 | 2000 | 6234783 | 870609 | 3246626 | 0.389 | 0.569 | 0.847 |
| A7 | 0.854 | 162.0 | 259.2 | 125 | 115681 | 48477 | 30067 | 0.390 | 0.573 | 0.853 |
| A8 | 0.569 | 194.4 | 259.2 | 125 | 1760962 | 262570 | 188103 | 0.390 | 0.573 | 0.854 |
| B2 | 0.717 | 81.0 | 162.0 | 2500 | 2241152 | 710451 | 2600197 | 0.392 | 0.578 | 0.855 |
| B3 | 1.070 | 121.5 | 243.0 | 750 | 38277 | 24758 | 23178 | 0.420 | 0.622 | 0.901 |
| B4 | 1.138 | 129.7 | 259.3 | 250 | 19251 | 14597 | 10066 | 0.438 | 0.646 | 0.937 |
| B6 | 0.569 | 129.7 | 194.5 | 1750 | 6209151 | 869211 | 3246626 | 0.439 | 0.649 | 0.938 |
| B7 | 0.854 | 162.1 | 259.3 | 125 | 115169 | 48399 | 30067 | 0.440 | 0.653 | 0.944 |
| B8 | 0.569 | 194.5 | 259.3 | 125 | 1752539 | 262148 | 188103 | 0.440 | 0.653 | 0.945 |
| C2 | 0.683 | 77.8 | 155.6 | 1250 | 3308035 | 1030374 | 4032136 | 0.441 | 0.655 | 0.945 |
| C3 | 1.029 | 116.7 | 233.4 | 750 | 58384 | 34478 | 38715 | 0.459 | 0.686 | 0.973 |
| C4 | 1.165 | 128.7 | 257.4 | 250 | 20822 | 13650 | 6809 | 0.476 | 0.712 | 1.025 |
| C6 | 0.583 | 128.7 | 193.1 | 1500 | 6715998 | 812833 | 2196121 | 0.477 | 0.715 | 1.026 |
| C7 | 0.874 | 160.9 | 257.4 | 125 | 125309 | 45260 | 19001 | 0.478 | 0.719 | 1.035 |
| C8 | 0.583 | 193.1 | 257.4 | 125 | 1920002 | 245145 | 107916 | 0.478 | 0.720 | 1.037 |
| D2 | 0.683 | 75.1 | 150.1 | 1250 | 4650636 | 1190193 | 3979737 | 0.479 | 0.721 | 1.038 |
| D3 | 1.017 | 112.6 | 225.2 | 500 | 84309 | 41865 | 38102 | 0.487 | 0.738 | 1.056 |
| D4 | 1.234 | 135.1 | 270.1 | 250 | 12418 | 8787 | 2669 | 0.516 | 0.779 | 1.190 |
| D6 | 0.617 | 135.1 | 202.6 | 250 | 4005333 | 523274 | 860746 | 0.516 | 0.779 | 1.191 |
| D7 | 0.926 | 168.8 | 270.1 | 125 | 71797 | 29137 | 6225 | 0.518 | 0.786 | 1.219 |
| D8 | 0.617 | 202.6 | 270.1 | 125 | 1049751 | 157816 | 27148 | 0.519 | 0.787 | 1.226 |
| E2 | 0.650 | 77.2 | 154.3 | 1000 | 3573061 | 1305633 | 2378267 | 0.519 | 0.788 | 1.226 |
| E3 | 0.975 | 115.8 | 231.5 | 500 | 63460 | 44503 | 20870 | 0.530 | 0.804 | 1.261 |
| E4 | 1.302 | 141.4 | 282.7 | 250 | 7561 | 5812 | 1067 | 0.578 | 0.865 | 1.595 |
| E7 | 0.977 | 176.7 | 282.7 | 125 | 41954 | 19269 | 2036 | 0.582 | 0.875 | 1.683 |
| E8 | 0.651 | 212.0 | 282.7 | 125 | 584373 | 104371 | 6492 | 0.582 | 0.876 | 1.711 |


| Cont. Table D. 9 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{gathered} \Delta \varepsilon_{\max } \\ \times 10^{-3} \end{gathered}$ | $\sigma_{m}$ | $\sigma_{\text {max }}$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| F2 | 0.643 | 65.1 | 130.2 | 1000 | 17866580 | 2778027 | 14362266 | 0.582 | 0.877 | 1.711 |
| F3 | 0.962 | 97.7 | 195.3 | 500 | 356943 | 95840 | 166848 | 0.584 | 0.884 | 1.715 |
| F4 | 1.370 | 147.7 | 295.4 | 250 | 4655 | 3914 | 435 | 0.661 | 0.975 | 2.537 |
| F7 | 1.028 | 184.6 | 295.4 | 125 | 24748 | 12978 | 663 | 0.668 | 0.989 | 2.806 |
| F8 | 0.685 | 221.6 | 295.4 | 125 | 327540 | 70297 | 1452 | 0.669 | 0.992 | 2.929 |
| G2 | 0.632 | 54.3 | 108.5 | 1000 | 96693722 | 6366641 | 91135018 | 0.669 | 0.992 | 2.929 |
| G3 | 0.951 | 81.4 | 162.8 | 500 | 2139699 | 214724 | 1328907 | 0.669 | 0.995 | 2.930 |
| G4 | 1.439 | 154.1 | 308.1 | 250 | 2906 | 2676 | 178 | 0.792 | 1.129 | 4.941 |
| G7 | 1.079 | 192.6 | 308.1 | 125 | 14778 | 8874 | 209 | 0.804 | 1.149 | 5.793 |
| G8 | 0.720 | 231.1 | 308.1 | 125 | 185402 | 48067 | 289 | 0.805 | 1.153 | 6.412 |
| H2 | 0.628 | 44.8 | 89.5 | 1000 | 556780502 | 14583254 | \# | 0.805 | 1.153 | 6.412 |
| H3 | 0.938 | 67.1 | 134.3 | 500 | 13382532 | 507213 | 10429728 | 0.805 | 1.154 | 6.412 |
| H4 | 1.467 | 158.1 | 316.1 | 250 | 2173 | 2220 | 123 | 0.969 | 1.315 | 9.326 |
| H7 | 1.100 | 197.6 | 316.1 | 125 | 10740 | 7360 | 129 | 0.986 | 1.339 | 10.715 |
| H8 | 0.734 | 237.1 | 316.1 | 125 | 130113 | 39865 | 141 | 0.987 | 1.344 | 11.978 |
| 12 | 0.621 | 36.6 | 73.3 | 1000 | 3351541230 | 35352069 | \# | 0.987 | 1.344 | 11.978 |
| 13 | 0.931 | 54.9 | 109.9 | 500 | 86346366 | 1205309 | 80604937 | 0.987 | 1.344 | 11.978 |
| I4 | 1.496 | 162.1 | 324.1 | 250 | 1633 | 1844 | 84 | 1.206 | 1.538 | 16.223 |
| 17 | 1.122 | 202.6 | 324.1 | 125 | 7837 | 6113 | 78 | 1.229 | 1.567 | 18.518 |
| 18 | 0.748 | 243.1 | 324.1 | 125 | 91579 | 33108 | 66 | 1.231 | 1.573 | 21.210 |
| J2 | 0.813 | 29.8 | 59.5 | 1000 | \# | 27305201 | \# | 1.231 | 1.573 | 21.210 |
| J3 | 1.217 | 44.6 | 89.3 | 500 | 571133133 | 938931 | \# | 1.231 | 1.574 | 21.210 |
| J4 | 1.525 | 166.1 | 332.1 | 250 | 1232 | 1537 | 58 | 1.521 | 1.806 | 27.361 |
| J7 | 1.144 | 207.6 | 332.1 | 125 | 5740 | 5098 | 47 | 1.552 | 1.841 | 31.158 |
| J8 | 0.763 | 249.1 | 332.1 | 125 | 64625 | 27610 | 30 | 1.554 | 1.847 | 37.043 |
| K2 | 0.821 | 24.0 | 48.1 | 1000 | \# | \# | \# | 1.554 | 1.847 | 37.043 |
| K3 | 1.234 | 36.1 | 72.1 | 500 | \# | 2155261 | \# | 1.554 | 1.848 | 37.043 |
| K4 | 1.553 | 170.1 | 340.1 | 250 | 934 | 1291 | 40 | 1.937 | 2.124 | 45.962 |

K - first failure observed (García 2008)

| Table D.10 Config. \#20 |  |  |  |  |  |  |  |  |  |  |
| :--- | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
|  | $\Delta \varepsilon_{\text {max }}$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{S W T}$ | $F_{G}$ |
|  | $\times 10^{-3}$ |  |  |  |  |  |  |  |  |  |
| A1 | 1.138 | 128.9 | 257.8 | 2000 | 20439 | 14942 | 1814 | 0.140 | 0.191 | 0.284 |
| A2 | 1.138 | 129.9 | 259.8 | 3500 | 18874 | 14485 | 1631 | 0.405 | 0.536 | 0.781 |
| A3 | 1.138 | 128.9 | 257.8 | 1000 | 18874 | 14942 | 1631 | 0.474 | 0.632 | 0.923 |
| A4 | 1.174 | 128.5 | 257.0 | 250 | 18874 | 13305 | 1631 | 0.491 | 0.659 | 0.977 |
| A5 | 0.294 | 96.4 | 128.5 | 2000 | 3466429783 | 77071741 | 674296866 | 0.491 | 0.659 | 0.977 |
| A6 | 0.587 | 128.5 | 192.8 | 2000 | 6087747 | 792301 | 525979 | 0.492 | 0.663 | 0.978 |
| A7 | 0.881 | 160.6 | 257.0 | 125 | 112748 | 44116 | 3427 | 0.493 | 0.667 | 0.988 |
| A8 | 0.587 | 192.8 | 257.0 | 125 | 1712712 | 238953 | 12728 | 0.493 | 0.667 | 0.990 |
| B2 | 0.740 | 80.3 | 160.6 | 2500 | 2193298 | 646549 | 600911 | 0.495 | 0.673 | 0.992 |
| B3 | 1.104 | 120.5 | 240.9 | 750 | 37390 | 22531 | 4030 | 0.520 | 0.720 | 1.062 |
| B4 | 1.219 | 129.3 | 258.6 | 250 | 18874 | 11085 | 1631 | 0.538 | 0.753 | 1.152 |
| B6 | 0.610 | 129.3 | 194.0 | 1750 | 6087747 | 660076 | 525979 | 0.539 | 0.756 | 1.154 |
| B7 | 0.914 | 161.6 | 258.6 | 125 | 112748 | 36754 | 3427 | 0.540 | 0.761 | 1.171 |
| B8 | 0.610 | 194.0 | 258.6 | 125 | 1712712 | 199075 | 12728 | 0.540 | 0.762 | 1.175 |
| C2 | 0.731 | 77.6 | 155.2 | 1250 | 3249854 | 782463 | 949521 | 0.541 | 0.764 | 1.176 |
| C3 | 1.102 | 116.4 | 232.7 | 750 | 57273 | 26183 | 7006 | 0.559 | 0.805 | 1.242 |
| C4 | 1.229 | 130.6 | 261.2 | 250 | 17795 | 10274 | 1507 | 0.579 | 0.840 | 1.343 |
| C6 | 0.615 | 130.6 | 195.9 | 1500 | 5739707 | 611804 | 485989 | 0.579 | 0.844 | 1.345 |
| C7 | 0.922 | 163.3 | 261.2 | 125 | 105827 | 34066 | 3112 | 0.581 | 0.849 | 1.366 |
| C8 | 0.615 | 195.9 | 261.2 | 125 | 1599181 | 184516 | 11244 | 0.581 | 0.850 | 1.370 |
| D2 | 0.721 | 76.2 | 152.4 | 1250 | 4040711 | 895837 | 1221585 | 0.581 | 0.852 | 1.371 |
| *D3 | 1.073 | 114.3 | 228.6 | 500 | 72439 | 31511 | 9467 | 0.591 | 0.874 | 1.405 |

* first failure observed (García 2008)

| Table D. 11 Config. \#21 |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{gathered} \Delta \varepsilon_{\max } \\ \times 10^{-3} \end{gathered}$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{S W T}$ | $N_{G}$ | $F_{M}$ | $F_{\text {SWT }}$ | $F_{G}$ |
| A1 | 1.138 | 131.4 | 262.7 | 2000 | 16746 | 13826 | 10065 | 0.171 | 0.207 | 0.284 |
| A2 | 1.138 | 131.4 | 262.7 | 3500 | 16746 | 13826 | 10065 | 0.469 | 0.568 | 0.781 |
| A3 | 1.138 | 131.4 | 262.7 | 1000 | 16746 | 13826 | 10065 | 0.555 | 0.672 | 0.923 |
| A4 | 1.138 | 131.5 | 262.9 | 250 | 16610 | 13782 | 8714 | 0.576 | 0.698 | 0.964 |
| A5 | 0.285 | 98.6 | 131.5 | 2000 | 3076540827 | 79832560 | \# | 0.576 | 0.698 | 0.964 |
| A6 | 0.569 | 131.5 | 197.2 | 2000 | 5357362 | 820682 | 2810456 | 0.577 | 0.701 | 0.965 |
| A7 | 0.854 | 164.3 | 262.9 | 125 | 98255 | 45697 | 25393 | 0.578 | 0.705 | 0.972 |
| A8 | 0.569 | 197.2 | 262.9 | 125 | 1475615 | 247512 | 153424 | 0.578 | 0.706 | 0.973 |
| B2 | 0.717 | 82.2 | 164.3 | 2500 | 1954084 | 669709 | 2309980 | 0.580 | 0.711 | 0.974 |
| B3 | 1.070 | 123.2 | 246.5 | 750 | 32977 | 23338 | 20167 | 0.613 | 0.757 | 1.027 |
| B4 | 1.152 | 134.7 | 269.3 | 250 | 12822 | 11848 | 6144 | 0.641 | 0.787 | 1.086 |
| B6 | 0.576 | 134.7 | 202.0 | 1750 | 4135756 | 705545 | 1981741 | 0.641 | 0.791 | 1.087 |
| B7 | 0.864 | 168.3 | 269.3 | 125 | 74325 | 39286 | 16831 | 0.644 | 0.795 | 1.097 |
| B8 | 0.576 | 202.0 | 269.3 | 125 | 1089991 | 212788 | 93084 | 0.644 | 0.796 | 1.099 |
| C2 | 0.691 | 80.8 | 161.6 | 1250 | 2298050 | 836363 | 2706747 | 0.645 | 0.798 | 1.100 |
| C3 | 1.041 | 121.2 | 242.4 | 750 | 39334 | 27986 | 24297 | 0.672 | 0.836 | 1.144 |
| C4 | 1.151 | 134.6 | 269.2 | 250 | 12874 | 11910 | 6215 | 0.700 | 0.866 | 1.202 |
| C6 | 0.576 | 134.6 | 201.9 | 1500 | 4152374 | 709200 | 2004458 | 0.700 | 0.869 | 1.203 |
| C7 | 0.863 | 168.3 | 269.2 | 125 | 74647 | 39489 | 17060 | 0.702 | 0.874 | 1.213 |
| C8 | 0.576 | 201.9 | 269.2 | 125 | 1095132 | 213890 | 94627 | 0.703 | 0.875 | 1.215 |
| D2 | 0.675 | 78.5 | 157.0 | 1250 | 3027048 | 1038448 | 3700281 | 0.703 | 0.877 | 1.216 |
| *D3 | 1.005 | 117.8 | 235.6 | 500 | 53003 | 36527 | 35003 | 0.717 | 0.896 | 1.236 |

* first failure observed (García 2008)

| Table D. 12 Config. \#24 |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{gathered} \Delta \varepsilon_{\max } \\ \times 10^{-3} \end{gathered}$ | $\sigma_{m}$ | $\sigma_{\max }$ | $n$ | $N_{M}$ | $N_{\text {SWT }}$ | $F_{M}$ | $F_{S W T}$ |
| A1 | 0.874 | 118.0 | 235.9 | 2000 | 52189 | 64893 | 0.055 | 0.044 |
| A2 | 1.138 | 129.2 | 258.3 | 3500 | 20061 | 14834 | 0.304 | 0.381 |
| A3 | 1.138 | 128.9 | 257.8 | 1000 | 20480 | 14954 | 0.374 | 0.477 |
| A4 | 1.138 | 129.0 | 257.9 | 250 | 20396 | 14930 | 0.391 | 0.501 |
| A5 | 0.285 | 96.7 | 129.0 | 2000 | 3726839595 | 86482235 | 0.391 | 0.501 |
| A6 | 0.569 | 129.0 | 193.4 | 2000 | 6578473 | 889041 | 0.392 | 0.504 |
| A7 | 0.854 | 161.2 | 257.9 | 125 | 122553 | 49503 | 0.393 | 0.507 |
| A8 | 0.569 | 193.4 | 257.9 | 125 | 1874378 | 268129 | 0.393 | 0.508 |
| B2 | 0.717 | 80.6 | 161.2 | 2500 | 2352644 | 725493 | 0.395 | 0.513 |
| B3 | 1.070 | 120.9 | 241.8 | 750 | 40350 | 25282 | 0.421 | 0.555 |
| B4 | 1.212 | 129.6 | 259.2 | 250 | 19330 | 11245 | 0.440 | 0.587 |
| B6 | 0.606 | 129.6 | 194.4 | 1750 | 6234783 | 669610 | 0.440 | 0.591 |
| B7 | 0.909 | 162.0 | 259.2 | 125 | 115681 | 37285 | 0.442 | 0.596 |
| B8 | 0.606 | 194.4 | 259.2 | 125 | 1760962 | 201950 | 0.442 | 0.597 |
| C2 | 0.727 | 77.8 | 155.5 | 1250 | 3320306 | 793764 | 0.442 | 0.599 |
| C3 | 1.096 | 116.6 | 233.3 | 750 | 58618 | 26561 | 0.461 | 0.639 |
| C4 | 1.346 | 144.3 | 288.6 | 250 | 6025 | 4643 | 0.520 | 0.716 |
| C6 | 0.673 | 144.3 | 216.5 | 1500 | 1943220 | 276464 | 0.521 | 0.724 |
| C7 | 1.010 | 180.4 | 288.6 | 125 | 32778 | 15394 | 0.526 | 0.735 |
| C8 | 0.673 | 216.5 | 288.6 | 125 | 445974 | 83380 | 0.527 | 0.738 |
| D2 | 0.790 | 84.2 | 168.3 | 1250 | 1544946 | 404814 | 0.528 | 0.742 |
| D3 | 1.176 | 126.3 | 252.5 | 500 | 25516 | 14239 | 0.556 | 0.792 |
| D4 | 1.439 | 153.8 | 307.5 | 250 | 2970 | 2698 | 0.676 | 0.924 |
| D6 | 0.720 | 153.8 | 230.6 | 250 | 958027 | 160677 | 0.677 | 0.927 |
| D7 | 1.079 | 192.2 | 307.5 | 125 | 15139 | 8947 | 0.688 | 0.947 |
| D8 | 0.720 | 230.6 | 307.5 | 125 | 190417 | 48459 | 0.689 | 0.950 |
| E2 | 0.822 | 87.9 | 175.7 | 1000 | 1017396 | 286083 | 0.691 | 0.955 |
| E3 | 1.233 | 131.8 | 263.6 | 500 | 16170 | 9751 | 0.735 | 1.029 |
| E4 | 1.507 | 163.4 | 326.7 | 250 | 1490 | 1730 | 0.975 | 1.235 |
| E7 | 1.130 | 204.2 | 326.7 | 125 | 7080 | 5735 | 1.000 | 1.266 |
| E8 | 0.754 | 245.0 | 326.7 | 125 | 81748 | 31062 | 1.002 | 1.272 |


| Cont. Table D. 12 |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{gathered} \Delta \varepsilon_{\max } \\ \times 10^{-3} \\ \hline \end{gathered}$ | $\sigma_{m}$ | $\sigma_{\text {max }}$ | $n$ | $N_{M}$ | $N_{S W T}$ | $F_{M}$ | $F_{\text {SWT }}$ |
| F2 | 0.851 | 74.1 | 148.3 | 1000 | 5247994 | 502175 | 1.002 | 1.275 |
| F3 | 1.273 | 111.2 | 222.4 | 500 | 96037 | 17325 | 1.010 | 1.316 |
| F4 | 1.570 | 171.8 | 343.5 | 250 | 831 | 1183 | 1.440 | 1.618 |
| F7 | 1.178 | 214.7 | 343.5 | 125 | 3704 | 3923 | 1.488 | 1.663 |
| F8 | 0.785 | 257.6 | 343.5 | 125 | 39474 | 21251 | 1.492 | 1.672 |
| G2 | 0.872 | 61.8 | 123.6 | 1000 | 29169863 | 970329 | 1.492 | 1.673 |
| G3 | 1.312 | 92.7 | 185.3 | 500 | 602436 | 32726 | 1.494 | 1.695 |
| *G4 | 1.630 | 179.5 | 358.9 | 250 | 494 | 843 | 2.217 | 2.119 |

- first failure observed (García 2008)


## E. Fatigue Damage Results for the Proposed Parameter Estimation Method

Table E. 1 Fatigue damage calculated at the middle of the loading sublevel in which failure was observed by García (2008).

| Config. \# | $F_{S W T}$ | Load level |
| :---: | :---: | :---: |
| 3 | 0.964 | D 4 |
| 4 | 1.000 | F 3 |
| 5 | 1.035 | H |
| 6 | 1.075 | F 3 |
| 13 | 1.041 | E 4 |
| 15 | 0.948 | C |
| 17 | 1.000 | I 4 |
| 18 | 1.000 | $\sim \mathrm{G} 8$ |
| 19 | 1.089 | K |
| 20 | 0.996 | D 3 |
| 21 | 1.000 | D 3 |
| 24 | 0.948 | G 4 |


[^0]:    *García (2008)

[^1]:    K - first failure observed (García 2008)

